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A STUDY OF CONTROL SYSTEM DESIGN
AND HARDWARE SELECTION BY COMPUTER

A THESIS

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AND HARDWARE SELECTION BY COMPUTER

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SUMMARY

The objective of the reported study was to determine the feasibility of writing a digital computer program which will, given certain required parameters, select control system components, compute system cost, and analyze system performance with the selected hardware. This objective was achieved by writing a computer program which met the above requirements and comparing its cost (development cost plus cost per run) with the cost of a conventional design of the same control system. The term "conventional" refers to a method whereby the engineer achieves the system design using tabulated data and past experience. The control system selected for this study was a tank liquid level control system.

It was found that even if program development costs were neglected, some computer designs would cost more than equivalent conventional designs, depending on the amount of required run time (the time needed to determine the response of the control system). This result is brought about because the cost per unit run time on the computer is greater than ten times the cost per unit run time for the conventional design method.

If an experienced programmer had been assigned to develop the program, it is estimated that the development cost would be approximately \$4,580. This cost is made up of two components, a cost corresponding to a programming time of 500 hours and a cost corresponding to a computer time of 250 minutes.

Because the computer design method cannot compete with the conventional design method on a cost per design basis and because of the high program development cost required, it is felt that computer design of liquid level control systems using a digital computer program of the type developed for this study is not economically feasible.

Even if the results of this study had indicated that a design program of this nature would be slightly profitable, the wisdom of putting such a program into production would be questionable, because of the conservatism of the customers who buy this type of control system.

Three possible means of reducing the required computer time are discussed. The first deals with the selection of a faster numerical integration method, the second is concerned with the use of analog computational units for determining the system response curve, and the third suggests the possibility of using approximate flow equations.

NOMENCLATURE TABLE

A_1	Internal cross sectional area of pipe at D_1
A_3	Internal cross sectional area of pipe at D_3
A_8	Tank cross sectional area
CR	Control range
CV	Valve size coefficient
CVI	Inlet valve size coefficient
CVO	Outlet valve size coefficient
Cost _{c.d.}	Cost of computer design
Cost _{m.d.}	Cost of manual design
d	Nominal pipe diameter
D	Inside diameter of pipe
D_1	See Figure 1
D_3	See Figure 1
D_8	See Figure 1
E	Measure of roughness of inside pipe surface
F_e	External force
F	Friction factor
F_f	Friction force in pipe
F2	Friction factor—L2 section of pipe
F3	Friction factor — L3 section of pipe
F4	Friction factor—L4 section of pipe
g	Acceleration due to gravity

g_c	Gravitational constant
G	Specific weight
h_f	Head loss due to friction
H_I	Initial value of head
H	Head of liquid in tank
H1	Required head
H2	See Figure 1
H3	See Figure 1
K_1	Constant \$0.04/sec.
K_2	Constant whose value is a function of the system type
K_3	Constant—Cost per second for keeping an engineer in the field
K_4	Constant whose value depends on the system type
KI	Ratio of flow through inlet control valve to maximum flow through the same control valve
K20	Ratio of flow through outlet control valve to maximum flow through the same control valve
L	Pipe length
L1	Tank height
L2	Equivalent pipe lengths (See Figure 1)
L3	
L4	
L5	
L5T	
L2T	True pipe lengths
L3T	
L4T	
L5T	
L5T	

M	Mass of fluid in pipe
N	Number of moles of gas
P	Control pressure
P0	See Figure 1
P00	Initial tank charging pressure
P1	} See Figure 1
P2	
P3	
P4	
P5	
P6	
P7	
P13	Constant pressure above liquid in tank
PB	Proportional band
P_{mi}	See Figure 10
P_{ma}	
Q	Flow rate
Q_I	Flow rate into tank
Q_m	Maximum flow rate
Q_o	Flow rate from tank
QOT	Quick opening time
r	Radius of pipe bend
R	Valve rangeability
R_e	Reynolds number
R_o	Universal gas content
RR	Repeat rate

S.G.	Specific gravity
T	Time
T_I	Initial temperature
T1	Time interval
T7	Temperature at D_7 section
TRR	Required rise time
U	Fluid viscosity
V1	See Figure 9
V	Velocity of fluid in pipe
V_m	Motor voltage
V_o	Tank volume
V_{oi}	Initial tank volume
V_r	Liquid velocity in control volume
w	Fluid specific weight
w_w	Specific weight of water
x	Position of valve stem
X	Maximum position of valve stem
ΔH	See Figure 8
ΔP	Pressure drop across the control valve
ρ_r	Density of liquid in region R
ρ_s	Density of liquid at surface S

CHAPTER I

INTRODUCTION

The object of the study described in this thesis is to determine the feasibility of writing a digital computer program which will, given certain required parameters such as maximum allowable rise time, maximum overshoot, maximum settling time, and maximum allowable steady state error, select system components for a control system with a known block diagram, compute system cost, and analyze the system performance with the selected hardware.

Since the inception of modern high speed digital computers, attempts have been made to apply them to various engineering design problems. Computer programs have been written for finding thermal stresses in piping, for calculating pressure drops in piping systems, for calculating stresses in structures, and for calculating power plant heat balances.

In every case listed above, the digital computer was ideally suited to the problem because of the large number of repetitive calculations required for solution.

Recently, attempts have been made to enlarge the field of computer applications. The IBM "COMMEND" program is one such attempt (1). This program attacks design problems previously done by graphical or experimental methods, and short duration calculations performed occasionally by an engineer. The following mechanical design areas are included in

COMMENT I: kinematic synthesis, kinematic analysis, component design, physical properties calculations, dynamic analysis, detail drawings, numerical control machining instructions, and engineering information retrieval.

The computer-aided design project at the Massachusetts Institute of Technology is an example of work in another area of computer application (2). This effort centers around the use of a light pen and a cathode-ray oscilloscope. With this concept, the engineer draws his design on the oscilloscope and the computer analyzes the design almost concurrently.

During and after World War II, tremendous advances in automatic control design theory took place. Control design changed from an art to a science with the increased application of the mathematical approach to control problems.

The use of both analog and digital computers for determining the time response of automatic control systems is common today (3). Gain settings which will give the required system response are the end product of most present computer solutions. Most of these programs require frequent communication between the computer and the engineer during the design process. If the control system is to be made up of purchased hardware, as is often the case in process control work, the designer must still select components with the required parameters from various catalogs.

The objective of this study, outlined in the first paragraph of this section, will be achieved by writing a program meeting the above requirements and comparing its cost (computer cost plus programming

cost) with an estimated cost for a conventional design. This cost comparison must be augmented by consideration of the ability of the two methods to accomplish the design.

Some of the information obtained from the specific control system design study above can be extrapolated to provide information on the feasibility of computer-aided control system design in general.

CHAPTER II

PROCEDURE USED IN FEASIBILITY STUDY

Comparison Method

One method of studying the feasibility of control system design by digital computer would involve selecting a control system, writing a computer program capable of carrying out the specified design, determining the total cost (program development cost plus cost per run) and comparing this cost to an estimated cost for a conventional design of the same system. This was the method used in the reported study.

Assumptions

The above comparison method makes the following assumptions:

1. The control system requires design methods of sufficient complexity to warrant computer solution.
2. The results of the conventional design and the computer design are equivalent.
3. There is no advantage of one design method over the other that cannot be evaluated numerically.

Control System Considerations

Any control system that is eligible for complete design by computer must meet several requirements. The block diagrams for the various configurations of the system must be fairly well standardized.

The subject study is concerned with control systems having known block diagrams. The design consists of finding the right components and the correct gain adjustments, if applicable, for those components. As long as the forms of the block diagrams are known, several different forms can be tried sequentially until an acceptable design is found. No component design will be required. The dynamic equations representing the plant and the control system must be available. Finally, there must be a systematic procedure available for determining the required gain settings.

A partial list of systems for possible use in this study was compiled from the literature. This list is given in Table 1.

Table 1. Systems Considered for Study

PROCESS

Distillate Column Control
Exothermic Reactor Temperature Control
Liquid Level Control

SERVOMECHANISM

Position Control
Speed Control

POWER GENERATION

Combustion Control
Feedwater Flow Control
Superheater and Reheater Temperature Control
Air Heater Temperature Control

It was felt that the control system for this study should be selected from the industrial controls field rather than the military

or aerospace areas, primarily because of the larger market for industrial control systems. Other considerations included the special design requirements connected with most military applications and the uncertainty of the military market.

The power generation control systems and the distillate column control listed in Table 1 were eliminated from consideration in the study because of the lack of usable information on dynamic equations describing the systems and due to the general complexity of the systems. Position and speed controls were eliminated because no complete listing of components was available. A tremendous amount of research would have been required to compile such a list. The exothermic reactor temperature control system was given considerable serious thought, but was eliminated on the basis of discussions with local consulting engineers and representatives of control system component manufacturers. These discussions indicated that the market for exothermic reactor controls was fairly small.

These considerations resulted in the selection of the tank liquid level control system as the system to be used in this study. Liquid level control systems find application where a constant liquid level or head is required. They are widely used in the textile, chemical, petroleum, food, and metals industries. Because liquid level control systems are so widely used, it was felt that there might be enough demand for new liquid level control systems to warrant development of a computer program specifically for their design.

Level control systems require varying degrees of design sophistication. Level tolerances in industry range from fractions of an inch

to several feet. Therefore, component and system costs will vary widely. Overdesign of the system can be fairly expensive.

Another reason for the selection of the liquid level control system was the availability of the dynamic equations representing the response of the system components. Campbell (4), Buckley (5), and Eckman (6) concern themselves to varying degrees with liquid level control.

The amount of available information on components was another factor in the selection of the liquid level control system. One of the key components of a liquid level control system is the control valve; and most valve manufacturers supply publications on the characteristics of their particular product line. Another source is a paper by Forman and Oriolo (7) which contains a large amount of information on control valves.

CHAPTER III

COMPUTER PROGRAM DESCRIPTION

General Description

From a list of available components, the computer program written for this study selects the two most economical sets of components that will meet the given response requirements. This is accomplished by determining a trial CV (valve size coefficient) on the basis of the initial flow into the tank, setting the gain adjustments to predetermined values, and plotting a trial response curve. If the resulting response curve does not meet the design criteria satisfactorily, the trial system is adjusted, either by changing component gains or by changing the type of component used, and the data for the response curve is calculated again. If the response curve is satisfactory, control valves, level sensors, and controllers with approximately the same characteristics as those which resulted in the acceptable response curve found above are selected from a list of components stored on a tape. The characteristics of the two most economical sets of selected components are used in the response curve calculations to verify the acceptability of the resulting control system and, if the system is acceptable, cost, component ordering information, and gain settings are printed as the program output.

If one or both of the two selected sets of components fail to achieve the required response curve characteristics, gain adjustments

are made if possible, or the system or systems are discarded and other sets of components are selected. If no components with the required characteristics are available, a message to that effect is printed, and work on the program stops.

The program simulates the types of control systems available (low cost proportional, general proportional, and proportional plus integral) in order of increasing cost. The first control system to be simulated (the low cost proportional system) requires a proportional band setting of 30 per cent. If the response curve corresponding to a representative low cost system is acceptable, the two most economical sets of hardware that are compatible with the low cost system are selected, and the response curves are run again to verify the acceptability of the two resulting low cost proportional control systems. If the low cost proportional system is not acceptable, the general proportional system (proportional band equal five per cent) is simulated and, if its response is adequate, two sets of components compatible with the general proportional control system are selected. The same procedure is used for the proportional plus integral control systems. In every case, after the response of a representative of one of the three types of control systems is acceptable, the two most economical sets of hardware (control valve, level sensor, and controller) compatible with that type control system are selected and checked in the response calculations. The coordinates of the response curves can be printed if desired.

Component Information

Honeywell components were used exclusively throughout the study

as it was felt that program capability could be demonstrated adequately with only one manufacturer's components.

The following types of control valves are listed on the tape used with the program: angle, double seated, single seated, low flow, Saunders, split body, and cage. Two types of pneumatic actuators and one type of electric actuator are listed. The valve size range covered in the program is from three-fourths to four inches nominal diameter. Foreman (8) states that 96 per cent of all valves used in general chemical or petrochemical facilities are four inches or under. The same reference indicates that butterfly valves are not generally used in sizes less than four inches and none was considered in the study.

A positioner is sometimes used in connection with a valve actuator if the static friction forces which result when the valve is opened are large or when the actuator response is too slow. Basically, the positioner is a form of pilot valve. When a positioner is used, the control air pressure is not applied to the actuator diaphragm as it is when no positioner is used, but is used to vary a nozzle opening which controls the flow of supply air (at pressures of 20 to 100 psig) to the actuator diaphragm. A feedback linkage is required to re-position the flapper when the valve opening corresponding to the input pressure is reached.

Forman also states that positioners are used on all four inch and larger control valves. Therefore, the devised program was set up so that selection of a four inch valve automatically calls for the use of a positioner. Saunders valves used in throttling service require the use of a positioner on all sizes, and the program provides these posi-

tioners automatically. All valves fail shut.

Information on 81 full capacity and reduced capacity v-port and quick-opening control valves is stored on the tape.

Based on the Corrosion Table on pages 74 and 75 of reference (9), 161 possible flowing mediums can be handled by this program. A "Proceed with Caution" message can be seen on the program output (page 16 of this study) after the body material entry of the first system selected in the first example included in this study. Certain combinations of flowing fluids and body materials will cause this message to be printed. The "Proceed with Caution" message indicates that corrosion will result with this combination of fluid and body material, but that it is not dangerously rapid.

Cast iron, bronze, cast steel, and 316 stainless steel are the available body materials for valves included in the study. Available trim materials are stellite, 440C stainless steel, Hastelloy B and Hastelloy C, Durimet 20, Monel, and 316 stainless steel. Radiating fin, bellows seal, and extension column bonnets can be selected by this program.

Honeywell offers a wide range of level sensors, controllers, and displacer-controller combinations. Four types of displacer-controller combinations are available. Since the major difference in these four types of displacer-controllers is the method of mounting, only one type, the top mounted Series 1 unit was included on the component tape. Slight price increases must be added to the cost listed on the program output if one of the more expensive displacer-controllers is desired. Other types of level sensors included are static pressure sensors and a differential pressure-to-current transmitter.

Controllers with three types of control action are included in this study. These three types are on-off, proportional, and proportional plus integral.

The on-off controller, which is the least expensive of the three types, can only maintain a liquid level between two points; it cannot hold a given level. Furthermore, the on-off system must be used with an electrically conductive flowing medium.

The proportional systems used in the study can be divided into two types; a type in which a general controller is used which has an infinite number of gain settings, and a type called the low cost system. Two low cost systems are available from Honeywell: System A and System B. System A is made up of a static pressure transmitter, a positioner-controller which mounts on the control valve actuator, and the control valve. Only control valves with Honeywell 05 actuators can be used in System A. Three possible proportional band settings are available with System A. The value of these three settings is determined by the control valve travel. System B is slightly higher in price than System A, and is made up of a static pressure transmitter, a controller which mounts on the static pressure transmitter, and the control valve. Any Honeywell valve can be used in System B, but only three proportional band settings (30, 50, and 100 per cent) are available.

The most sophisticated controller used in this study is the proportional plus integral controller. Proportional plus integral control is available with a displacer-controller combination (72-25, Series 1) or a general purpose two mode controller used in conjunction with a static pressure transmitter.

Since controller selection is largely a matter of individual preference after the number of control modes required is determined, no attempt was made to list all available controllers on the component tape for this feasibility study. The controller prices listed in the program must be adjusted if special controller features are desired.

Two tank configurations, Configuration One and Configuration Two are used in this study. Configuration Two (Figure 1) requires that flow out of the tank be through a discharge pipe and is the more general configuration. Configuration One, which requires that flow out be from the top of the tank, would be used in a dipping or dyeing process in which material is placed in the tank. When this material leaves the tank, some of the liquid in the tank is removed with it.

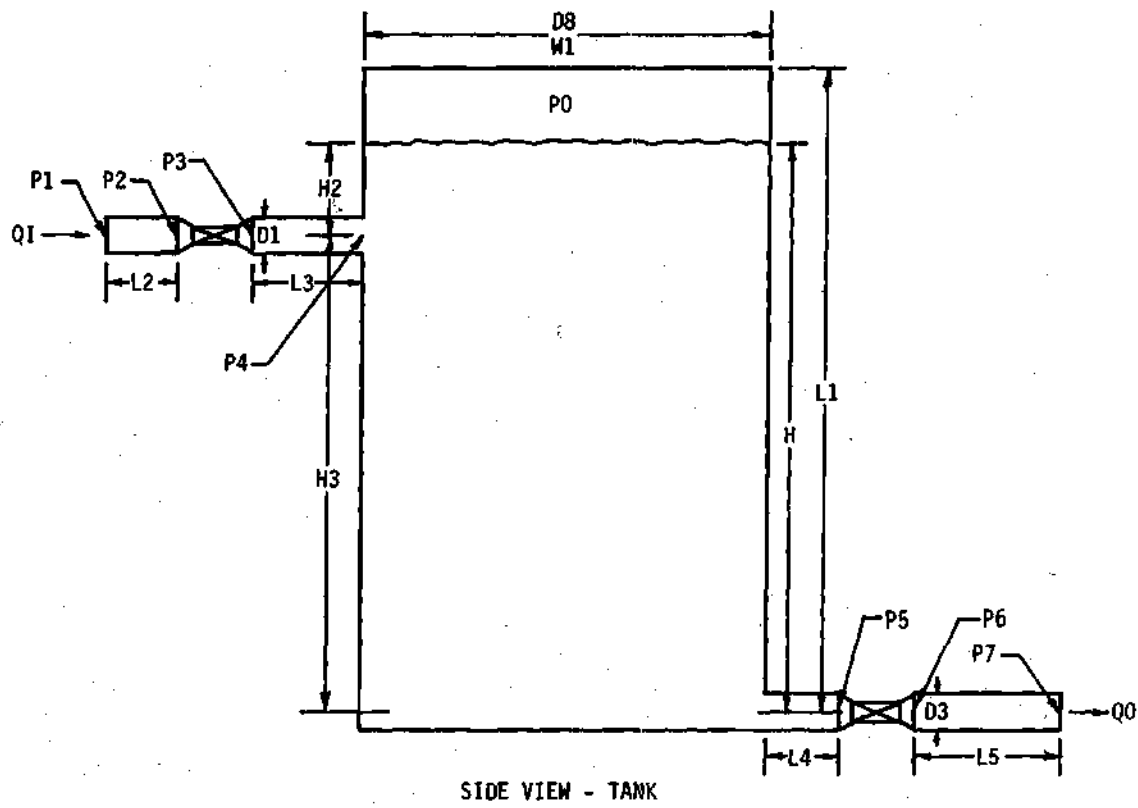
The complete computer program is given in Appendix F.

Program Results for Two Examples

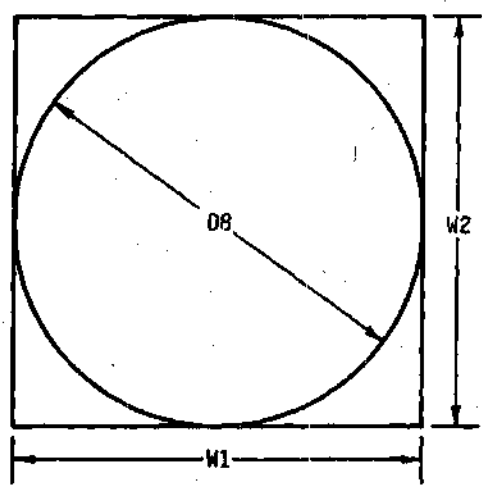
Input information and program results are given below for two examples.

Example One

It is desired to design a system having the following known physical characteristics. Configuration Two was selected because it illustrates more of the program capabilities. The tank used is a 125-lb., vented tank. Water is to be the system fluid, pneumatic control is desired, and a full capacity valve is needed. For the location of the following variables, see Figure 1.



SIDE VIEW - TANK



POSSIBLE TANK CROSS SECTIONS

Figure 1. Diagram of Configuration Two.

Pressures—psia

P0	15
P1	30
P7	17.25

Temperatures—F

T7	80
----	----

Dimensions—Inches

D1	4.0
D3	4.0
DV3	4.0
D8	36.0
H3	12.0
L1	200.0
L2T	150.0
L3T	100.0
L4T	100.0
L5T	150.0

Outlet Valve Size Coefficient—Inches⁴/Lb. Seconds

CVO	654
-----	-----

Pipe Roughness—Inches

E	0.00085
---	---------

Fluid Characteristics

U	1×10^{-7} lb sec/in ²
G	0.036 lb/in ³

The following fittings are included in the indicated pipe sections:

L2 section—1 conventional swing check valve

L3 section—1 conventional disc gate valve, 2-90° pipe bends
(r/d = 4.0)

L4 section—1 in-line ball check valve

L5 section—1 conventional disc gate valve

This system is to meet the following criteria if the required head

H1 is changed by a step change from 100 inches to 125 inches with the

outlet valve fully opened, ($K_{20} = 1.00$).

Required rise time	45 sec.
Maximum overshoot	2.0 in.
Maximum settling time	100 sec.
Maximum steady state error	0.500 in.

The initial conditions are:

$$Q_I = 768 \text{ in}^3/\text{sec.}$$

$$Q_O = 768 \text{ in}^3/\text{sec.}$$

$$H = 100 \text{ in.}$$

$$X_T = 0.82$$

$$H_1 = 125 \text{ in.}$$

The program output (excluding response curve points) for Example One is given below.

Proportional Band = 0.05

System	Component	Cost
ONE	Type 72-25 Series 1	\$ 260.00
	Valve Model No. 1101	
	Size 3.00 in.	
	Type double seated	
	CV 112.00	
	Actuator No. 13	0
	Body Material cast iron	
	(Proceed with caution)	

System	Component	Cost
ONE (Continued)	Body rating 125 lb.	\$ 326.00
	(Flanged end)	
	Plain bonnet	
	Trim material 316 S.S.	<u>0</u>
	TOTAL COST	<u>\$ 586.00</u>
TWO	Type 72-25 Series 1	\$ 260.00
	Valve Model No. 1001	
	Size 3.00 in.	
	Type Angle	
	CV 115.00	
	Actuator No. 13	0
	Body Material 316 S.S.	
	Body Rating 150 lb.	914.00
	(Flanged end)	
	Plain bonnet	
Trim material 316 S.S.	<u>0</u>	
TOTAL COST	<u>\$1,174.00</u>	

Figure 2 shows plots of level versus time for a low cost system and the first of the systems selected by the program, a 72-25 Series 1 system using a Model 1101, three inch, double seated control valve.

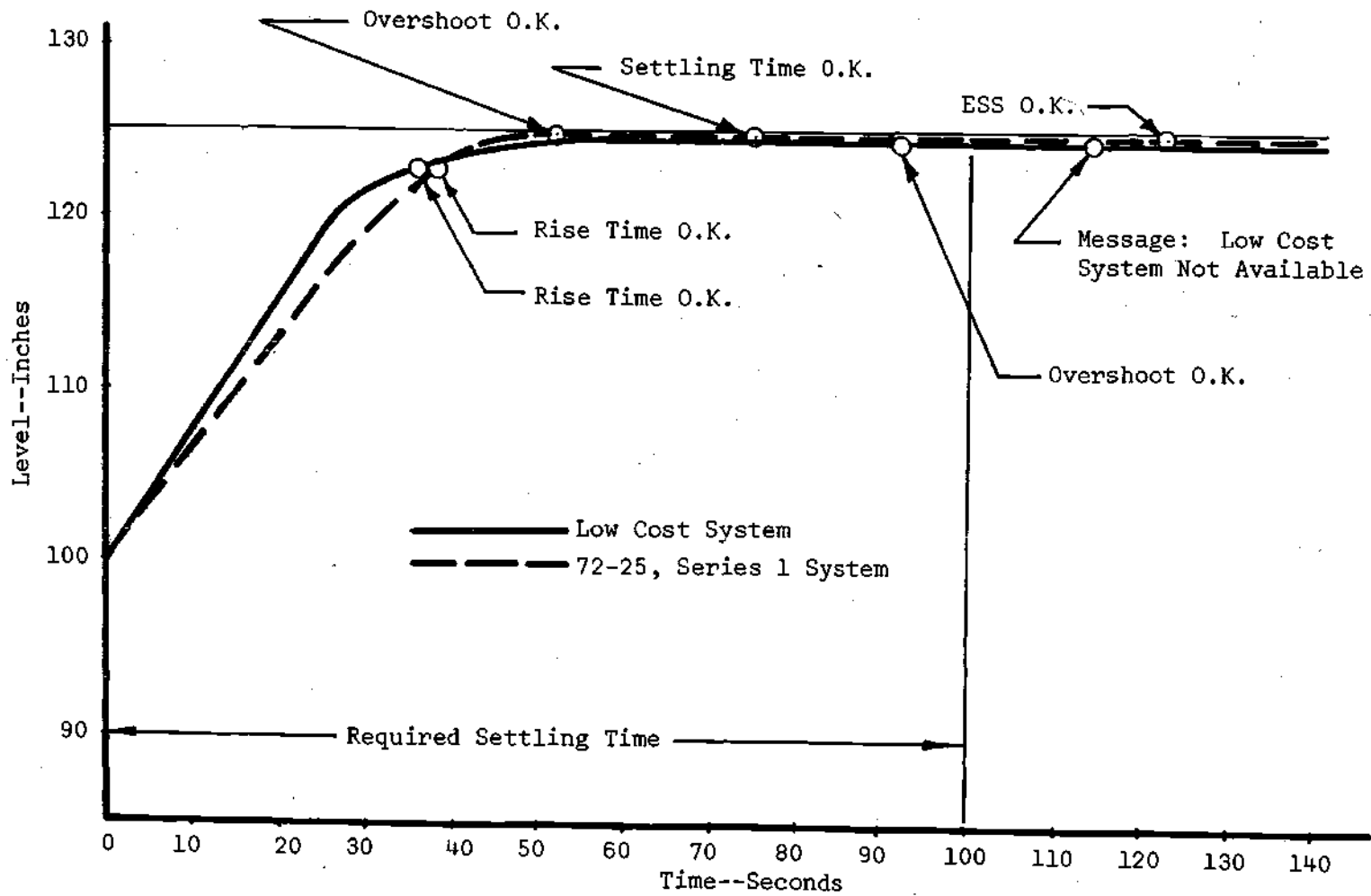


Figure 2. Plot of Level Versus Time for a Low Cost System and a 72-25, Series 1 System.

The low cost system was eliminated by the program because it did not meet the settling time requirement. Inspection of Figure 2 indicates that the low cost system does meet this requirement if it is defined as the time required for the response to stay within a band, centered on the desired level, whose width equals 10 per cent of the initial error. The problem is caused by the fact that the program is set up so initiation of the settling time check must wait until the overshoot check is completed. Because of the response sluggishness caused by the high proportional band requirement for the low cost system, the overshoot check consumed so much time that the maximum allowable settling time was exceeded. It is felt that the program method of checking settling time is acceptable for program demonstration, but, for production use, the program should be changed to allow the overshoot and the settling time checks to run concurrently.

Had the low cost system been selected, an example of the program output would appear as shown below.

Proportional Band = 0.30

System	Component	Cost
ONE	System B	\$ 147.00
	Valve Model No. 1101	
	Size 3.00 in.	
	Type double seated	
	CV 112.00	

System	Component	Cost
	Actuator No. 13	
	Body Material cast iron (Proceed with Caution)	
	Body Rating 125 lb. (Flanged end)	\$ 326.00
	Plain Bonnet	
	Trim Material 316 S.S.	<u>0</u>
	TOTAL COST	<u>\$ 473.00</u>

Figure 2 clearly illustrates the effect of CV and proportional band on the system response. The level of the low cost system, with a CV of 485.80 rises faster than that of the 72-25 system which used a CV of 431.00. Because of its smaller proportional band, 0.05 versus 0.30, the 72-25 system reached its final position much faster. Figure 2 shows that the larger the proportional band used, the more sluggish is the resulting response.

Example Two

It is desired to design a system having the same physical characteristics, initial conditions, desired level change, required rise time, overshoot, and settling time. However, the allowable steady state error is now 0.07 inches.

The program output is given below.

Proportional Band = 0.05

Repeat Rate = 0.50

System	Component	Cost
ONE	Type 72-25 Series 1	\$ 285.00
	Valve Model No. 1101	
	Size 3.00 in.	
	Type double seated	
	CV 112.00	
	Actuator No. 13	0
	Body Material cast iron	
	(Proceed with caution)	
	Body Rating 125 lb.	326.00
	(Flanged end)	
	Plain Bonnet	
Trim Material 316 S.S.	<u>0</u>	
	TOTAL COST \$ <u>611.00</u>	
TWO	Type 72-25 Series 1	\$ 285.00
	Valve Model No. 1001	
	Size 3.00 in.	
	Type Angle	
	CV 115.00	
	Actuator No. 13	0
	Body Material 316 S.S.	
	Body Rating 150 lb.	914.00

System	Component	Cost
TWO (Continued)	(Flanged End)	
	Plain Bonnet	
	Trim Material 316 S.S.	<u>0</u>
		<u>TOTAL COST \$1,199.00</u>

Plots of level versus time for a proportional system and for the first proportional plus integral system selected by the program are given in Figure 3, which indicates that the addition of the integral mode makes the transient response less stable (the curve is now oscillatory) but it forces the steady state error to approach zero. As in Example One, the difference in slopes during the transient response for the two systems is caused by a difference in the CV used.

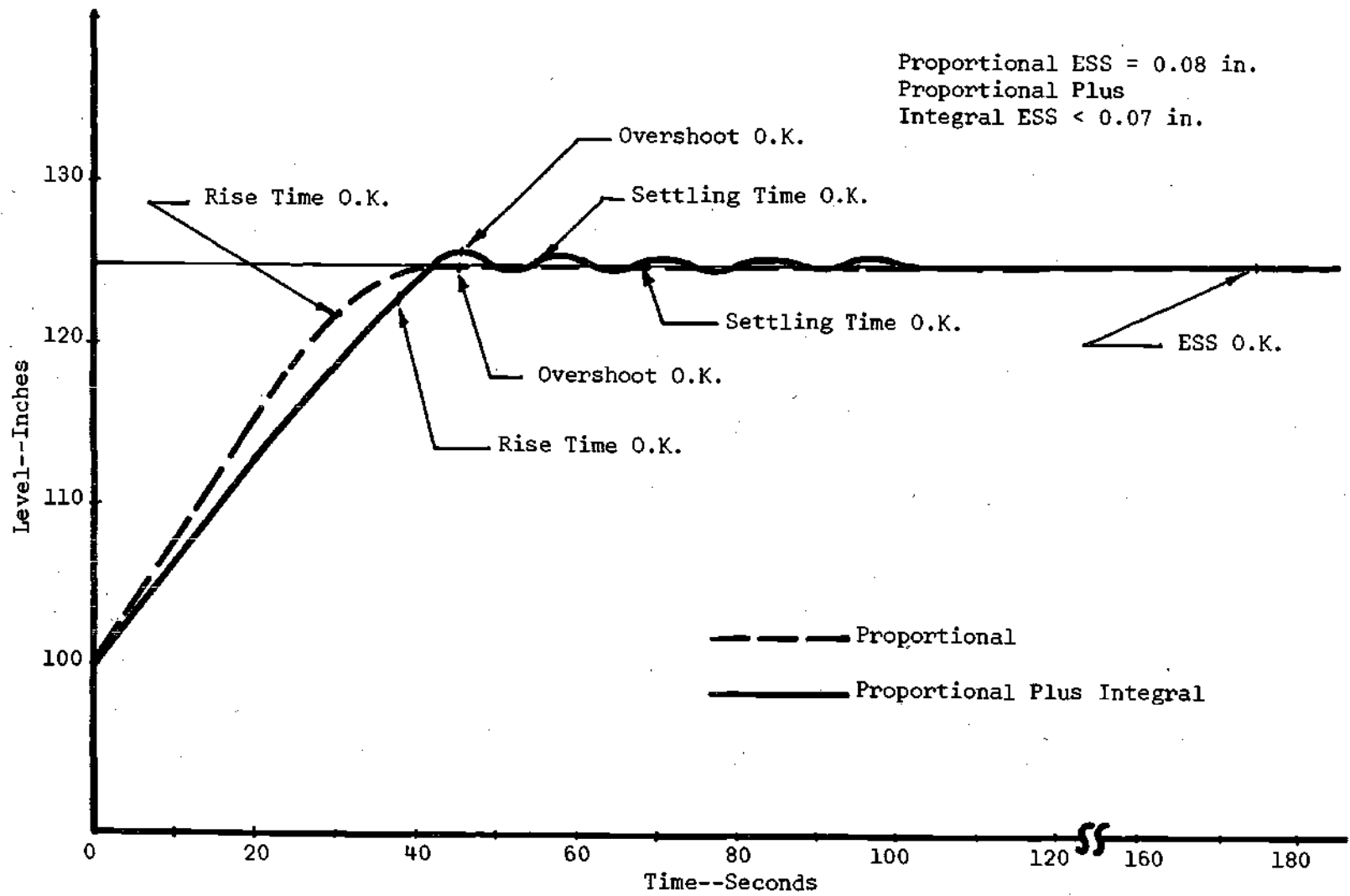


Figure 3. Plot of Level Versus Time for a Proportional System and for the First Proportional Plus Integral System Selected by the Program.

CHAPTER IV

RESULTS OF THE COMPARISON OF COST FOR THE TWO DESIGN METHODS

Neglecting development cost and any costs common to both conventional and computer designs, the cost per design completed on the computer and the cost per conventional design are made up of two quantities.

Computer Design Cost Considerations

The computer design cost consists of a constant cost which covers compilation time, component selection time, input time, and any other miscellaneous times that can be considered constant for all runs, and a variable cost which is a function of the type system used and the time required to test the system response on the computer. For this program, the constant time was found to be approximately 170 seconds.

The variable cost was determined as follows: The cost of problem time on the Burrough's B5500 computer was taken as \$0.04 per problem second. This cost was obtained by dividing the computer cost per hour by 3600 since one problem second took approximately one real time second.

The present program requires that the system response curve coordinates be calculated several times. The number of times that the curve must be determined is a function of the type system necessary to meet the design requirements since the program always tries the different types of control systems in order of increasing cost.

Three calculations of the response curve are required if the low cost proportional system can be used; one general plot which determines the applicability of a representative low cost system, and one run through the response curve for each of the two low cost systems (sets of hardware) selected.

If the low cost system cannot meet any one of the required design criteria, a general proportional control system is simulated. If this general proportional system will satisfy the design criteria, the system response curve will have to be calculated four times; once for the low cost system which could not meet the design criteria, once for the general proportional system, and once for each of the specific sets of proportional system components selected by the program.

In both cases above, no time need be required for setting system gains. The response curve for most practical systems having proportional control is exponential and there is no problem with stability. The proportional band is set at a predetermined minimum and no further adjustment is possible. Gain adjustment is necessary only when simulating a proportional plus integral control system.

Finally, if a proportional plus integral system is required, and it is assumed that the system gains can be adjusted in two trials, seven plots of the system response curves are required. The first checks the response of the low cost proportional system, which is not acceptable. The second calculation of the response curve coordinates is made for a general proportional system which is also not acceptable. The third, fourth, and fifth determinations of the system's response are required to test a general proportional plus integral control system

and to adjust its gains, and the sixth and seventh runs are made with the specific sets of components to be used in the final systems.

It is assumed that the person in charge of installation of the equipment can make the gain settings at the time of installation, since these gains are given on the computer output. If this is true, very little design time is required for tuning the system in the field.

The length of time required per run is assumed to be M , an arbitrary constant found in the program by multiplying the required rise time by four. Although the program is set up so that run time does not reach M in all cases because calculation of the response curve points is stopped when the design requirements are met, no estimate of the shorter time is possible and M must be used. This fact will result in calculated computer design costs slightly higher than true computer design costs.

Mathematically, the above information can be expressed by the following equation:

$$\text{Cost}_{\text{c.d.}} = K_1(\text{Constant time}) + K_1 K_2 M \quad (1)$$

where

$$M = 4(\text{TRR})$$

$$K_1 = \$0.04/\text{second}$$

$$K_2 = \begin{cases} 3 & \text{Low cost system} \\ 4 & \text{Proportional system} \\ 7 & \text{Proportional plus integral system} \end{cases}$$

or

$$\text{Cost}_{c.d.} = \$6.80 + \$0.04 K_2 M \quad (2)$$

Conventional Design Cost Considerations

Conventional design cost is made up of a constant component selection cost and a variable cost due to the time spent tuning the completed control system in the field.

The component selection cost was arrived at as follows: It is assumed that the designer's salary works out to \$4.00 per hour, that the overhead per hour in the office is 100 per cent, and that two hours are required to complete the design using the outline given in Table 2. Therefore the constant design cost will be

$$(2.00)(2 \text{ hours})(\$4.00/\text{hour}) = \$16.00$$

Table 2. Outline of Conventional Design Method for Liquid Level Control System

-
- I. Find Maximum Flow into Tank, QI
 - A. Use maximum flow from tank plus safety factor
 - B. Find the flow rate required to fill the tank in a specified time
 - II. Find Pressure Drop Across the Control Valve
 - A. Assume $\Delta P = P_1 - P_0$ (P_1, P_0 are both known)
 - III. Find CV

Table 2. Outline of Conventional Design Method for Liquid Level Control System (Continued)

$$A. \quad CV = \frac{QI}{\sqrt{\frac{\Delta P}{S.G.}}}$$

IV. Select Control Valves on the Basis of

- A. CV
- B. Body material
- C. Trim material
- D. Maximum allowable pressure drop across the control valve
- E. Maximum allowable static pressure
- F. Bonnet required
- G. Body strength
- H. Availability of Positioner

V. Select Level Sensor on the Basis of

- A. Tank configuration
 - 1. Open
 - 2. Closed
- B. Required control range

VI. Select Controller

- A. Allowable steady state error

System tuning cost is estimated as follows: It is assumed that the field engineer's salary is equivalent to \$4.00 per hour, that the

overhead cost per hour in the field is 200 per cent of the engineer's salary, that the field engineer can tune the system in the same number of trials as used in the computer program, and that the average time required for the field engineer to travel to and from the equipment location is one hour. In contrast to the computer simulation of the control system, on-site calibration must take into consideration the time for the tank to return to its initial condition between trials. This time is taken as M . Therefore, one trial takes M seconds, two trials take $3M$ seconds, and three trials take $5M$ seconds. The coefficients of M just found are called K_u and are listed below as a function of the system used.

Table 3. Value of Constant K_u

System	K_u
Low cost	1
Proportional	1
Proportional plus integral	5

The cost per second of keeping the field engineer on the job is a constant called K_3 . This constant equals $(\$4.00/\text{hour})(3.0)(1/3600 \text{ sec.})$.

The above information can be summarized by the following equation for conventional design cost.

$$\text{Cost}_{m.d.} = \$16.00 + K_3(K_4M + 3600) \quad (3)$$

where K_4 and M are as previously defined.

A plot of Design Cost versus M for the three types of control systems used in this study is shown in Figure 4.

Determination of Program Development Cost

The engineering time required for writing the computer program included here was 1041 hours. The computer time used was 495.82 minutes. It is felt that both the time required for writing the program and the computer time could be cut in half if an experienced person had done the work. This would result in a programming time of approximately 500 hours and a computer time of 250 minutes. If computer time cost \$140 per hour, the engineer's salary was equivalent to \$4.00 per hour, and overhead per hour is 100 per cent of the engineer's salary, the program development cost would be \$4,583.

It is obvious from Figure 4 that even neglecting program development costs, using the present numerical integration technique will result in some computer designs costing more than equivalent conventional designs, since many practical systems will require run times of several hundred seconds. This is due to the fact that the cost per unit of run time (which is one component of the slope of the lines in Figure 4) for the computer design is greater than ten times the cost per unit time for the conventional design.

Since almost all of the run time is spent in performing the numerical integration necessary for solution of the system of differ-

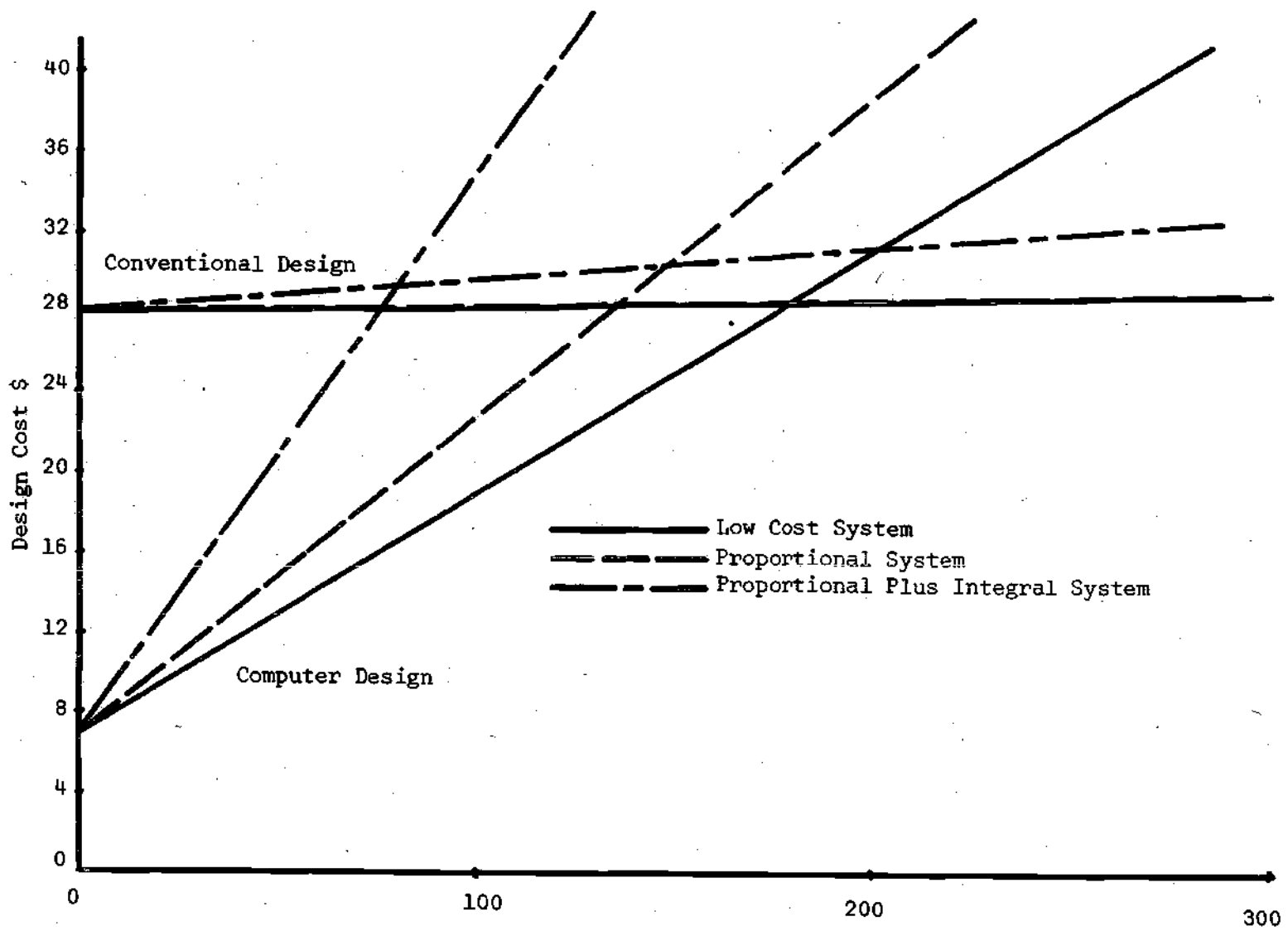


Figure 4. Plot of Design Cost Versus Run Time.

ential equations, the use of any numerical method which requires less computer time than the Runge-Kutta method will make the computer design more competitive.

Because of the lack of clear cut savings in design cost and because of the high program development cost, any justification for computer design of tank liquid level control system must come from consideration of advantages of the computer design not available with the conventional design. That there are several of these advantages will be brought out in Chapter V. Unfortunately, it is extremely difficult to place these advantages on a numerical basis.

CHAPTER V

CONCLUSIONS

Based only on comparative values of cost per design for the two design methods, the computer design cannot compete with the conventional design if the numerical integration method in the present program is used.

This conclusion must be tempered by consideration of other information. Although a large number of levels of design sophistication are possible, all liquid level control systems can be divided into three categories. The first category contains control systems that are only required to maintain a liquid level within a fairly wide band. On-off and proportional controllers are used in this category. The second category is made up of systems that maintain the liquid level fairly accurately, say within a band of one inch or less. The third and most sophisticated of the level control design categories contains control systems for which the transient response as well as close maintenance of the required liquid level is important. Proportional and proportional plus integral controllers are used for the last two categories.

The conventional design method is much less sophisticated than the computer design method. Liquid level control systems are not considered as systems when designed conventionally. The control valve is sized on the basis of the estimated flow necessary to maintain the

desired liquid level. The assumptions of steady flow and constant CV are made.

Controller selection is made strictly on the basis of past experience. If no stand-off error is allowable, it is known that a proportional plus integral controller must be used. If some stand-off error is allowable, the applications engineer must decide whether to use a low cost proportional system, with its limited proportional band range, or a general proportional system.

The average applications engineer is not concerned with determining rise time, amount of overshoot, or settling time. He hopes that the system can be tuned to meet the desired criteria after it is installed in the field.

The computer solution considers the level control system as a system; the group of differential equations representing the control system are solved simultaneously. The control valve is sized on the basis of required rise time and the computer solution takes unsteady flow and the variation of CV with Reynolds number into consideration. The system response is simulated on the computer. Therefore, a system can be selected that will meet any reasonable rise time, overshoot, settling time, and/or steady state error requirements. Since potential systems are tested in order of increasing system cost, the customer is guaranteed that he isn't buying more control system than he needs. Very little time is required to select gains after the system is installed, since approximate gains are obtained from the computer simulation.

Comparison of the three levels of design sophistication mentioned

earlier with the level of sophistication available from the two design methods covered here reveals that the conventional design is all that is required for the first category. Customers interested in control systems in this category are not the least bit interested in rise time and overshoot and there is generally little question about what type of controller to use. The computer design method becomes more attractive if the level of design sophistication is that of category two; not because of any need to find approximate gain settings or any interest in rise time, overshoot, or settling time, but because the computer design guarantees that the least expensive system will be selected. If there is any question at all in the applications engineer's mind about whether to use a proportional or a proportional plus integral system because of a small allowable steady-state error, he will call for a proportional plus integral controller. This is because he never knows exactly what steady-state error the proportional system will have until the system is installed. The customer would be extremely unhappy if the control system had been installed and it could not meet his steady-state error requirements. No problem like this would arise if the system were simulated on the computer. The proper controller can be selected with a high degree of accuracy. System cost for a low cost proportional system can be as much as \$100 less than the cost of a proportional plus integral system.

If the level of design sophistication required corresponds to Category Three, there is no guarantee that the conventional design method will be entirely satisfactory in many cases. If the level control system is part of a large process, the prospect of tuning the

level control system after installation is not pleasant, particularly when adjusting the level control system's gains might necessitate recalibration of the rest of the process. Total system characteristics cannot be determined until the system is installed if the conventional design method is used.

There has been considerable interest generated recently about computer control of processes. Conventional design of control systems included in larger systems controlled by computer is out of the question. Accurate data on system response is necessary for the design of the computer control circuits.

It appears that computer design can only compete with conventional design if the level of sophistication required is that of Category Two or Three. Discussions with manufacturer's representatives indicate that the bulk of the liquid level control system market is in the first category. Although the demand for systems of the second and third level of design sophistication is increasing, it is felt that this demand will fall far short of the number of systems required to justify the large development cost for a program of this nature.

Reference to an on-off control system can be found throughout this study. The original study outline called for the inclusion of an on-off, a proportional, and a proportional plus integral control system in the computer program.

When an example problem using the on-off control system was introduced into the program, the Runge-Kutta method of solution of the equation representing flow into the tank required a time increment of 0.001 seconds for stability when the control valve was in the closed

position. Calculation of response curve points for any normal liquid level control system would be extremely expensive with this small time increment. Since the comparison of available levels of sophistication for the computer and the conventional design methods and the required level of design sophistication for the three categories listed above indicates that on-off control system design by computer is not feasible, it was decided to discontinue development of the on-off control section of the computer program.

Weighing these considerations with the original conclusion; that on a cost per design basis the computer design cannot compete with the conventional design, can only lead to the conclusion that computer design of liquid level control systems using a digital computer program of the type developed for this study is not economically feasible.

If the computer run time could have been reduced by a factor of ten, use of such a computer program might become feasible on a cost per run basis. Three possible methods of decreasing the time required to obtain system response are discussed in Chapter VI

Based on the results of this study, component selection by computer for certain classes of components can be profitable. Honeywell's method of selecting components and finding their cost involves searching through two different sets of books, one listing the components and their characteristics and the other giving their cost. It is estimated that component selection for the program included in this study took approximately one minute of computer time. Equivalent manual selection would have taken approximately one hour. Only classes of components which can be selected on a logical basis would yield such savings. Con-

trol valves are excellent examples of this type of component.

Control system design by computer requires a knowledge of the process dynamic equations. The number of assumptions required in developing the flow equations for the simple system used in this study brought out the lack of basic knowledge about transient flow through valves and fittings.

A systematic method of gain adjustment must be available if the computer is to determine acceptable system response for the simulated control system without adjustment between runs by the system designer. Gain adjustment for the liquid level system used in this study was no problem, but had a control system which included a rate mode been selected, a fairly elaborate trial and error method of gain variation would have been required in the computer program.

Even if the results of this study had indicated that a design program of this nature would have been slightly profitable, it might not be a wise idea to develop such a program for production. Customers for control systems of the type used as an example in this study are extremely conservative. An interview with an instrument engineer for a local consulting engineering firm revealed that his company's policy is to never incorporate any new design features into their designs until these new features have been proven in someone else's installation. Any cost advantage due to computer design can be lost if upon completion of the design, the customer wanted to increase the control valve diameter one size to insure that he will have adequate flow capacity.

Many control systems are sold on the strength of considerations which cannot be evaluated on a numerical basis. One such consideration

is the availability of maintenance and replacement parts. The sales ability of the applications engineer and the personal preference of the customer or his representative are two large considerations since purchases in the price range of most of the control systems considered here (\$1,000 or less) do not have to be justified on a comparative cost basis to management.

CHAPTER VI

RECOMMENDATIONS

It has been found (see Chapter V) that if the time required for the computer to make the calculations necessary to plot the system response curve could be cut by a factor of ten, the computer design would be competitive with the conventional design on a cost per run basis. Three alternate approaches to reducing the computer time required per design are presented here and are recommended as areas in which further profitable research might be done.

The first approach concerns using a faster numerical integration technique for the response curve determination. One candidate for this application would be the modified Euler method. If the modified Euler method were substituted for the Runge-Kutta method, and only two cycles were required per point on the curve, the computer time required would be one half that required for the Runge-Kutta method, which requires four cycles per point.

It is believed that a second approach, concerned with utilization of analog computational techniques, promises greater reductions in computer time required for control system design. In general, the digital computer is slower and more accurate than the analog computer. The value of this increased accuracy is questionable in determination of control system response in many cases. On the other hand, component selection and involved decision making is impossible on the analog

computer. These facts lead one to consider the possible applications of a hybrid digital-analog computer.

A large amount of research is now being directed toward determination of optimization methods. One approach to the determination of optimum gain settings for control systems involves trial and error methods of gain adjustment. The systematic method of gain variation used in the trial and error approach could be programmed into the digital part of a hybrid computer. The digital section would then control the gain settings of the analog part of the hybrid computer on which a large number of trial response curves could be run in a much shorter time than on a digital computer alone.

The third method of attempting to reduce the required computer time is suggested by the results of the tests of the system equations in Appendix B. It can be seen that for the chosen example and the short run time, the approximate method in which pipe friction is ignored resulted in values within 5 per cent of those found by the more rigorous computer solution. Further research could be directed toward determining the applicability of approximate expressions for fluid flow that could be solved in less computer time than the expressions used in this study.

The need for a large amount of experimental work in the field of transient fluid flow, particularly in connection with flow through valves and fittings was demonstrated by the number of assumptions required in the development of the flow equations used in this study. Several articles have been written on control valve selection, but none is satisfactory from an unsteady flow standpoint. Each of the flow assumptions mentioned above must be justified if the computer solution is to

have any practical value. The use of system engineering concepts in process work requires much stronger definitions of some of the basic flow processes than are now available.

APPENDIX

APPENDIX A

SYMBOL TABLE

A	Number of fluid as read from compatibility table on input tape
A8	Area of tank
CAPCTY	Code indicating whether capacity is full or reduced (1--full; zero--reduced)
CHANGE	Value of sign of dH/dT at $T-\Delta T$
COMPAT	Number of control valves which meet system requirements
COMPONENT	Type of low-cost system (A or B)
CONFIG	Code indicating whether system is of configuration 1 or 2 (1--flow out of top of tank; 2--flow out of discharge pipe)
COST	Temporary storage for cost of an item used during printing
CR	Control range
CV1	Value of valve size coefficient at a given Reynolds number (inlet side)
CV3	Value of valve size coefficient at a given Reynolds number (outlet side)
CVCN1	Catalog valve size coefficient (inlet side)
CVCN3	Catalog valve size coefficient (outlet side)
CVMAX	Maximum allowable C_v of control valve
CVMIN	Minimum allowable C_v of control valve
CVO	Outlet valve size coefficient
D1	Pipe diameter on inlet side
D3	Pipe diameter on outlet side
D8	Diameter of circular tank

P	Pressure drop
DELTP1	ΔP required by system
DELTT2	Time increment used in checking system response
AH	Desired band width $\times (1/2)$
DH	
DIFF	H1-initial value of H
DIAP1	Parameter used in Runge-Kutta routine to represent D1
DIAP3	Parameter used in Runge-Kutta routine to represent D3
DIAV1	Parameter used in Runge-Kutta routine to represent diameter of control valve
DIAV3	Parameter used in Runge-Kutta routine to represent diameter of valve on outlet side
DT	ΔT used in Runge-Kutta
DUM	Dummy variable used to store absolute values
DV1	Control valve diameter
DV3	Valve diameter on outlet side
DX1	Used as an index
E	A measure of pipe roughness
ESS	Allowable steady state error
EXT	Code indicating whether extension column is required or not (1--yes; zero--no)
F2	Inlet pipe friction factor
F4	Outlet pipe friction factor
FAC0	} Constant factors used in Runge-Kutta routine
FAC1	
FAC10	
FAC2	
FAC20	

FAC3	} Constant factors used in Runge-Kutta routine
FAC30	
FAC300	
FAC33	
FAC4	
FAC40	
FAC5	
FAC50	
FAC6	
FLOUT	Constant flow out
FMAX	Factor by which catalog valve size coefficient (inlet) is multiplied to get CVMAX
FMIN	Factor by which catalog valve size coefficient (inlet) is multiplied to get CVMIN
G	Specific weight of fluid
H1	Desired level in tank
H1X2	H1-H(t) at a given time
H2	See Figure 3
H3	See Figure 3
I	Used as an index
IND	Used as an index
INITL	Initial value of an index used in a program loop
IFINAL	Final value of an index used in a program loop
J	Used as an index
J1	Used as an index
K	Used as an index
K1	Used as an index
K20	Measure of degree of opening of outlet valve
L	Used as an index
L1	Height of tank
M	Used as an index

MEDM Number of fluid desired by customer

MONEL Code indicating whether a monel bellows seal can be used by the system (1--yes; zero--no)

N Number of differential equations being solved in Runge-Kutta routine

NEXTV Code indicating whether next cheapest valve is to be checked for response characteristics or not (1--yes; zero--no)

NMRTR Storage for numerator of initial CV equation

NVALVS Total number of valves in valve table on input tape

ONOFF Code indicating whether system is on-off or not (1--yes; zero--no)

OP Code indicating whether manual operator is desired or not (1--yes; zero--no)

OS Maximum allowable overshoot

OSOK Code indicating whether overshoot has been checked or not (1--yes; zero--no)

P Used as a factor in Runge-Kutta routine

PO See Figure 1

P1 See Figure 1

P1MAX Maximum value of P1

P4MIN Minimum value of P4

P7 See Figure 1

P8 See Figure 1

P13 Pressure for regulated tank

P15 Allowable system pressure

P30 Allowable system pressure

P40 Allowable system pressure

P60 Allowable system pressure

PB Proportional band

PBMAX Limit on proportional band

PLAIN Code indicating whether plain bonnet is required or not
(1--yes; zero--no)

PNEU Code indicating whether system is pneumatic or not
(1--yes; zero--no)

QI Flow in

QO Flow out

QOM Maximum flow out

QOTIM Parameter used in Runge-Kutta to represent closing time
of a quick-opening valve

QOT Quick opening time

R Valve rangeability
Maximum controllable flow/Minimum controllable flow

RADFIN Code indicating whether radiating fin is required or not
(1--yes; zero--no)

RD r/d for elbows

RE1 Storage for inlet pipe Reynolds number or control valve
Reynolds number, depending upon whether valve has been
chosen or not

RESTART Parameter used in Runge-Kutta to indicate if next cheapest
valve is to be checked for adequate response

RP1 Inlet pipe Reynolds number

RP3 Outlet pipe Reynolds number

RUNGKUTTA Name of Runge-Kutta subroutine

RR Integral gain, repeats per second

RV1 Control valve Reynolds number

RV3 Outlet valve Reynolds number

SAUNDERS Code indicating whether Saunders valve is to be considered
or not (1--yes; zero--no)

SEALI Code indicating whether bellows seal is required or not
(1--yes; zero--no)

SETTLNG Code in Runge-Kutta indicating if settling time has been
checked (1--yes; zero--no)

SLURRY Code indicating whether fluid is a slurry or not
(1--yes; zero--no)

SPECIAL Code indicating whether special transducer is to be ordered
or not (1--yes; zero--no)

STATIC Code indicating if static pressure sensor is desired
(1--yes; zero--no)

STEEL Code indicating whether 316 stainless steel bellows seal
can be used or not (1--yes; zero--no)

STOPX Code indicating whether program is to be stopped at this
point (1--yes; zero--no)

SUML Sum of equivalent lengths of fittings in a length of pipe

T Accumulated time (seconds) in Runge-Kutta

T1 Time accumulated during calculation of on-off XT in
Runge-Kutta

T3 Time accumulated during checking of settling time during the
Runge-Kutta

T4 Time accumulated during checking of steady state error in
Runge-Kutta

T7 Temperature of fluid

TANK Code indicating whether tank is vented, regulated, or trapped
(1--vented; 2--regulated; 3--trapped)

TLAST Time which is equal to present time minus DT in the
Runge-Kutta

TNKRATNG Tank rating

TOP Code indicating whether a manual operator is mounted in top
or side position (1--top; zero--side)

TOTAL Total cost of a system in printing routine

TRO Rise time or difference between TR(2) and TR(1) in Runge-Kutta

TRR Required rise time

TSETR Required settling time

U Fluid viscosity

V1 Motor voltage used computing XT for on-off system

VLAST Previous motor voltage

W1 See Figure 1

W2 See Figure 2

X1M Design flow in

XT x/X , per cent opening for control valve used in program

Z Computer time stored at the beginning of the program

ARRAYS:

ACTCOST Valve actuator or motor cost

ACTNO Valve actuator or motor number

AIR Air-to-diaphragm value of a valve

AVAIL Number of a valve that is usable in the system so far

AVAIL Temporary storage for AVAIL

B Number of material that is compatible with fluid

BCOST Cost of valve body material

BODY Number of available body materials for a valve

BONCOST Valve bonnet costs

CHEAP Number of a valve that has been inserted in the list of valves in order of cost

COL Location of the applicable CV of a valve

COSTX Temperature storage for cost of valve body material

CV Valve CV

CX Factor used in computing rise time in Runge-Kutta

D	Derivative used in Runge-Kutta
DELTP	ΔP of valve
DISCOST	Level sensor cost
DISPL	Control range of level sensor
EXT	Code indicating whether valve has extension column or not
INDX1	Row location of least expensive body material and rating of a valve
INDX2	Column location of least expensive body rating of a valve
LD	Equivalent length of a fitting
LE	Equivalent length of a length of pipe
LIMVFC	Location of last available v-port CV of a full capacity valve
LIMVRC	Location of last available v-port CV of a reduced capacity valve
LT	True length
MAN	Row location of applicable valve manual operator cost
MCOST	Cost of a valve manual operator
MODEL	Model number of a valve
NAME	Alphanumeric name of a body or trim material
NB	Valve body material
NBI	Required body material
NBON	Body material corresponding to a valve bonnet cost
NBX	Temporary storage for cheapest valve body material
NFIT	Number of a type of fitting in one length of pipe
NT	Valve trim material
NTI	Required trim material
NTX	Trim material compatible with fluid
NUMCVS	Total number of available CV's of a valve

OMIT	Number of valves omitted from list of usable valves
ORIGX	Initial value of an element in the X array
POS	Location of a positioner on a valve
POSCOST	Cost of a positioner on a valve
QO	Code indicating whether valve is quick-opening or not
QOLINK	Linkage number of a quick-opening valve
QOTIME	Closing time for a quick-opening valve
RAD	Code indicating whether valve has radiating fin available
RATNG	Body rating of valve
RATNGI	Required body rating
RATNGX	Temporary storage for rating of cheapest body material
SEAL	Code indicating whether valve has bellows seal or not
SIZE	Diameter of a valve
SPRMAX	Maximum pressure to be balanced by valve actuator spring
SPRMIN	Minimum pressure to be balanced by valve actuator spring
TMAX	Maximum allowable ambient temperature for valve
TMCOL	Location of cheapest trim material of valve
TMCOST	Cost of a valve trim material
TMIN	Minimum allowable ambient temperature for valve
TOTCOST	Total cost of a useable valve
TR	Time stored for use in calculating rise time in Runge-Kutta
TRAVL	Valve travel
TSET	Time stored for use in calculating settling time
TYPE	Valve type
VPFC	Number of CVs available with v-port characteristics in a full capacity valve

VPLINK Linkage number of a valve with v-port characteristics

VPORT Code indicating that valve has v-port characteristics

VPRC Number of CVs available with a reduced capacity valve with
v-port characteristics

VPTIME Closing time of a valve with v-port characteristics

X Solution of differential equation in Runge-Kutta

XLAST Solution at T- Δ T of differential equations in Runge_kutta

APPENDIX B

DERIVATION OF SYSTEM FLOW EQUATIONS

Two tank configurations are utilized in this study. The first, Configuration One, represents a system containing a tank discharging from the top. Configuration Two, shown in Figure 1 which is repeated here for convenience, contains a tank which discharges thru an outlet pipe. Configuration Two is illustrated because it is the more general of the two. For the same reason equations of motion will be developed for Configuration Two. Flow into the tank and the liquid level in the tank of Configuration One can be represented by the corresponding equations for Configuration Two if flow from the tank, Q_o , is assumed constant or varies only by step changes.

The tank shown in Figure 1 is typical of those found in industry. It has vertical sides and a round or a rectangular cross section. All pipes are assumed level, and L_2 , L_3 , L_4 , and L_5 are equivalent lengths made up of the true pipe length and the equivalent length of any bends or fittings included in that run of piping. It is assumed that flow will never reverse direction in the systems covered by this derivation. To insure fully developed flow, all sections of pipe must have an L/D ratio greater than 20, where L is the pipe length and D is the inside diameter of the pipe.

The definitions of the symbols used in the following derivations are found in the Nomenclature Table.

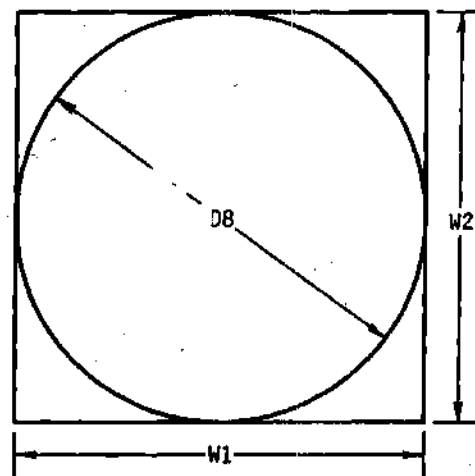
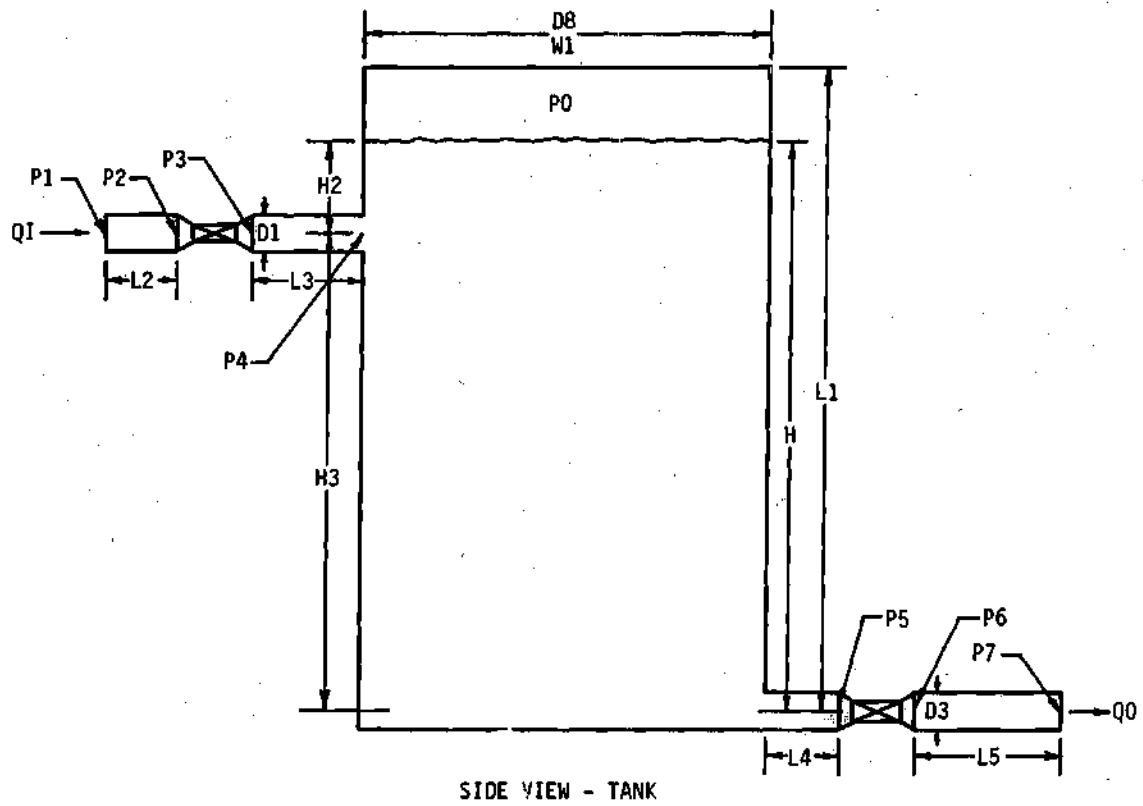


Figure 1. Diagram of Configuration Two.

According to Eckman (10), the flow rate of a liquid through a fully opened control valve is given by

$$Q = CV \sqrt{\frac{\Delta P}{S.G.}} \quad (4)$$

The specific gravity uses the specific weight of water at 60°F as the reference specific weight.

$$S.G. = w/w_w \quad (5)$$

The valve size coefficient CV is defined as the flow rate of water in gallons per minute provided by a pressure differential of one pound per square inch through a fully opened control valve.

Forman (11) recommends the equal-percentage spool characteristic for control valves used in most process situations, primarily on the basis of its good rangeability. Therefore, only control valves with equal-percentage characteristics will be used in the throttling systems of this study. With an equal-percentage spool characteristic, each increment of plug movement produces a change in flow which is proportional to the amount flowing before the change occurred. Rangeability is the ratio of maximum controllable flow to minimum controllable flow. The flow through a sliding stem equal-percentage control valve is given by Eckman (12) as

$$Q/Q_m = R^{(x/X-1)} \quad (6)$$

A plot of flow versus valve lift for an equal-percentage valve is given in Figure 5.

The flow-lift characteristic for Honeywell quick-opening control valves was not available at the time of this writing. For lack of better information, the quick-opening valves were assumed to have approximately linear characteristics described by Eckman (13) as

$$Q/Q_m = 1/R [1 + (R-1)x/X] \quad (7)$$

Let $Q/Q_m = KI$. Combining KI , Equation (4), and Equation (5) results in the following expression for flow through a control valve at any stem position.

$$Q = (KI)CV \sqrt{\frac{w \Delta P}{w}} \quad (8)$$

Applying this equation to the control valve on the inlet side of the tank in Figure 1 yields

$$Q_I = (KI)CVI \sqrt{\frac{w \Delta P_I}{w}} \quad (9)$$

Forman (14) states that control valves are sometimes one pipe size larger or smaller than the pipe in which they are connected. In these cases, reducers are used between the pipe and the control valve. If the friction loss in the reducers is neglected,

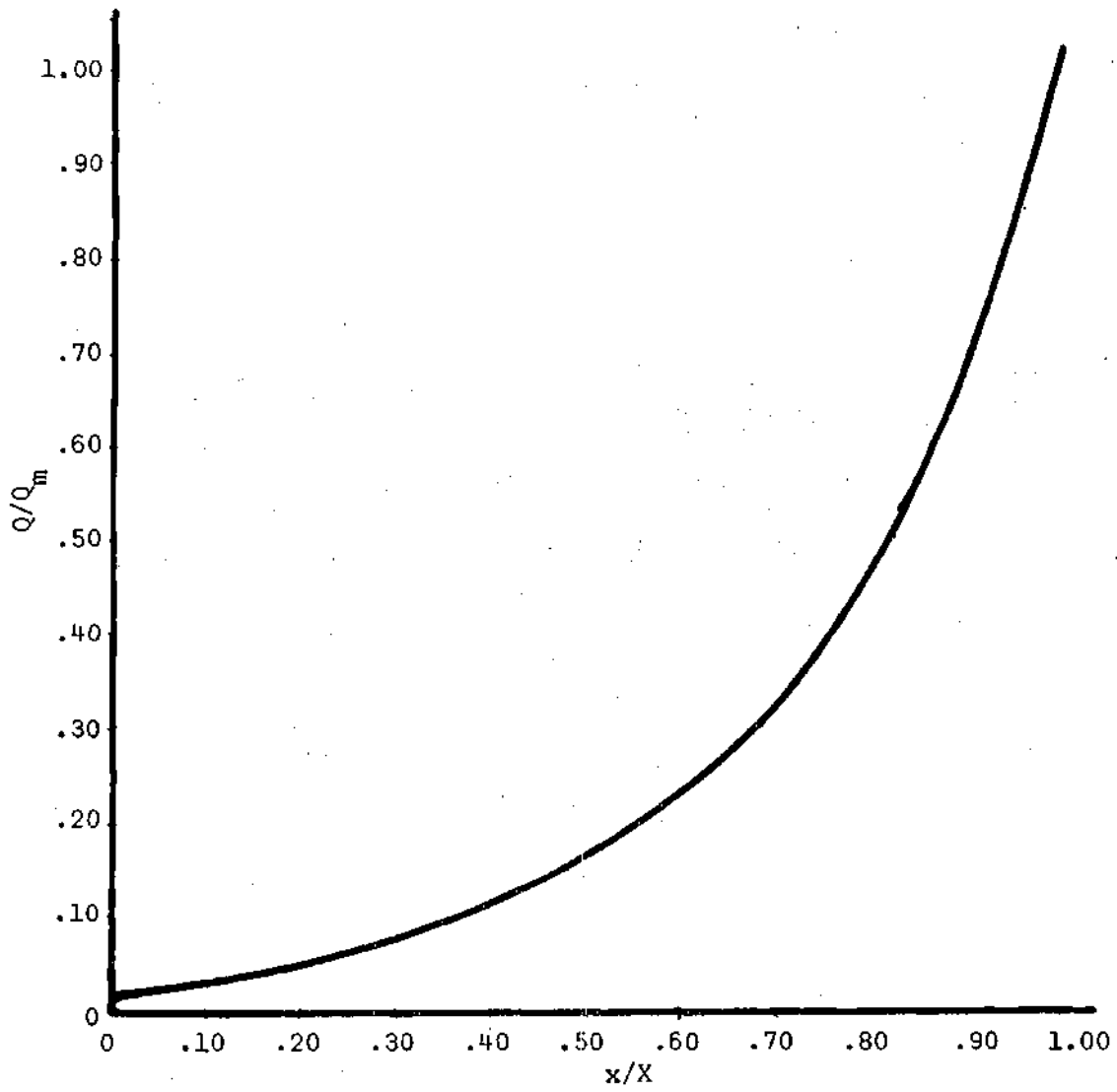


Figure 5. Flow Versus Valve Lift for an Equal-Percentage Control Valve.

$$\Delta P = P_2 - P_3 \quad (10)$$

It can be shown (15) that the momentum equation for fluid flow applied to a control volume R, bounded by a surface S, is

$$\Sigma \vec{F} = \frac{1}{g_c} \left[\frac{d}{dt} \int_R p_R \vec{V}_R d\vec{R} + \int_S (p_S \vec{V}_S) \vec{V}_S \cdot d\vec{S} \right] \quad (11)$$

where $\frac{d}{dt} \int_R p_R \vec{V}_R d\vec{R}$ is the time rate of change of momentum of the fluid within R and bounded by S and $\int_S (p_S \vec{V}_S) \vec{V}_S \cdot d\vec{S}$ is the time rate of net efflux of momentum across the control surface S. Since the flow is uniform and since \vec{V}_1 equals \vec{V}_2 which equals \vec{V}_R (the velocity in the region) if \vec{V}_R is set equal to \vec{V} ,

$$\Sigma \vec{F} = \frac{1}{g_c} \left[\frac{d}{dt} \int_R p_R V_R dR + \dot{m}_2 \vec{V}_2 - \dot{m}_1 \vec{V}_1 \right] \quad (12)$$

Since $\vec{V}_1 = \vec{V}_2 = \vec{V}_R$,

$$\Sigma \vec{F} = \frac{1}{g_c} \left[\frac{d}{dt} (M \vec{V}_R) \right] \quad (13)$$

Because the flow is incompressible,

$$\Sigma \vec{F}_E = M \frac{dV}{dt} \quad (14)$$

Three external forces act on the fluid flowing in the "L2" section of

the pipe as shown in Figure 6. Since all the forces act along the same axis, the vector notation can be dropped and

$$\Sigma F_E = M \frac{dV}{dt} \quad (15)$$

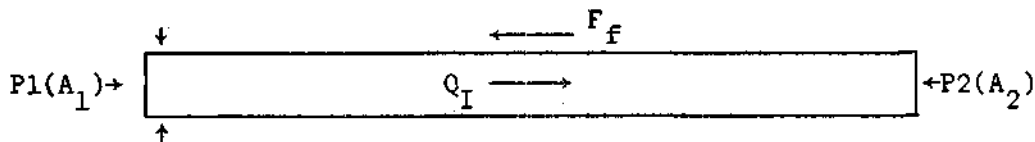


Figure 6. Free-Body of Fluid in Pipe "L2."

The friction force, F_f , is shown in Figure 6. Assuming that resistance to flow in unsteady flow is equal to steady flow resistance at the same velocity (16), the head loss due to resistance is

$$h_f = \frac{FLV^2}{2gD} \quad (16)$$

For the case in Figure 6,

$$V = \frac{Q_I}{A_I} \quad (17)$$

where $A_1 = \frac{\pi}{4} D_1^2$.

The friction force F_f is the product of the head loss, the fluid specific weight, and the cross sectional area of the pipe as expressed by the equation

$$F_f = wA_1 h_f \quad (18)$$

Substituting Equation (16) and Equation (17) into Equation (18) yields

$$F_f = \frac{wF_2(L_2)Q_T^2}{2gD_1 A_1} \quad (19)$$

for the pipe in Figure 6.

The mass of the fluid in the pipe, M, is given by

$$M = \frac{w(L_2)A_1}{g} \quad (20)$$

Since

$$V = \frac{Q_1}{A_1}$$

$$\frac{dV}{dt} = \frac{1}{A_1} \frac{dQ_1}{dt} \quad (21)$$

Substituting Equations (19), (20), and (21) into Equation (15) and rearranging results in

$$P_2 = P_1 - \frac{wF_2(L_2)}{2gD_1 A_1^2} Q_1^2 - \frac{w(L_2T)}{gA_1} \frac{dQ_1}{dt} \quad (22)$$

Likewise, for pipe length L3,

$$P_3 = P_4 + \frac{wF_3(L_3)}{2gD_1 A_1^2} Q_I^2 + \frac{w(L_3T)dQ_I}{A_1 g dt} \quad (23)$$

Combining Equations (22) and (23) and assuming the friction factor in L2 is equal to the friction factor in L3,

$$P_2 - P_3 = P_1 - P_4 - \frac{wF_2}{2D_1 g A_1^2} [L_2 + L_3] Q_I^2 - \frac{w}{g A_1} [L_2 T + L_3 T] \frac{dQ_I}{dt} \quad (24)$$

As can be seen from Figure 1, P4 may be made up of two components: the pressure due to a head of liquid above the pipe inlet in the tank, and the pressure of the gas above the liquid level. Three conditions are covered here. The tank can be vented, i.e., the pressure above the liquid level is taken as 15 psia; the pressure above the liquid in the tank is regulated or maintained at a constant pressure P13; or, the gas above the liquid level is trapped so that the pressure above the liquid is a function of the level. Representing P4 mathematically results in

$$P_4 = P_0 + (H_2)w \quad (25)$$

where

$$P_0 = 15 \text{ psi}$$

Vented

$$= P_{13}$$

Regulated

$$= \frac{(P_{00})L_1}{L_1 - H}$$

Trapped

H2 is defined as follows.

$$H_2 = \begin{cases} 0 & H \leq H_3 \\ H - H_3 & H > H_3 \end{cases} \quad (26)$$

The expression for P_0 for the trapped tank is found as follows. It is assumed that the empty tank is charged to a pressure, P_{00} . Using the perfect gas law,

$$(P_{00})V_{oi} = N R_c T_i \quad (27)$$

where i subscripts indicate initial conditions, and

$$(P_0)V_o = N R_c T \quad (28)$$

Since the tanks covered by this derivation will generally not be insulated, it is assumed that the gas above the liquid will not experience any appreciable temperature change during level changes and the expansion and contraction of this gas will therefore be assumed isothermal. Therefore,

$$(P_0)V_o = (P_{00})V_{oi} \quad (29)$$

where

$$V_o = A_g[L1 - H] \quad (30)$$

and

$$V_{oi} = A_g(L1) \quad (31)$$

Substituting Equation (30) and (31) into Equation (29) yields

$$P_o = \frac{(P_{oo})L1}{L1 - H} \quad (32)$$

Substituting Equation (24) into Equation (9) and rearranging yields the following equation for flow into the tank,

$$\frac{dQ_I}{dt} = \frac{gA_1}{(L2T + L3T)} \left[\frac{P1 - P4}{w} - \left[\frac{1}{((KI)CVI)_{w_w}^2} + \frac{F2(L2 + L3)}{2gD_1(A_1)^2} \right] Q_I^2 \right] \quad (33)$$

The equation to be used for the friction factor, F2, depends on the Reynolds number for flow in the pipe where

$$Re = \frac{4Q_t w}{D_1 U_g}$$

For $Re \leq 2000$,

$$F = \frac{64}{Re} \quad (34)$$

Therefore,

$$F_2 = \frac{16 \pi D_1 U_g}{Q_I w} \quad (35)$$

From Potter (17), if the Reynolds number is greater than 4000, the friction factor F can be found from the equation,

$$F = 0.0055 \left[1 + \left(20,000 \frac{E}{D} + \frac{10^6}{Re} \right)^{\frac{1}{3}} \right] \quad (36)$$

Assuming that the friction factor corresponding to Reynolds numbers in the range between 2000 and 4000 can be found by Equation (36).

For $Re > 2000$,

$$F_2 = 0.0055 \left[1 + \left(20000 \frac{E}{D_1} + \frac{10^6}{Re} \right)^{\frac{1}{3}} \right] \quad (37)$$

As shown in Chapter IV, control valve selection is based in part on a valve size coefficient, CV. This size coefficient is a measure of the flow rate through a control valve as a function of the fluid and pressure drop across the valve. The CV is determined for the flow system under consideration by the following relationship

$$CV = \frac{Q_I}{\sqrt{\frac{\Delta P}{S.G.}}}$$

The size and type of control valve required to meet a specific

application can be selected on the basis of a comparison of this calculated CV and the tabulated CV's found in the valve manufacturer's literature. The valve size coefficients given in the manufacturer's literature have been determined experimentally for each type and size valve.

Although the valve size coefficient is generally given as a constant in the manufacturer's catalogs, in reality, its value varies somewhat with Reynolds number. Even though manufacturer's information on this variation was not available at the time of this writing, a correction to be applied to the manufacturer's constant CV values is included in the computer program for demonstration purposes. The variation of CV with Reynolds number was assumed to be that of a 200 x 400 Venturi meter as given by the curve in the 4th edition of "Fluid Meters: Their Theory and Application" published by the American Society of Mechanical Engineers in 1937. This curve should be replaced with the manufacturer's curves of CV versus Reynolds number for various valves when they are available if the computer program is to be used in practice. The 200 x 400 Venturi meter curve is approximated by straight line segments as shown in Figure 7. In Figure 7, CV is the manufacturer's published valve size coefficient, CVI is the valve size coefficient corrected for the effect of the Reynolds number, and the Reynolds number is the Reynolds number for flow through the valve.

From Figure 7, the following relationships were found. (The program is not applicable for $RE < 200$). For,

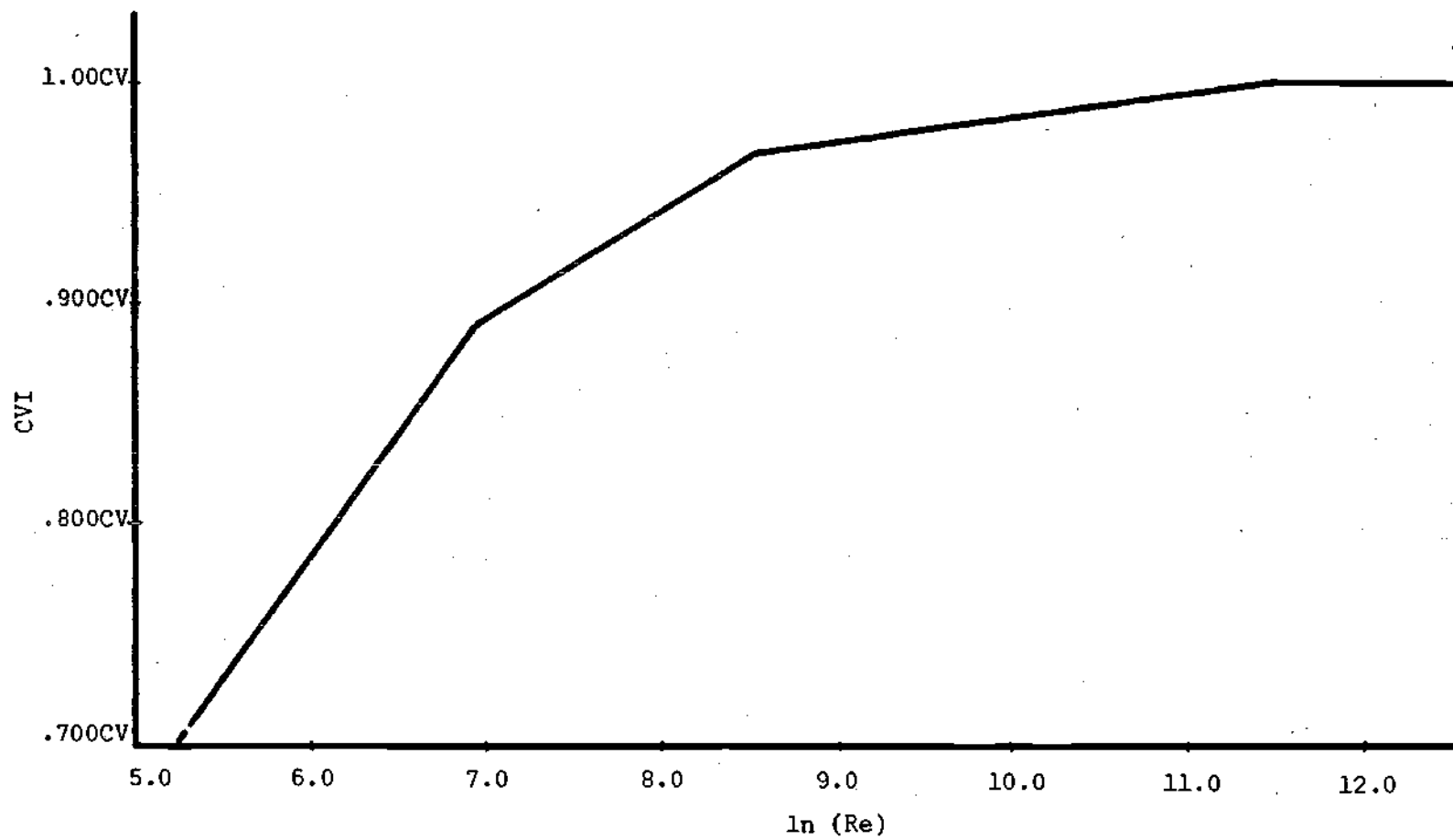


Figure 7. Straight-Line Approximation of Assumed Variation of CVI with Reynolds Number.

$$\begin{array}{ll}
 200 \leq Re \leq 1,000 & CVI=[0.707+0.11 (\ln Re-5.30)]CV \quad (38a) \\
 1,000 \leq Re \leq 5,000 & CVI=[0.884+0.0398 (\ln Re-6.91)]CV \quad (b) \\
 5,000 \leq Re \leq 100,000 & CVI=[0.948+0.01734(\ln Re-8.52)]CV \quad (c) \\
 100,000 \leq Re & CVI=(1.00)CV \quad (d)
 \end{array}$$

The pipe lengths used in the terms containing the friction factors in Equation (33) correspond to the true length of the pipe plus the equivalent length of any bends or fittings included in that length of pipe. For the purpose of the described study, it is assumed that the head loss effect of various fittings is the same in transient flow as in steady flow. It is felt that this assumption is justified for a feasibility study of this sort, but, in practical usage, the error caused by making this assumption should be determined. Table 4 contains a list of various valves and fittings covered by the program along with their equivalent lengths in pipe diameters. This list is taken from Crane (18). The equivalent length for a run of pipe is found by summing up the equivalent lengths in pipe diameters for all the fittings included in that run of pipe and multiplying that sum by the nominal pipe diameter. The equivalent length of a 90° pipe bend must be found from a "Chart for Resistance of 90 Degree Bends" on page A-27 in Crane. Assuming r/d will be greater than or equal to 3.0, but less than 20, the curve has been approximated by a polynomial curve fit computer program which resulted in the following equation,

$$\begin{aligned}
L/D = & 5.9997 \times 10^1 - (5.9953 \times 10^1)r/d + (2.8896 \times 10^1)(r/d)^2 \quad (39) \\
& - (7.2291)(r/d)^3 + (1.0614)(r/d)^4 - (9.3532 \times 10^{-2})(r/d)^5 \\
& + (4.8507 \times 10^{-3})(r/d)^6 - (1.3621 \times 10^{-4})(r/d)^7 \\
& + (1.5959 \times 10^{-6})(r/d)^8
\end{aligned}$$

If the tank is set up as in Configuration Two, the equation for flow from the tank is found in much the same way as that found for flow into the tank. This study assumes that K_{20} , the opening of the outlet valve is a known constant greater than or equal to zero, but less than or equal to one. Then from Equation (8),

$$Q_o = (K_o)CVO \sqrt{\frac{w \Delta P}{w}} \quad (40)$$

It is also assumed that the outlet valve CV varies with Reynolds number as shown in Figure 7, and that friction in the reducers can be neglected.

$$\Delta P_o = P_5 - P_6 \quad (41)$$

Substituting values in Equation (15) yields the following equations for P_5 and P_6 ,

$$P5 = P0 + Hw - \frac{wF4(L4)}{2g D_3(A_3)^2} Q_o^2 - \frac{w(L4T)}{gA_3} \frac{dQ_o}{dt} \quad (42)$$

$$P6 = P7 + \frac{wF4(L5)}{2g D_3(A_3)^2} Q_o^2 + \frac{w(L5T)}{A_3g} \frac{dQ_o}{dt} \quad (43)$$

Combining Equations (40), (41), (42), and (43) results in

$$\frac{dQ_o}{dt} = \frac{gA_3}{(L4T+L5T)} \left[\frac{P0-P7}{w} + H - \left[\frac{1}{w_w((KO)CVO)^2} + \frac{F4(L4+L5)}{2 D_3(A_3)^2 g} \right] Q_o^2 \right] \quad (44)$$

where F4, L4, L5, and CVO are found as on the inlet side.

The change in level in the tank is found by the following equation,

$$\frac{dH}{dt} = \frac{1}{A_8} (Q_I - Q_o) \quad (45)$$

Tests for the validity of the system flow equations derived above for use in the computer program were developed and are described on page 72.

Certain simplifying assumptions can be made to obtain approximate expressions for the time rate of change in head in the tank, for flow into the tank, and for flow from the tank. If the tank outlet valve is shut, ($Q_o = 0$) and the control valve is held open ($KI = 1.0$), an

Table 4. Equivalent Lengths of Valves and Fittings

Type	Sub-Type	<u>VALVES AND FITTINGS</u> Description	Equivalent Lengths L/D
Globe Valves	Conventional	With no obstruction in flat, bevel, or plug type seat. With wing or pin guided disc.	Fully open 340 Fully open 450
	Y-Pattern	(No obstruction in flat, bevel, or plug type seat.) Stem 60 degrees from run of pipe line. Stem 45 degrees from run of pipe line.	Fully open 175 Fully open 145
Angle Valves	Conventional	With no obstruction in flat, bevel, or plug type seat. With wing or pin guided disc.	Fully open 145 Fully open 200
Gate Valves	Conventional		Fully open 13
Check Valves		Conventional swing. Clearway swing. Globe lift or stop. Angle lift or stop. In-line ball.	Fully open 135 Fully open 50 Fully open 450 Fully open 200 Fully open 150
Fittings		90 Degree standard elbow. 45 Degree standard elbow. 90 Degree long radius elbow. 90 Degree street elbow. 45 Degree street elbow. Square corner elbow. Standard Tee With flow through run. Standard Tee With flow through branch.	30 16 20 50 26 57 20 60
Pipe		90 Degree pipe bends.	See Equation

approximate expression for head in the tank as a function of time can be found as follows:

$$\frac{dH}{dT} = \frac{1}{A_8} (Q_I)$$

For steady flow, neglecting friction, with the control valve open,

$$Q_I = (1.0)CVI \sqrt{\frac{(PI-PO)w}{w}}$$

For water,

$$\frac{w}{w} = 1.0$$

Therefore,

$$\frac{dH}{dT} = \frac{CVI}{A_8} \sqrt{PI-PO}$$

$$dH = \left[\frac{CVI}{A_8} \sqrt{PI-PO} \right] dT$$

Integrating,

$$H \Big|_0^H = \left[\frac{CVI}{A_8} \sqrt{PI-PO} \right]_0^T T$$

If the tank is empty at $T = 0$,

$$H = \frac{CVI}{A_8} \sqrt{PI-PO} T$$

Setting

$$P_0 = 15.0$$

$$P_I = 16.6$$

$$A_8 = 1000$$

$$CVI = 654$$

results in

$$H = (0.827) T$$

Table 5 shows the results obtained from the above expression and the corresponding computer solution and the per cent deviation for a five-second run. The per cent deviation uses the computer values as base.

Table 5. Results of the First Flow Equation Test

T	H_{Hand}	H_{Computer}	Per Cent Deviation
0.0	0	0	0
1.0	0.827	0.801578	3.12
1.5	1.240	1.20259	3.08
2.0	1.654	1.60360	3.11
2.5	2.070	2.00461	3.14
3.0	2.480	2.40563	3.08
4.0	3.310	3.02766	3.18
5.0	4.130	4.00968	3.00

Table 5 demonstrates the fairly close agreement between the results of the two methods of solution for the short time interval used. It can be seen that H_{hand} is consistently larger than $H_{computer}$. It is felt that this is due to the inclusion of the effects of pipe friction in the computer solution. This pipe friction is a retarding factor which reduces the flow rate into the tank. Therefore, the tank fills slower than when the pipe friction is ignored.

An approximate expression for the complete set of equations can be found for certain specific cases.

Using the head equation,

$$\frac{dH}{dT} = \frac{1}{A_8} (Q_I - Q_O)$$

If Q_I is held constant and

$$Q_O = (K20)CVO \sqrt{wH}$$

$$\frac{dH}{dT} = \frac{Q_I}{A_8} - \frac{(K20)CVO}{A_8} \sqrt{w} \sqrt{H}$$

Making the following substitutions,

$$u = \sqrt{H}$$

$$K = (K20)CVO \sqrt{w}$$

$$\frac{dH}{dT} = 2u \frac{du}{dT}$$

results in

$$2 A_8 u \frac{du}{dT} = Q_I - ku$$

Separating the variables,

$$2 A_8 \left[\frac{u du}{Q_I - ku} \right] = dT$$

Integrating the above equation and replacing u with H results in

$$T = 2A_8 \left[\frac{\sqrt{H} - \sqrt{H_I}}{k} + \frac{Q_I}{k^2} \ln \left[\frac{\sqrt{H} - Q_I/k}{\sqrt{H_I} - Q_I/k} \right] \right]$$

Table 6 shows comparative values of head and flow out, and lists per cent deviation for the two methods of solution. As before, per cent deviation uses the computer values as base.

It can be seen that there is a very small per cent deviation in head resulting from the two methods of solution. Since the hand calculation neglects pipe friction and the computer solution does not, it can be assumed, that for the specific example used, the effect of the pipe friction on the inlet side of the tank is virtually cancelled by the effect of the pipe friction on the outlet side of the tank. Table 6 also indicates that Q_{oHand} is consistently larger than $Q_{oComputer}$. This can also be explained as the effect of the inclusion of the friction factor in the computer solution.

Table 6. Results of Second Flow Equation Test

T	H _{Hand}	H _{Computer}	Q _{oHand}	Q _{oComputer}	Per Cent Deviation H	Per Cent Deviation Q _o
0.	200	200.000	1755	1755.00	0	0
0.459	199	199.265	1750	1677.20	0.133	4.34
1.048	198	198.338	1745	1667.88	0.170	4.63
2.416	196	196.196	1735	1658.65	0.100	4.60
3.192	194	194.998	1728	1653.57	0.200	4.50
4.240	192	193.370	1718	1646.64	0.713	4.33

APPENDIX C

DEVELOPMENT OF CONTROL EQUATIONS

Based on information obtained from Honeywell catalogs and discussions with local Honeywell engineers, the following Table of System Types was prepared.

Table 7. System Types

Control	Pneumatic	Electric
On-Off		Open or Closed Tank Versa-Tran (Controller) Probes (Sensor)
Proportional	Open Tank Low Cost System A Low Cost System B Type 71-06 Level Control (Displacer) Closed Tank Type 71-06 Level Control (Displacer)	Open or Closed Tank R7165A Proportioning Relay (Controller) P/I Transmitter (Sensor)
Proportional Plus Integral	Open Tank Type 71-06 Level Control (Displacer) 2-Mode Indicating Controller, Statis Pressure Transmitter (Sensor)	
Proportional Plus Integral	Closed Tank Type 71-06 Level Control (Displacer)	

It should be noted that no systems containing three mode controllers are included. Eckman (19), Buckley (20), and Honeywell (21) indicate that the rate mode is rarely used in liquid level control.

It can be seen from Table 7 that only proportional systems have both electric and pneumatic components. This does not mean that Honeywell cannot provide components from which an electric proportional plus integral control system, for example, could be made. It does indicate that the frequency of demand for such a system is so low that applications engineers at the local level do not have full information on the system and considerable correspondence with the factory would be required for the purchase of such a system.

The set of equations governing the valve opening for an on-off control system is obtained as follows. Figure 8 is a diagram of the on-off control system used in this study. It can be seen that this type of liquid level control can only maintain a level between two points; it cannot hold the level at a specified location. Honeywell on-off liquid level control systems require that the liquid used be an electrical conductor. The relationship of motor voltage V versus the error $(H_1 - H)$, in the liquid level is given in Figure 9.

A design requirement for the systems of this study was that all control valves fail shut. This means that all valves must be of the spring return type for which closing time is assumed negligible. On the other hand, opening time is a linear function of time. The motor-operated valves used in the study have opening times of 30 and 60 seconds.

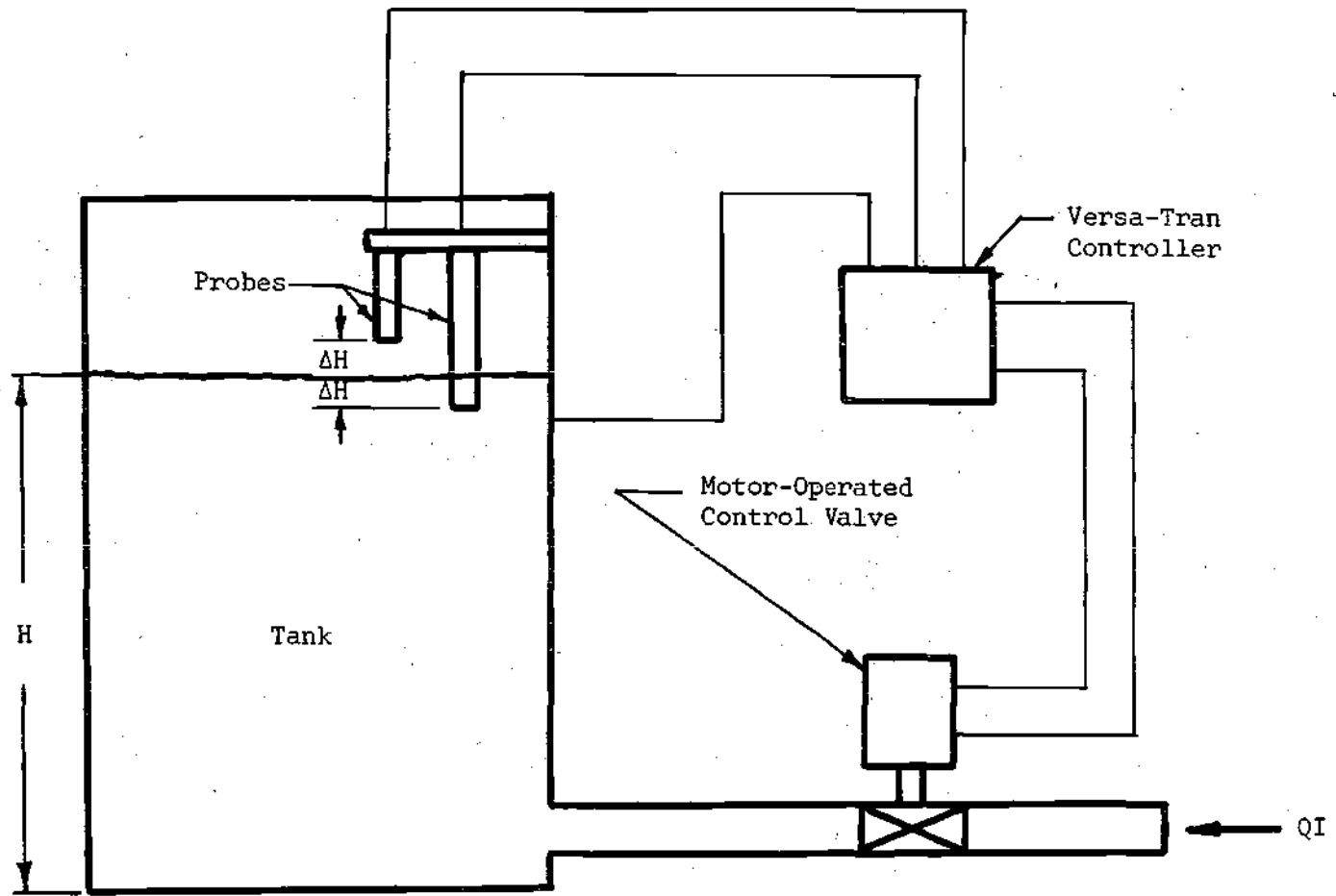


Figure 8. On-Off Control System Diagram.

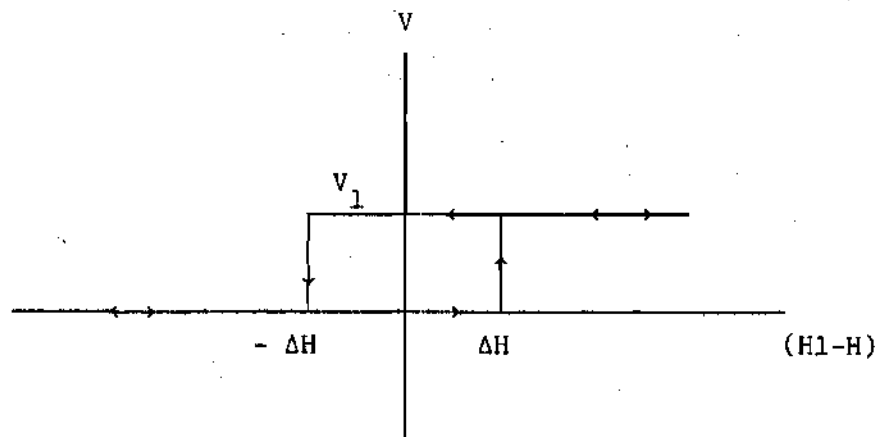


Figure 9. Plot of Motor Voltage Versus Error.

As shown in Figure 9, the voltage V , to the valve motor is not only a function of the error in the head $(H1-H)$, but is also a function of the voltage past history. In addition, the opening of the valve is a function of the voltage applied to the valve and a function of the time elapsed since the voltage was applied. With this in mind, the following set of equations can be written.

For $(H1-H) \leq -\Delta H$,

$$V = 0$$

and

$$\frac{x}{X} = 0 \quad (46)$$

For $-\Delta H < (H1-H) < \Delta H$

and $V(T-\Delta T) = 0$,

$$V = 0$$

and

$$\frac{x}{X} = 0 \quad (47)$$

For $-\Delta H < (H_L - H) < \Delta H$

and $V(T-\Delta T) = V_1$,

$$V = V_1$$

and if $T_1 \leq QOT$,

$$\frac{x}{X} = \frac{T_1}{QOT} \quad (48)$$

if $T_1 > QOT$,

$$\frac{x}{X} = 1.0 \quad (49)$$

For $(H_L - H) \geq \Delta H$,

$$V = V_1$$

and if $T1 \leq QOT$,

$$\frac{x}{X} = \frac{T1}{QOT} \quad (50)$$

or if $T1 > QOT$,

$$\frac{x}{X} = 1.0 \quad (51)$$

The control equation used for the proportional systems of this study is derived for a pneumatic system but can be used for an electric system.

According to Buckley (22), the natural frequency of a displacer type level sensor is one to three cycles per second. Since this frequency is much higher than the frequency of the tank level changes for any practical liquid level system, the transfer function for the displacer is represented by a constant. This same reasoning also holds for the other level sensors included in the study. For the purpose of the study, the system is assumed to be set up so that a zero error signal corresponds to flow through the control valve equal to one half the maximum flow for a given pressure drop across the valve.

From Figure 5 in Appendix B, when $Q/Q_m = \frac{1}{2}$, $x/X = 0.82$ for an equal-percentage control valve. The relationship between x/X and P is shown in Figure 10. From Figure 10,

$$P = P_{mi} + (P_{ma} - P_{mi}) \frac{x}{X} \quad (52)$$

The relationship between P and $(H1-H)$ is shown in Figure 11. Point P_c can be found by setting x/X equal to 0.82 in Equation (52). In Figure 11,

$$P = \frac{P_{ma} - P_{mi}}{PB(CR)} (H1-H) + P_c \quad (53)$$

Eliminating P between Equation (52) and Equation (53) yields

$$\frac{x}{X} = \frac{(H1-H)}{PB(CR)} + 0.82 \quad (54)$$

Because of the limiting nonlinearities in Figure 11, the expression for x/X for the entire range of allowable values is

$$\frac{x}{X} = \begin{cases} 0 & \frac{(H1-H)}{PB(CR)} + 0.82 \leq 0 \\ \frac{(H1-H)}{PB(CR)} + 0.82 & 0 < \frac{(H1-H)}{PB(CR)} + 0.82 < 1 \\ 1 & \frac{(H1-H)}{PB(CR)} + 0.82 \geq 1 \end{cases} \quad (55)$$

The most sophisticated control system used in this study uses a two mode controller incorporating a proportional mode and an integral or reset mode. The addition of the integral mode to the proportional controller results in the following control equation,

$$\frac{x}{X} = \frac{1}{PB(CR)} \left[RR \int_0^t (H1-H) dt + (H1-H) \right] + 0.82 \quad (56)$$

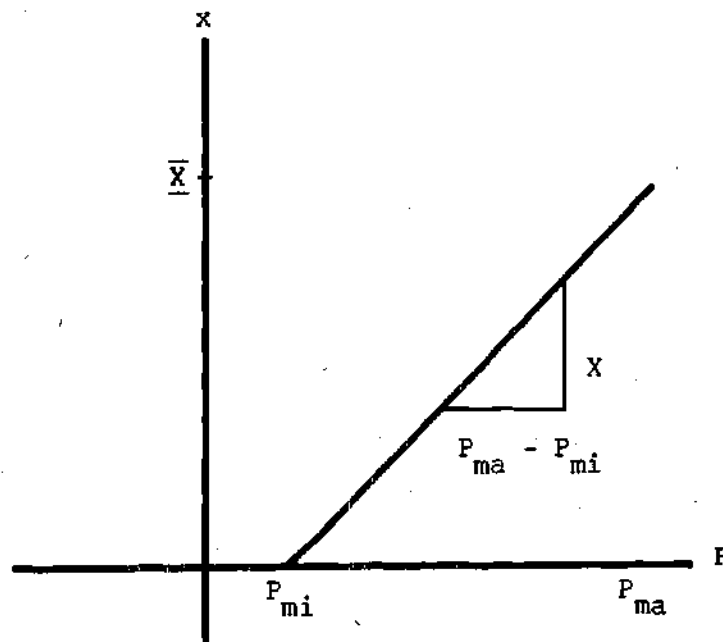


Figure 10. Plot of Valve Opening Versus Control Pressure.

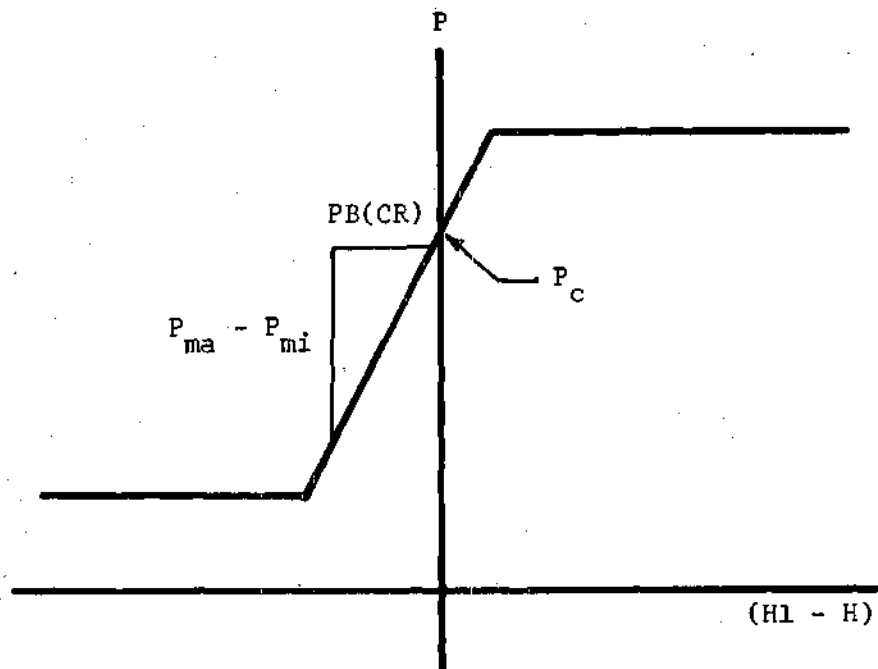


Figure 11. Plot of Control Pressure Versus Error in Level.

Again, the range of x/X is limited and, if

$$\frac{1}{PB(CR)} \left[RR \int_0^t (H1-H)dt + (H1-H) \right] + 0.82 = A ,$$

the complete proportional plus integral control equation is

$$\frac{x}{X} = \begin{cases} 0 & A \leq 0 \\ \frac{1}{PB(CR)} \left[RR \int_0^t (H1-H)dt + (H1-H) \right] + 0.82 & 0 < A < 1 \\ 1.0 & 1 \leq A \end{cases} \quad (57)$$

APPENDIX D

THEORY OF DIFFERENTIAL EQUATION SOLUTION

Determination of the time response of the tank liquid level requires the simultaneous solution of a system of first order nonlinear differential equations. The number of differential equations making up the system depends on the tank configuration and on the type of control system used.

The Runge-Kutta method of numerical integration is used to obtain the required solutions. A brief explanation of this method follows (23). Given $y' = f(x,y)$ and (x_0, y_0) , additional points (x_1, y_1) on the integral curve are desired. Let $\Delta x = h$ and $\Delta y = k$ so that

$$y_1 = y_0 + k \quad (58)$$

and

$$x_1 = x_0 + h \quad (59)$$

The problem is now reduced to finding values of k to the required degree of accuracy. There have been several methods presented for finding k based on the Taylor series expansion

$$k = y_0' h + y_0'' \frac{h^2}{2!} + y_0''' \frac{h^3}{3!} + y_0^{iv} \frac{h^4}{4!} + \dots \quad (60)$$

The easiest to calculate are the formulas of Runge, which have been modified by Heun and Kutta. The chief advantage of these formulas is that they can be written with the functional values only. Let k_1, k_2, k_3, \dots , be the successive approximations of k , which will be weighted and averaged as indicated by Equation (65) below. The formulas as modified by Kutta are

$$k_1 = f(x_0, y_0)h \quad (61)$$

$$k_2 = f\left(x_0 + \frac{h}{2}, y_0 + \frac{k_1}{2}\right)h \quad (62)$$

$$k_3 = f\left(x_0 + \frac{h}{2}, y_0 + \frac{k_2}{2}\right)h \quad (63)$$

$$k_4 = f(x_0 + h, y_0 + k_3)h \quad (64)$$

and

$$k = \frac{1}{6} (k_1 + 2k_2 + 2k_3 + k_4) \quad (65)$$

If y does not appear on the right side of the differential equation, the problem reduces to simple quadrature and Kutta's equations take the form of Simpson's rule.

The method can be extended to systems of simultaneous differential equations. For example, consider two simultaneous equations

$$y' = f(x, y, z) \quad (66)$$

and

$$z' = g(x, y, z) \quad (67)$$

By defining q such that

$$z_1 = z_0 + q \quad (68)$$

k and q can be written as follows:

$$k_1 = f(x_0, y_0, z_0)h \quad (69)$$

$$q_1 = g(x_0, y_0, z_0)h \quad (70)$$

$$k_2 = f\left(x_0 + \frac{h}{2}, y_0 + \frac{k_1}{2}, z_0 + \frac{q_1}{2}\right)h \quad (71)$$

$$q_2 = g\left(x_0 + \frac{h}{2}, y_0 + \frac{k_1}{2}, z_0 + \frac{q_1}{2}\right)h \quad (72)$$

$$k_3 = f\left(x_0 + \frac{h}{2}, y_0 + \frac{k_2}{2}, z_0 + \frac{q_2}{2}\right)h \quad (73)$$

$$q_3 = g\left(x_0 + \frac{h}{2}, y_0 + \frac{k_2}{2}, z_0 + \frac{q_2}{2}\right)h \quad (74)$$

$$k_4 = f(x_0 + h, y_0 + k_3, z_0 + q_3)h \quad (75)$$

$$q_4 = g(x_0 + h, y_0 + k_3, z_0 + q_3)h \quad (76)$$

and

$$k = \frac{1}{6}(k_1 + 2k_2 + 2k_3 + k_4) \quad (77)$$

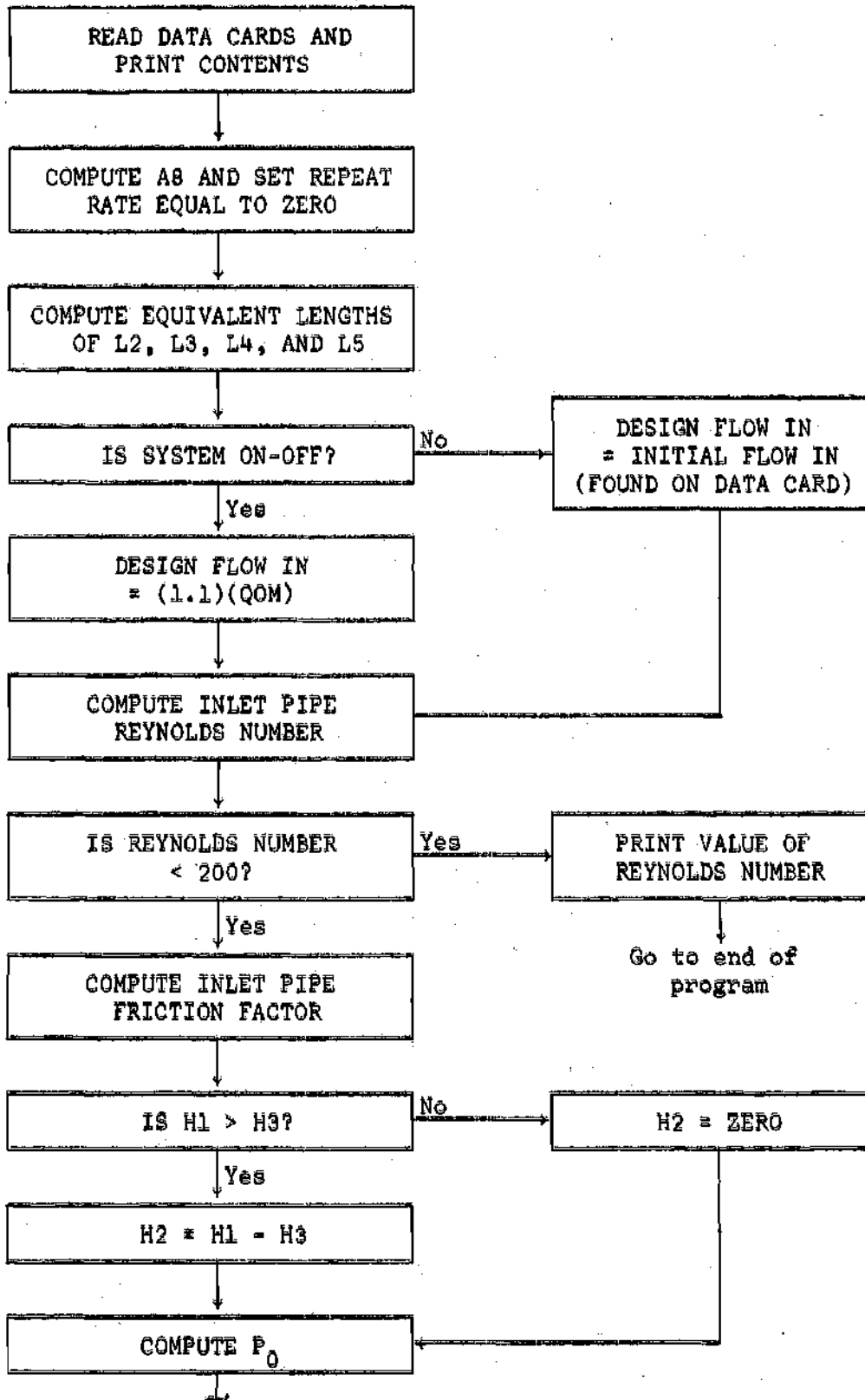
$$q = \frac{1}{6}(q_1 + 2q_2 + 2q_3 + q_4) \quad (78)$$

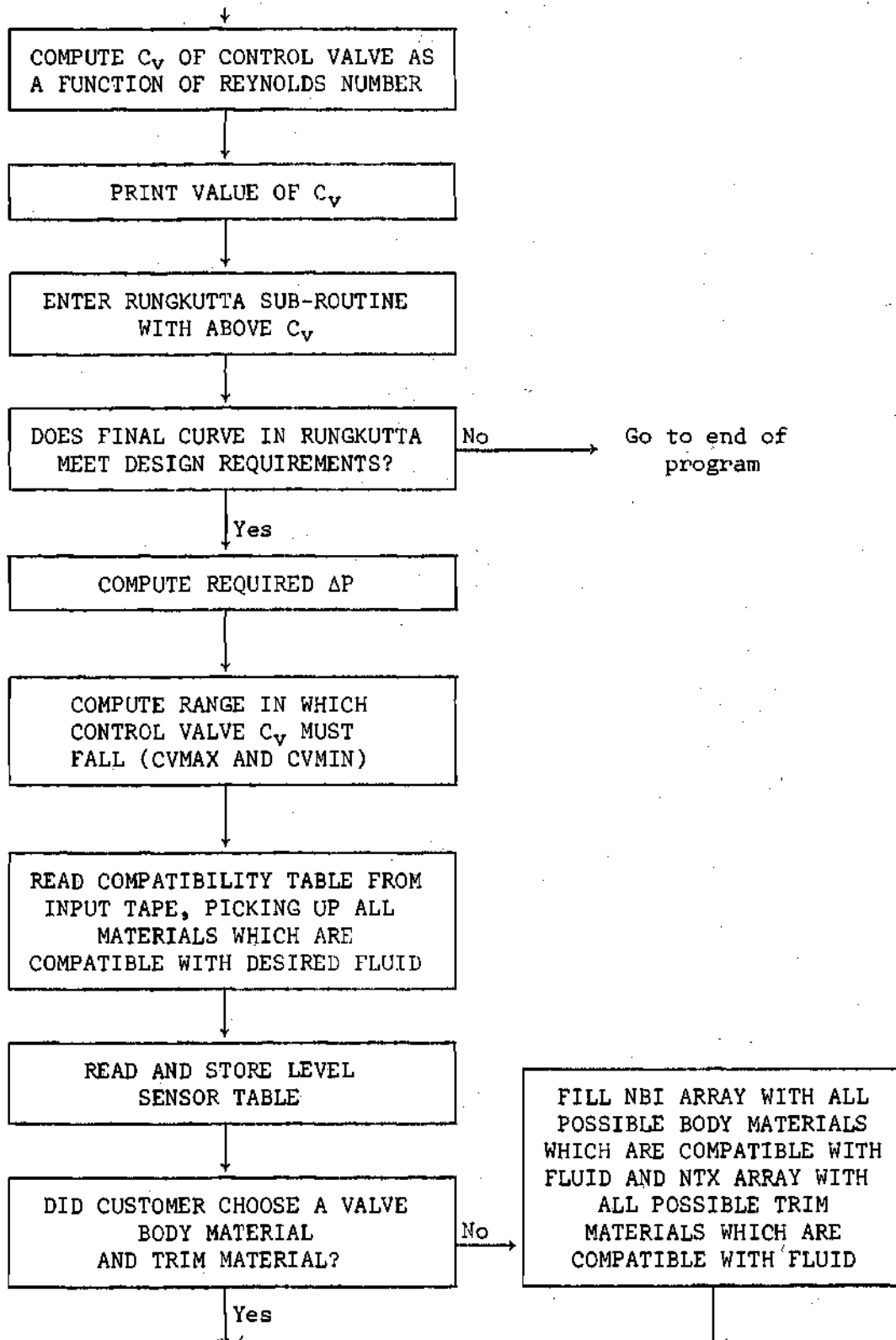
It can be shown that the error in these approximations is proportional to h to the fifth power (24). A simpler but less sophisticated method of determining the error involves re-running the calculation for a short interval of x at half the increment h . The number of significant figures found in agreement is an indication of the accuracy of the results.

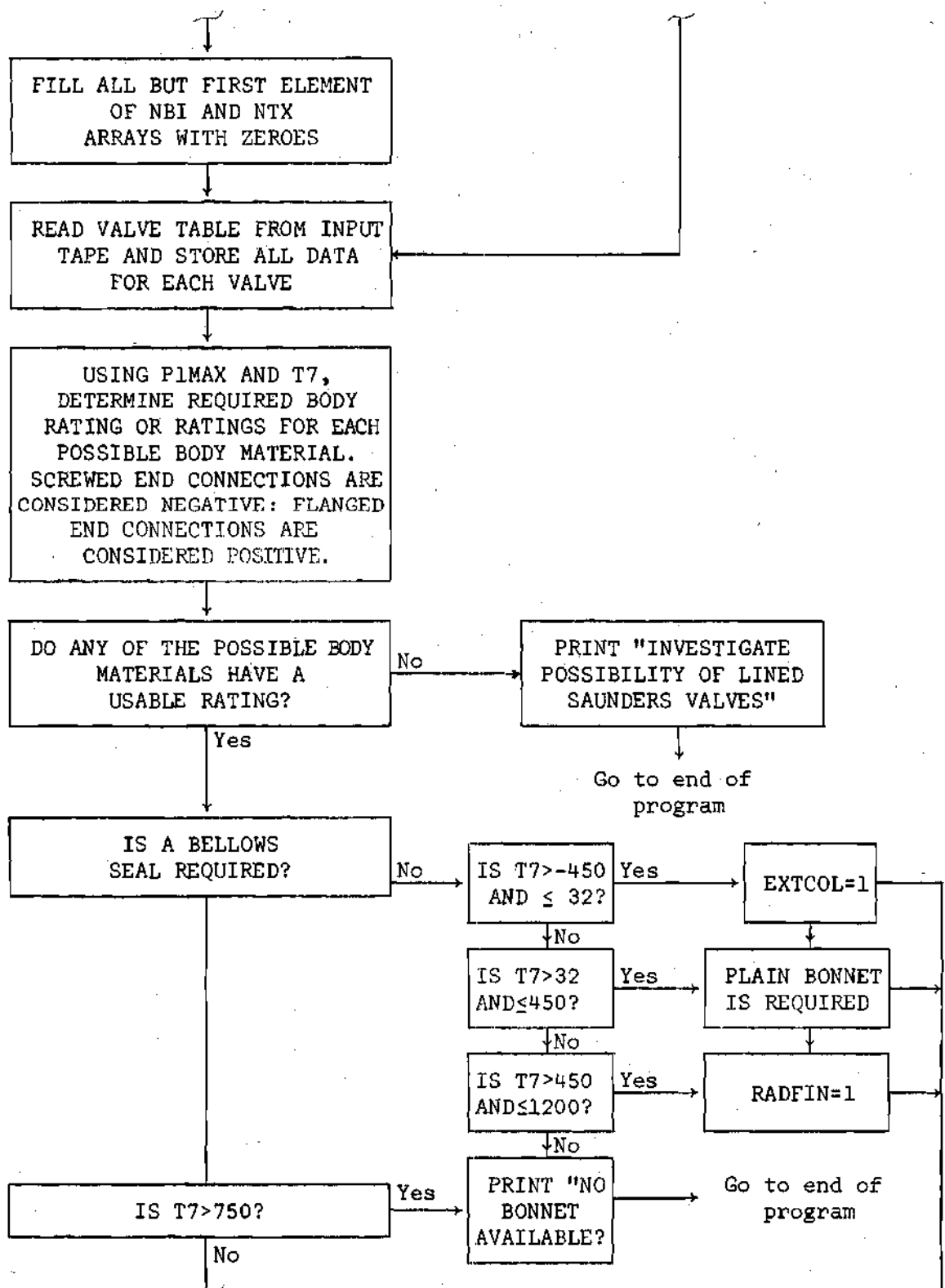
APPENDIX E

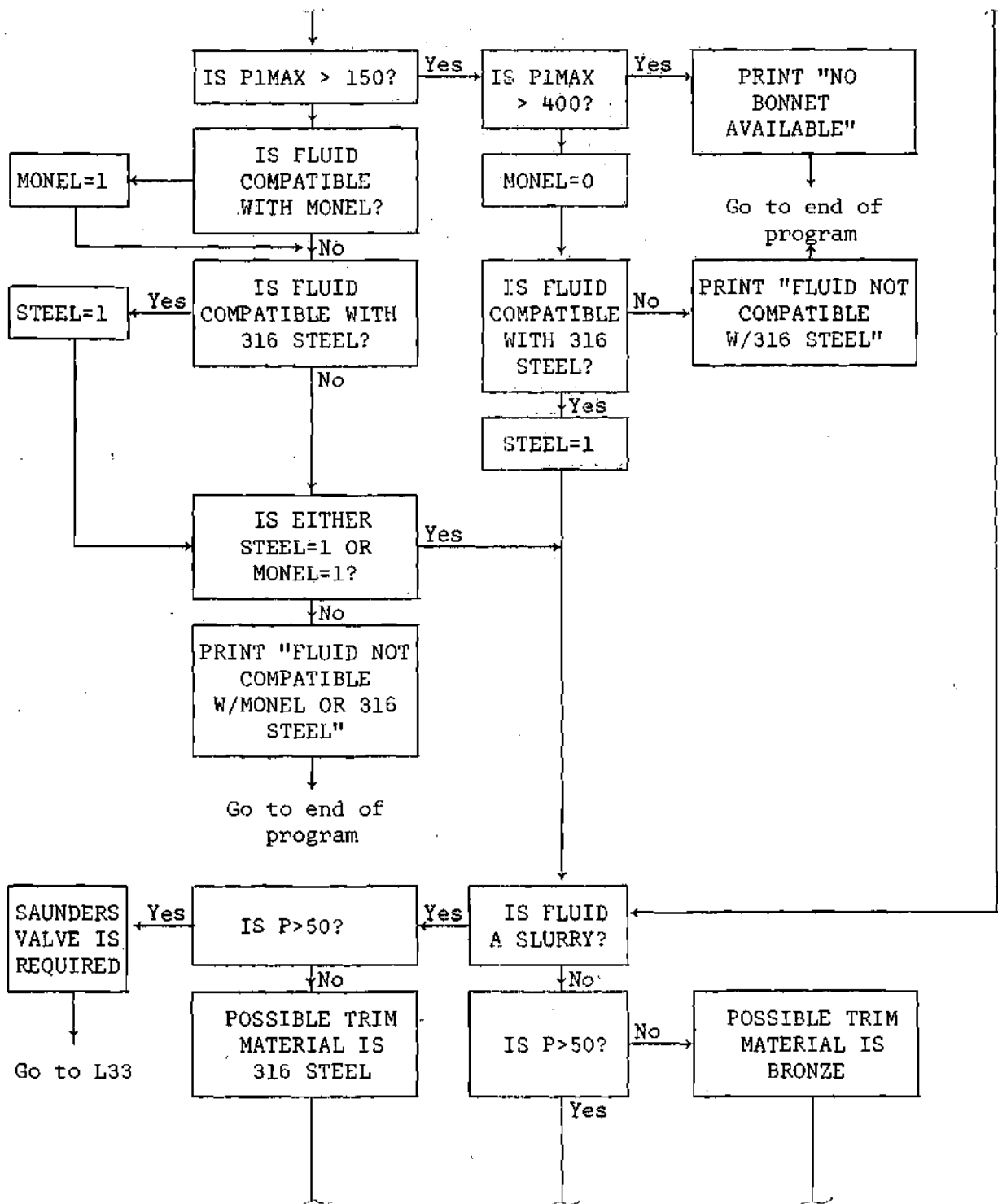
PROGRAM FLOW CHARTS

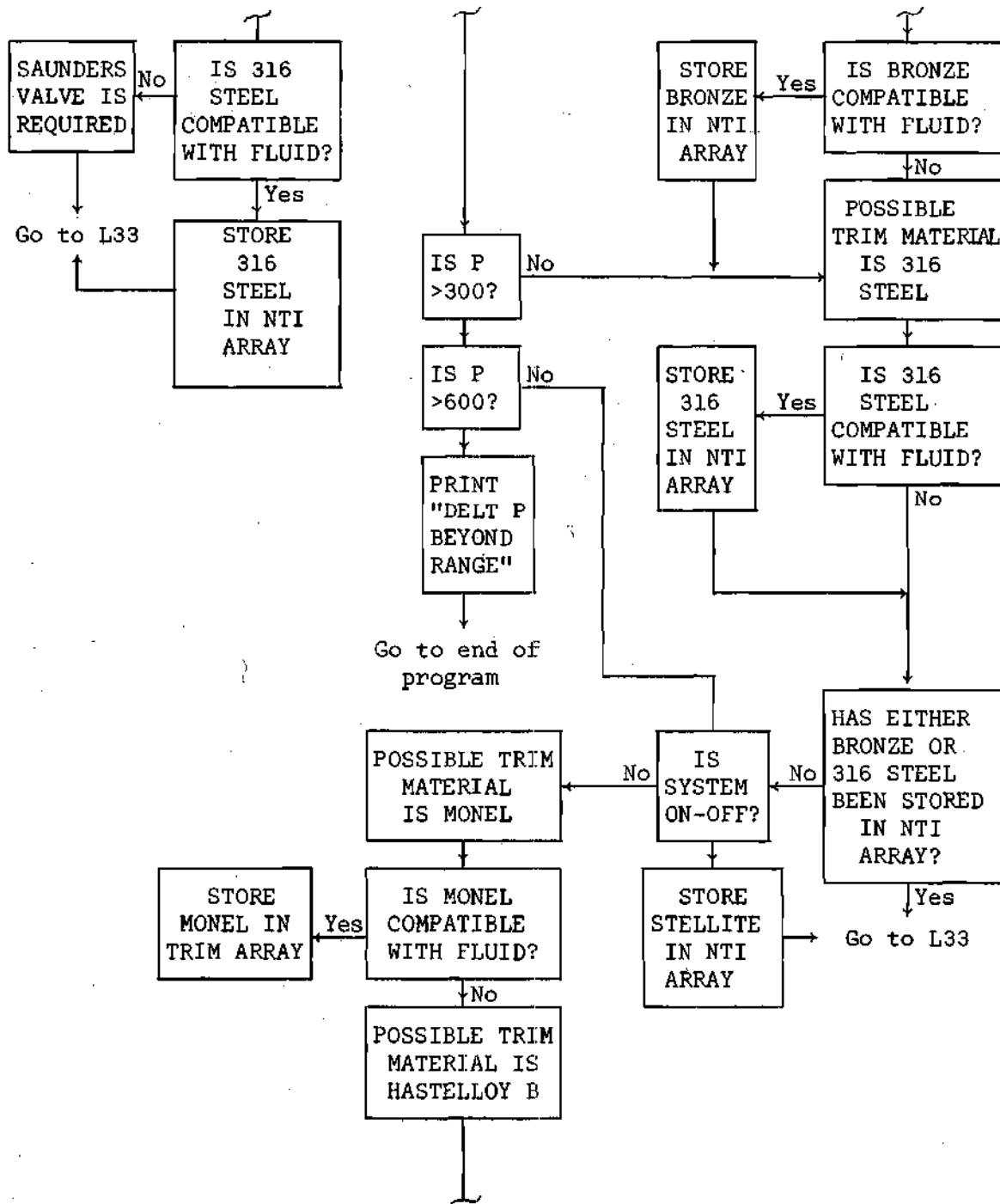
The Computer Program Flow Charts are given on the following pages. The Main Program Flow Chart starts on page 90 and the Flow Chart for the Runge-Kutta numerical integration method starts on page 103.

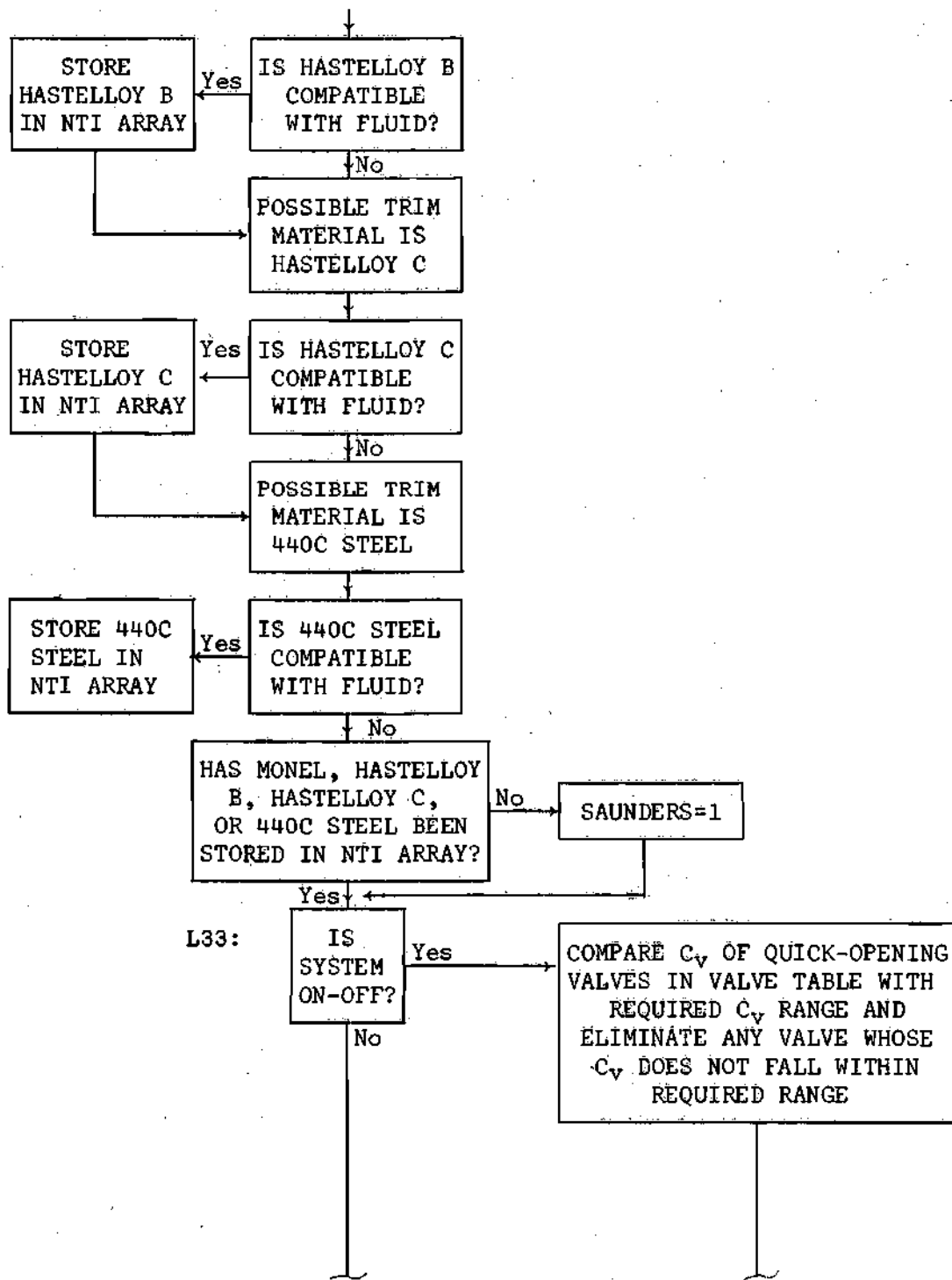


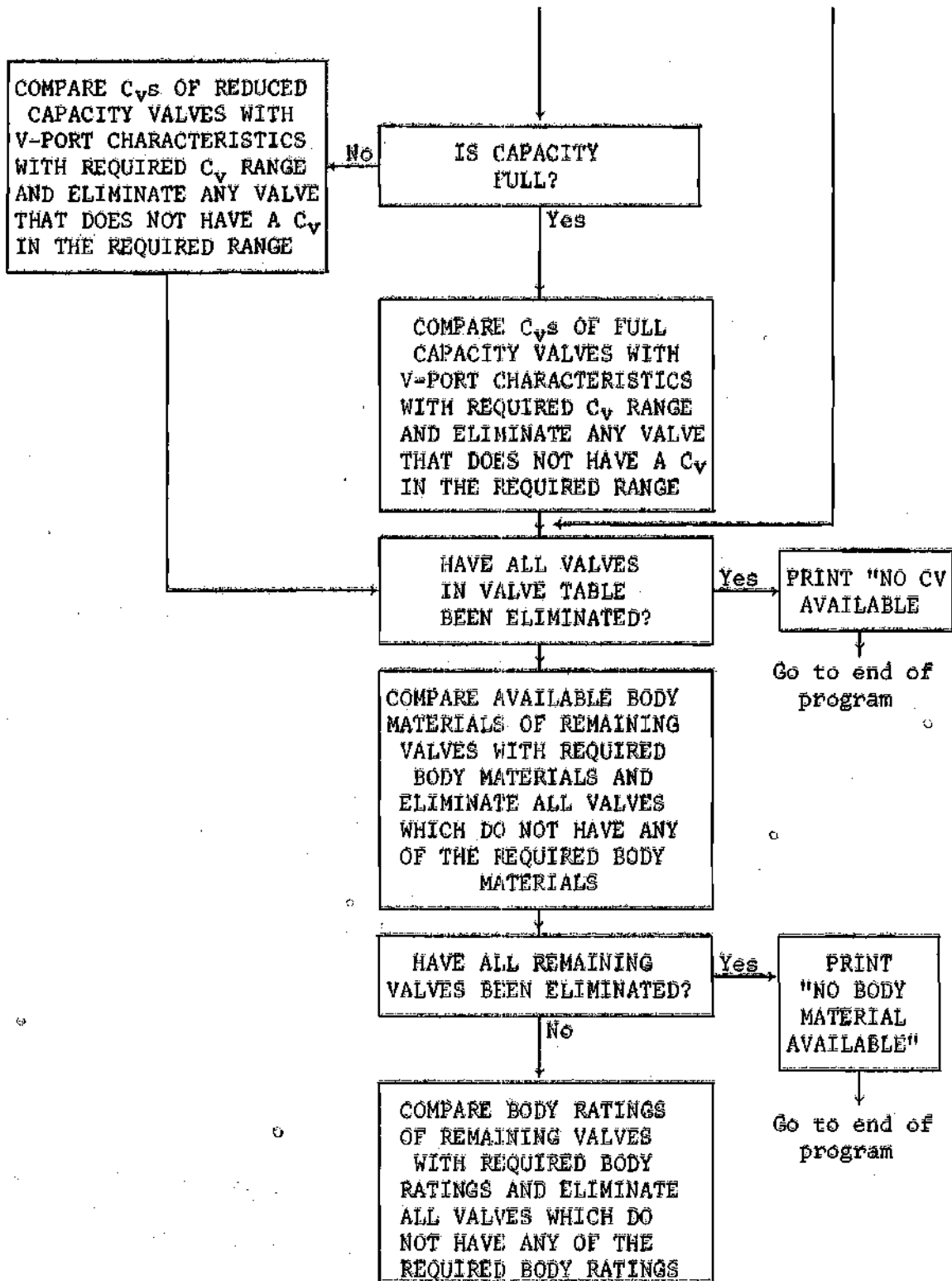


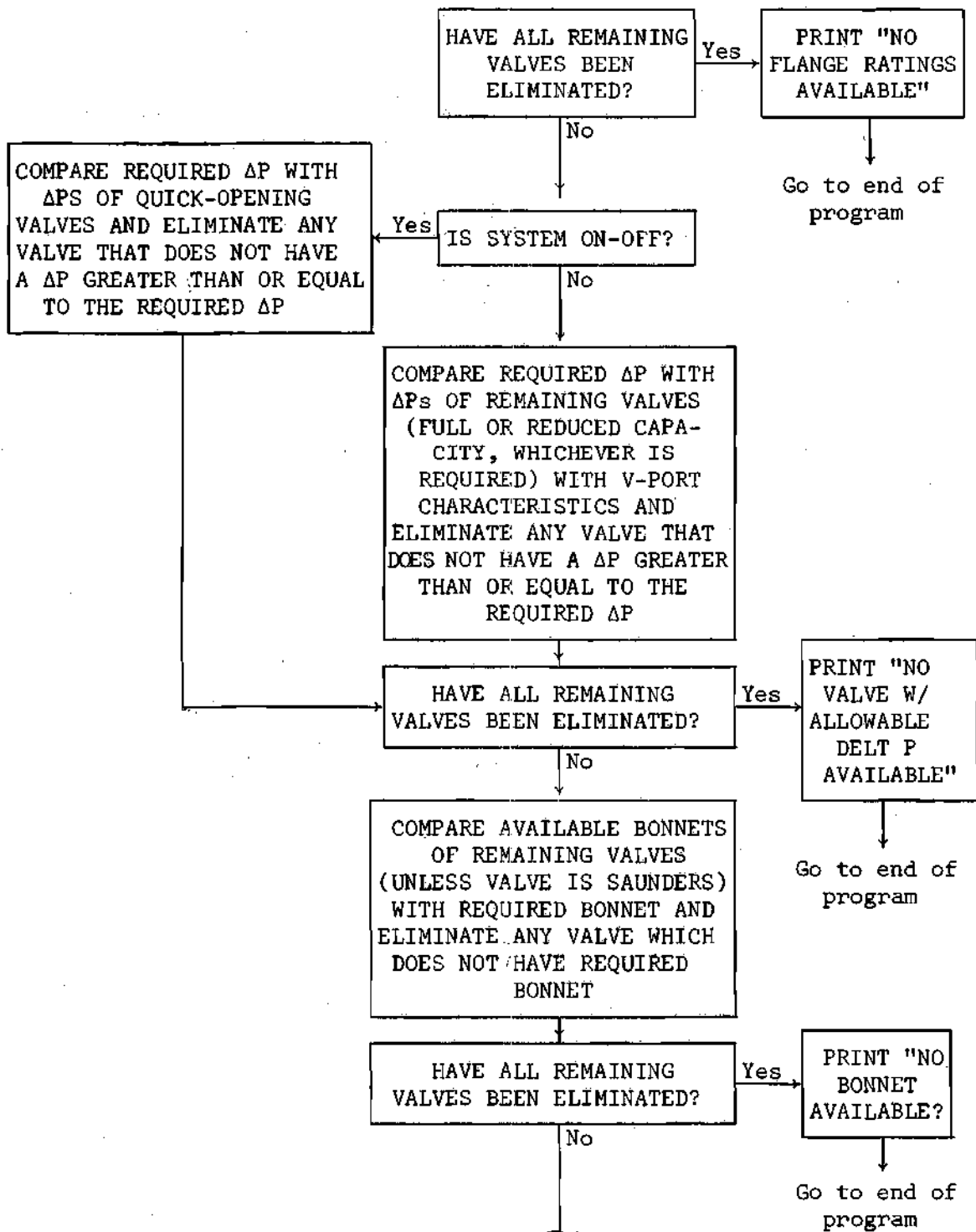


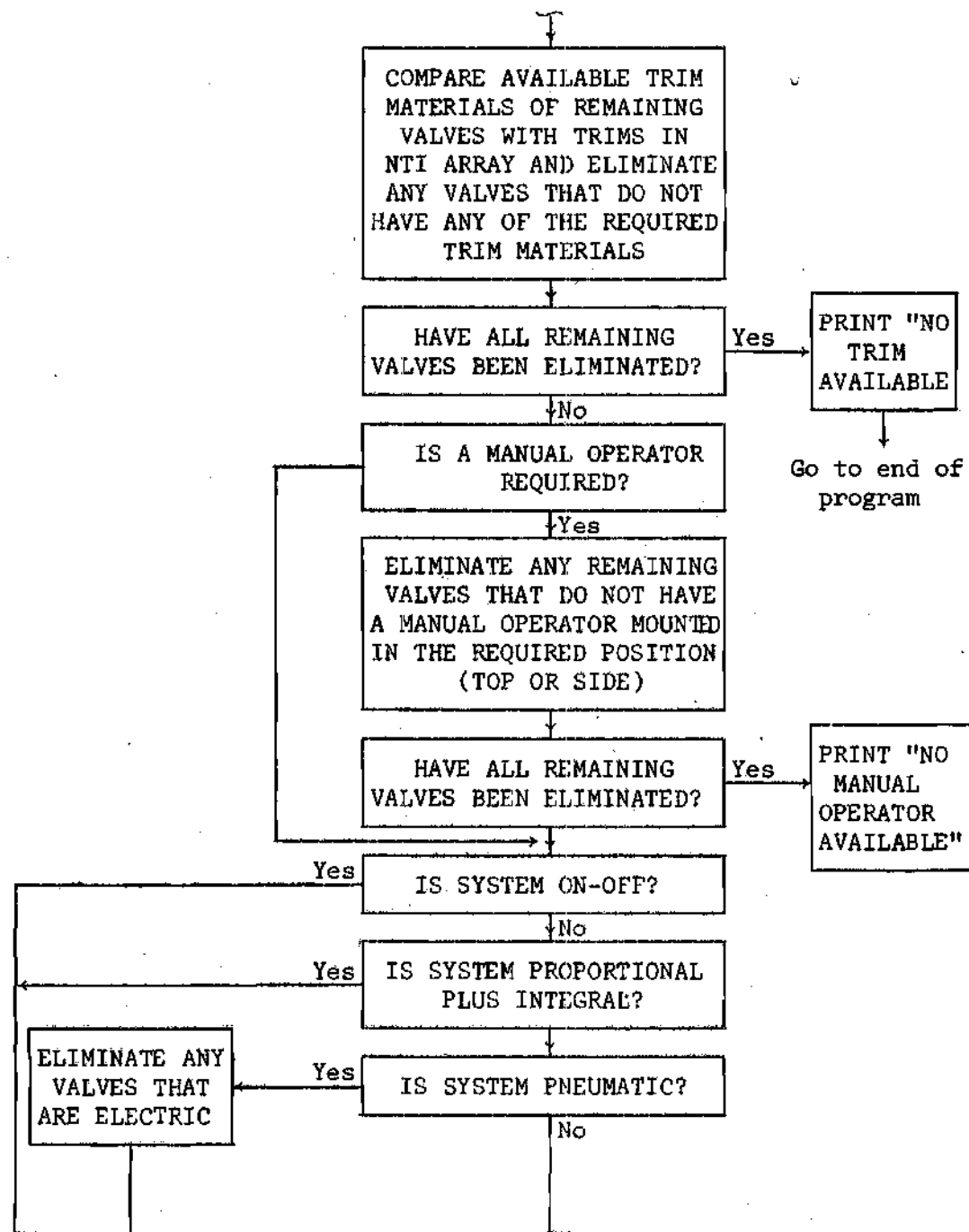


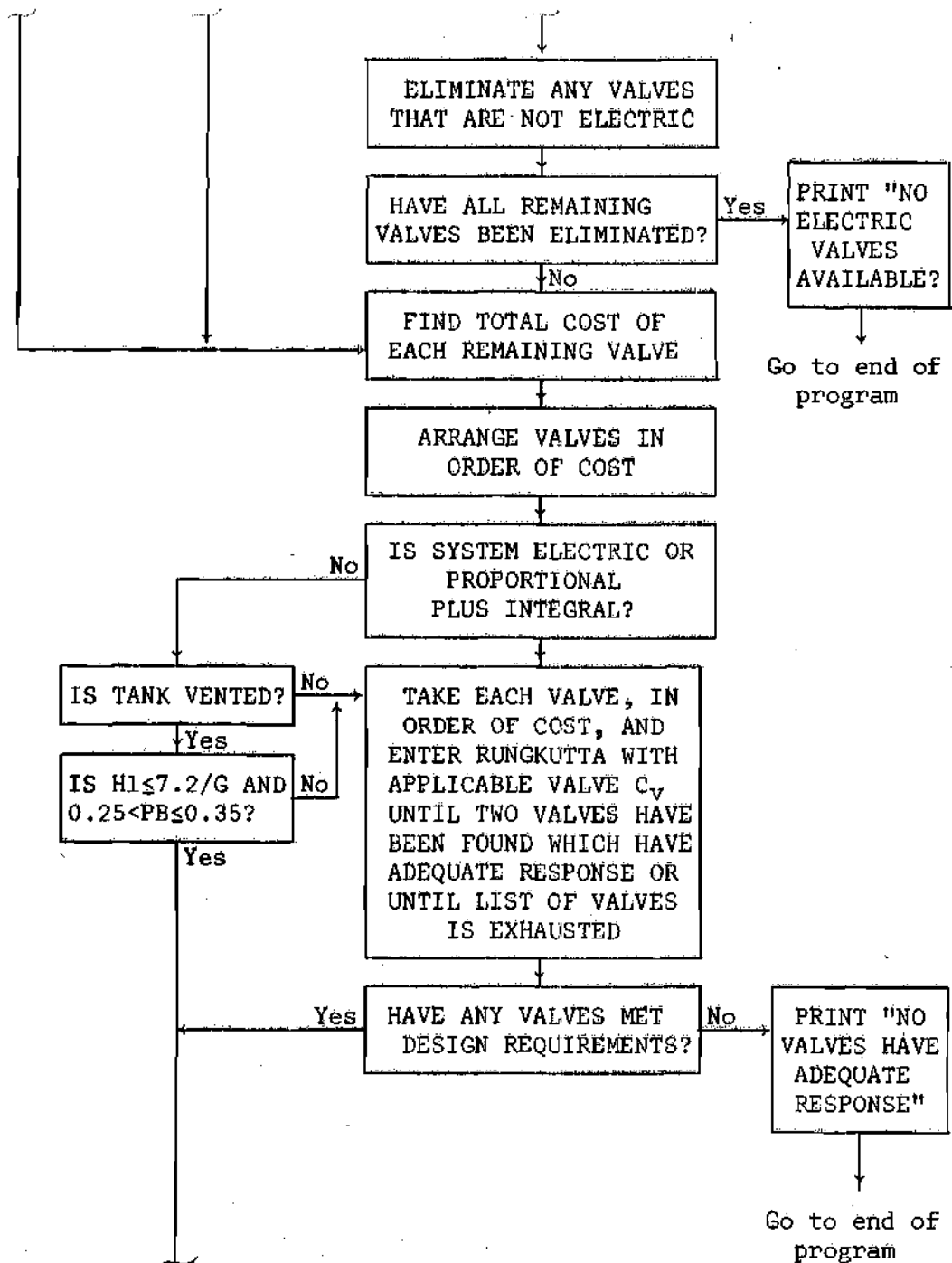


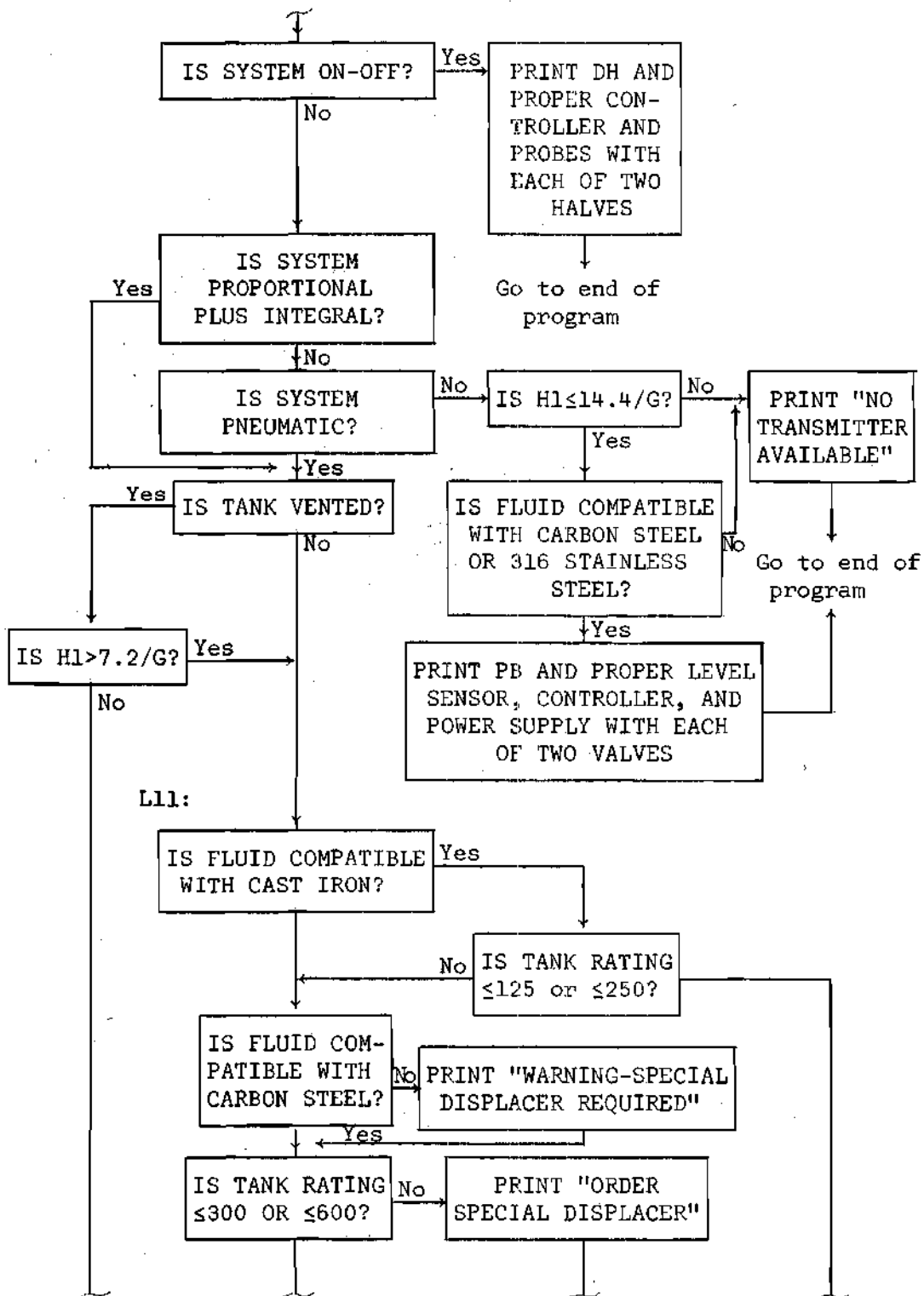


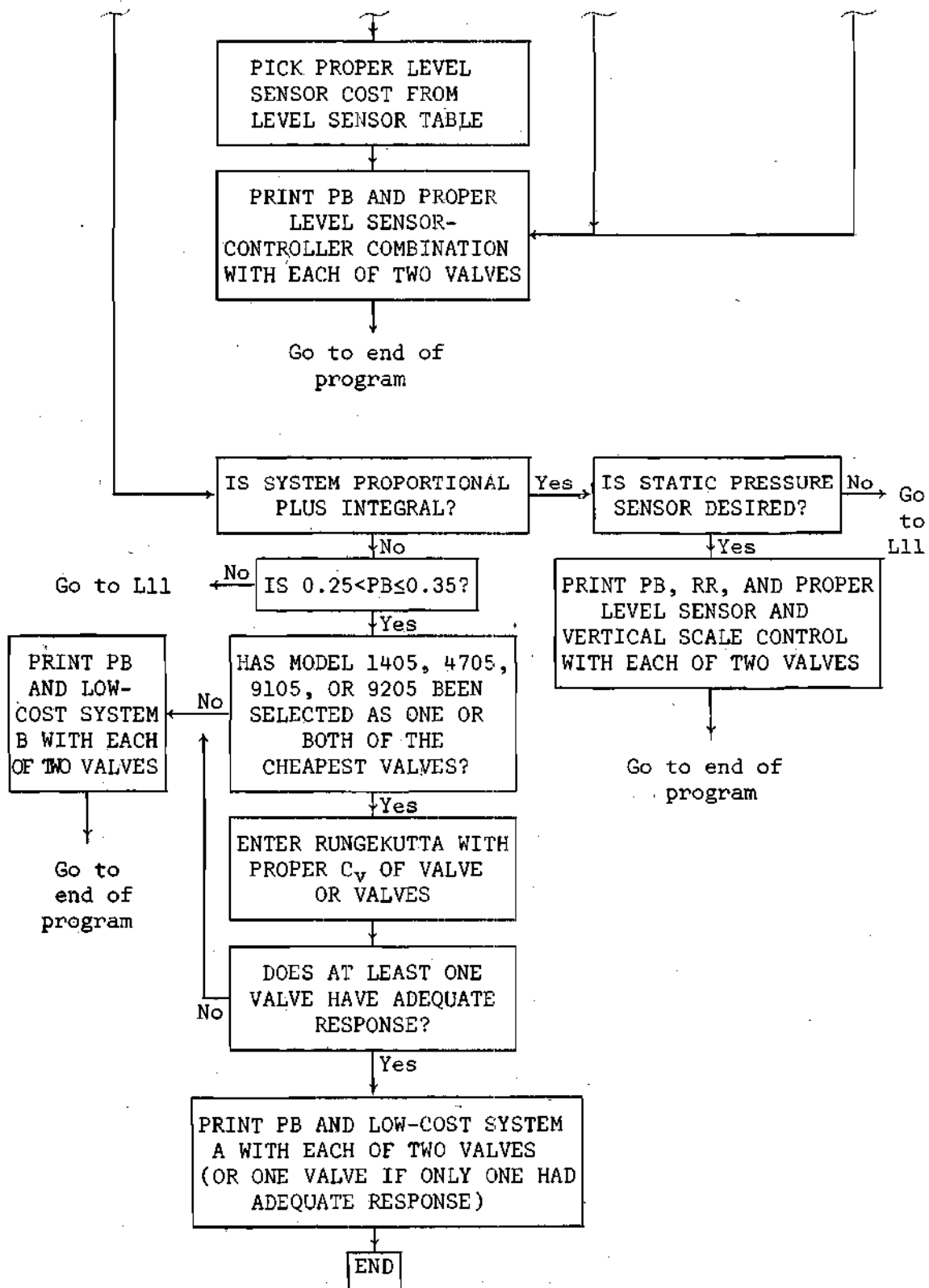


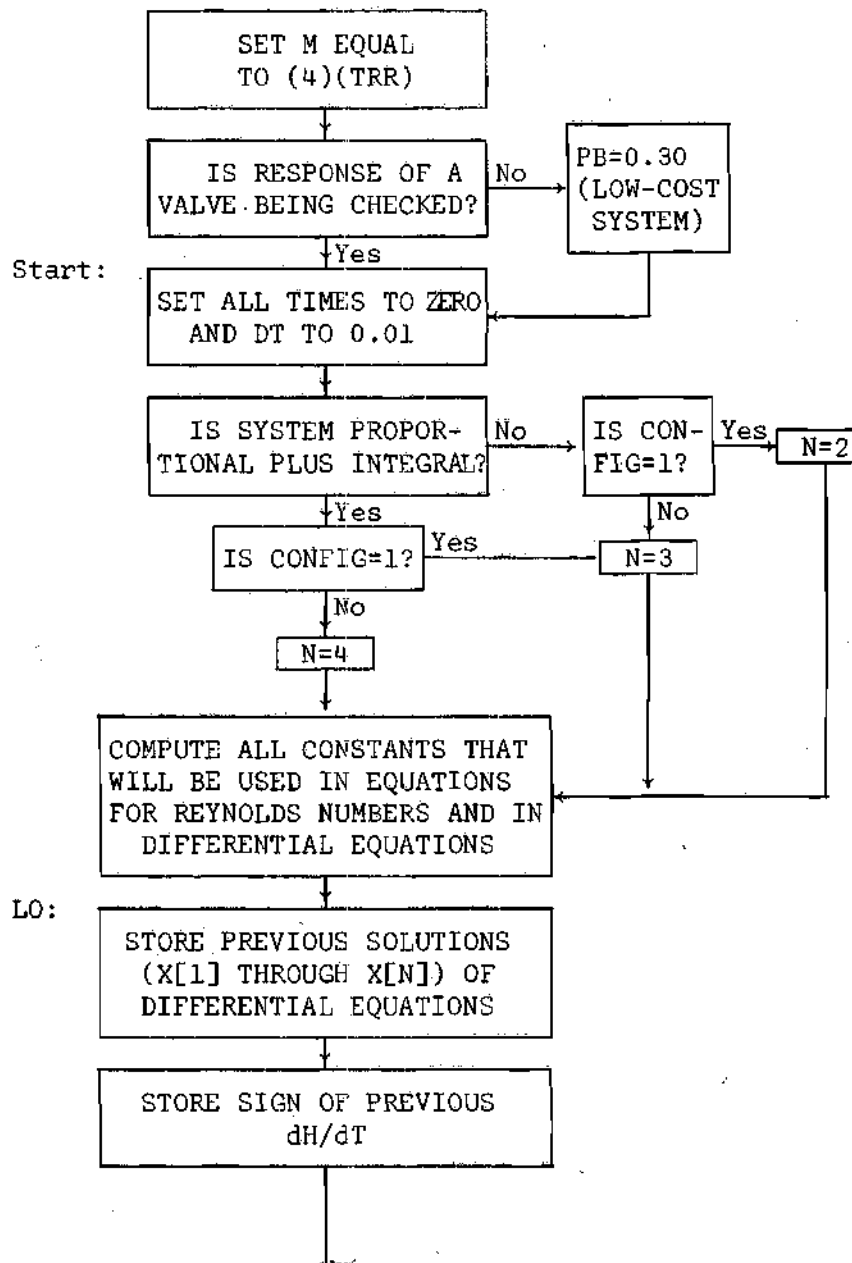




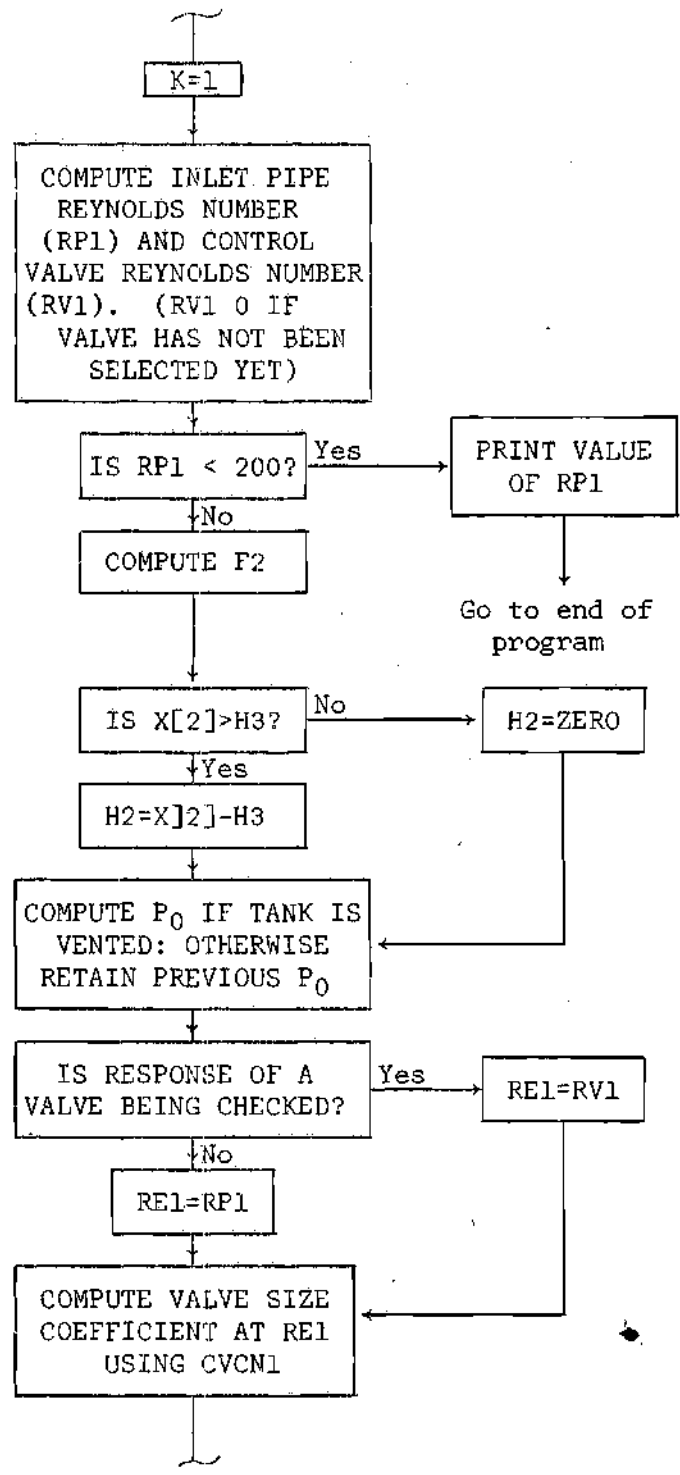


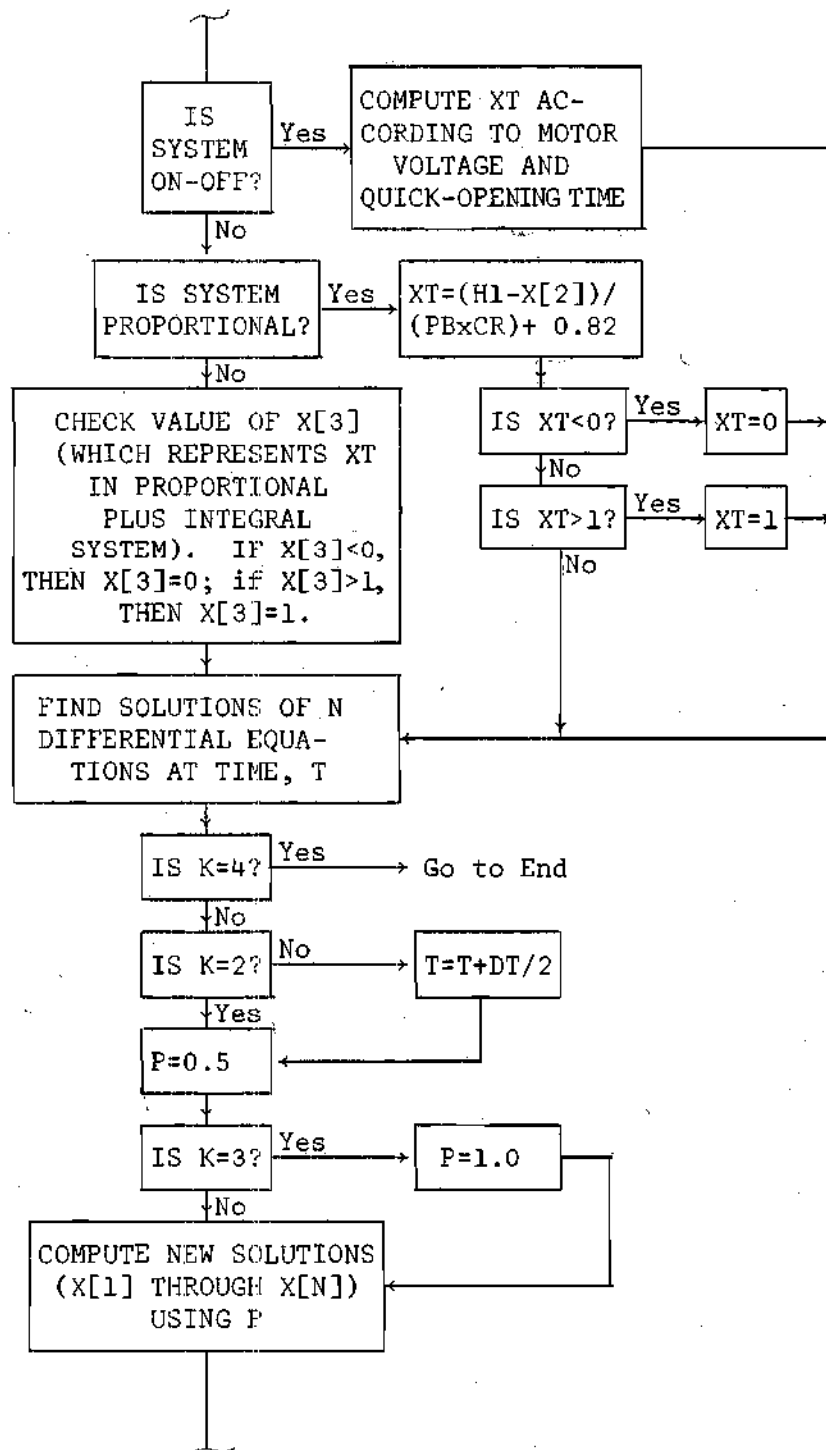


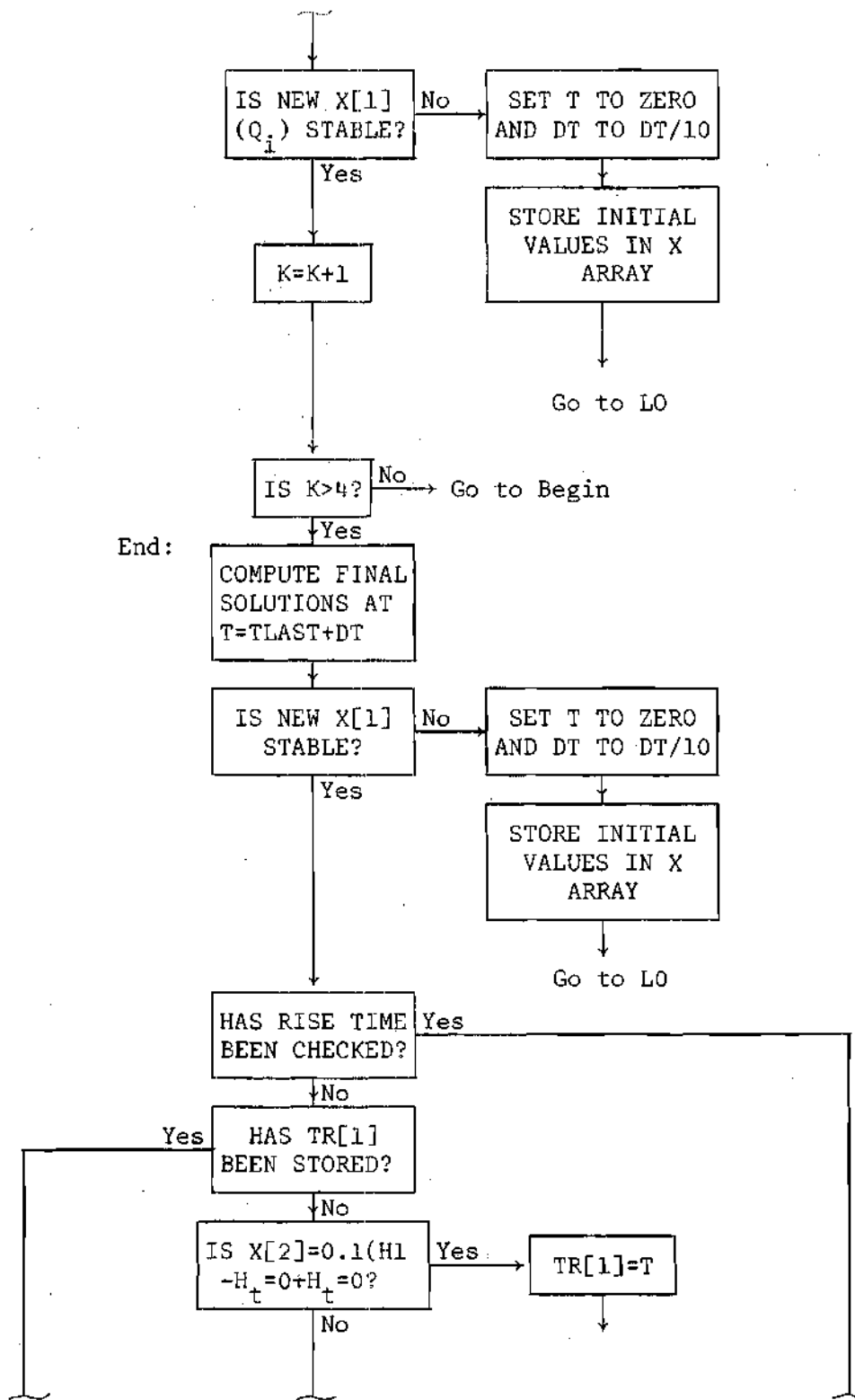


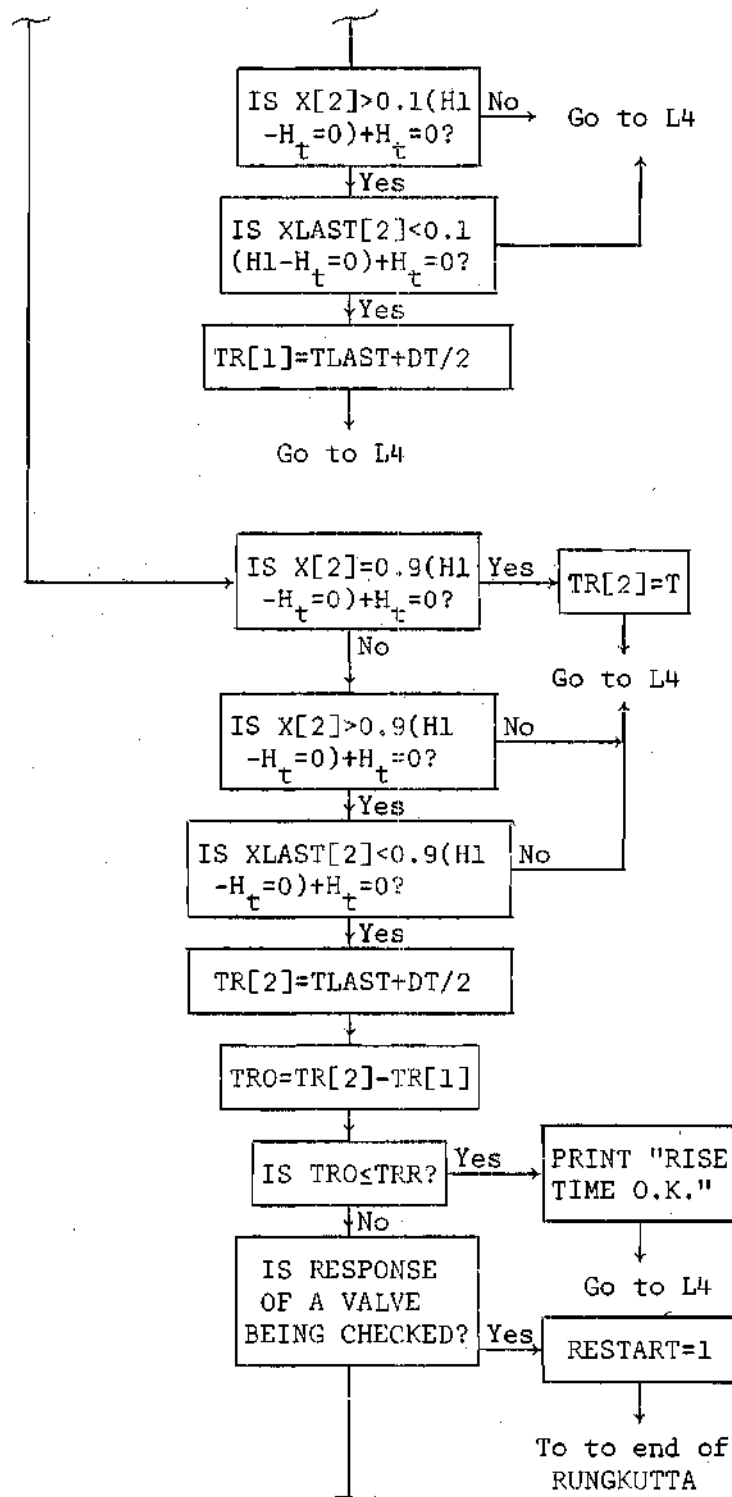


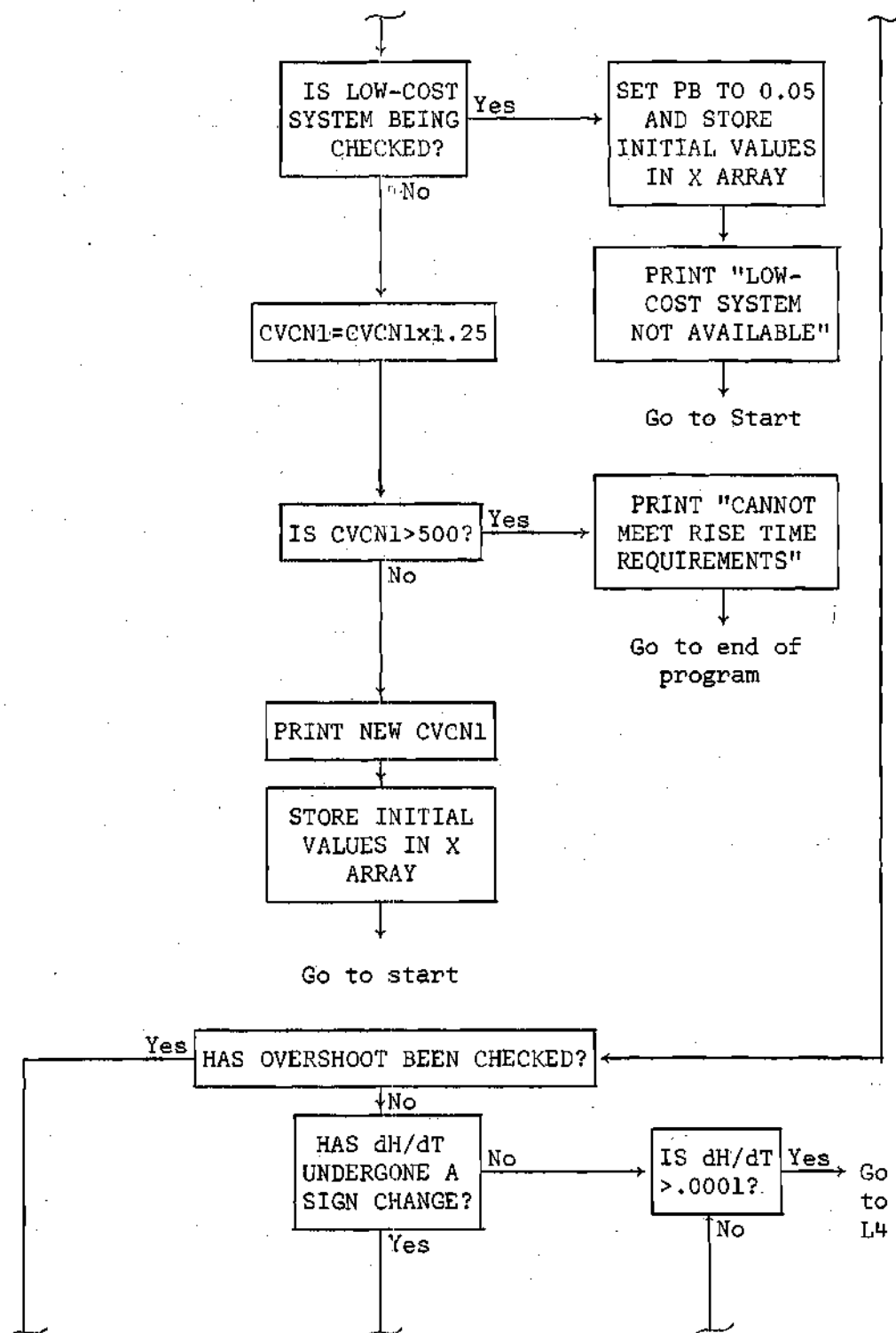
Begin:

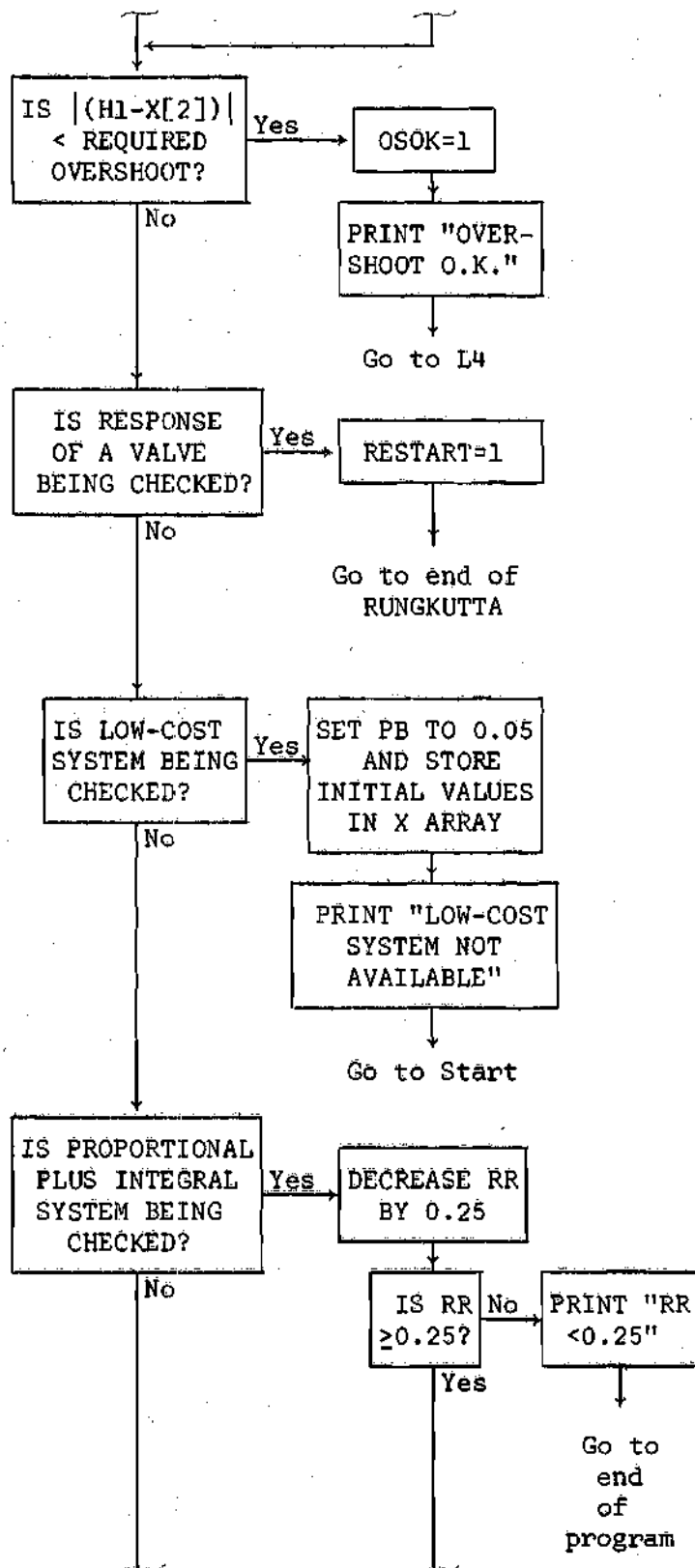


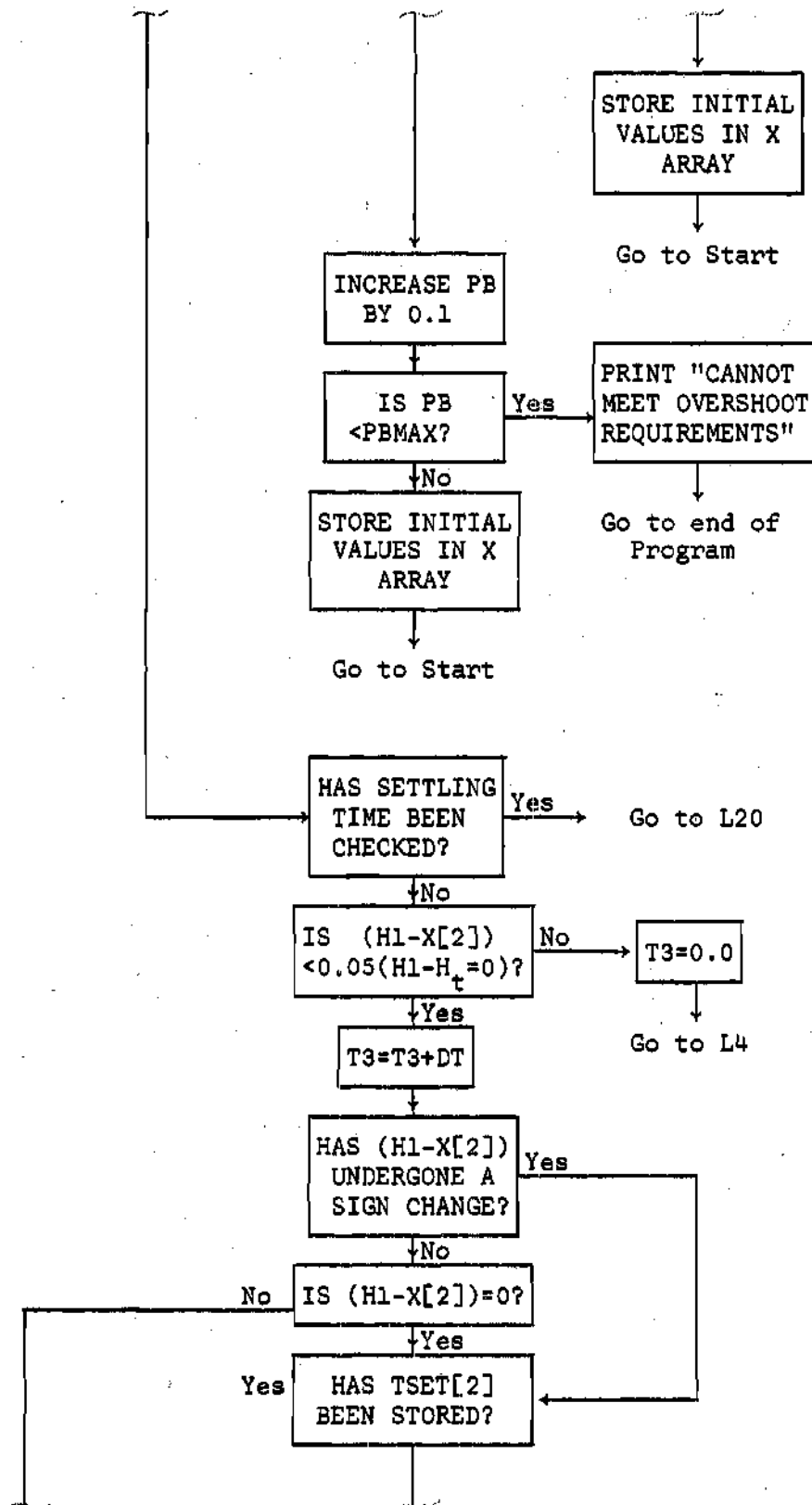


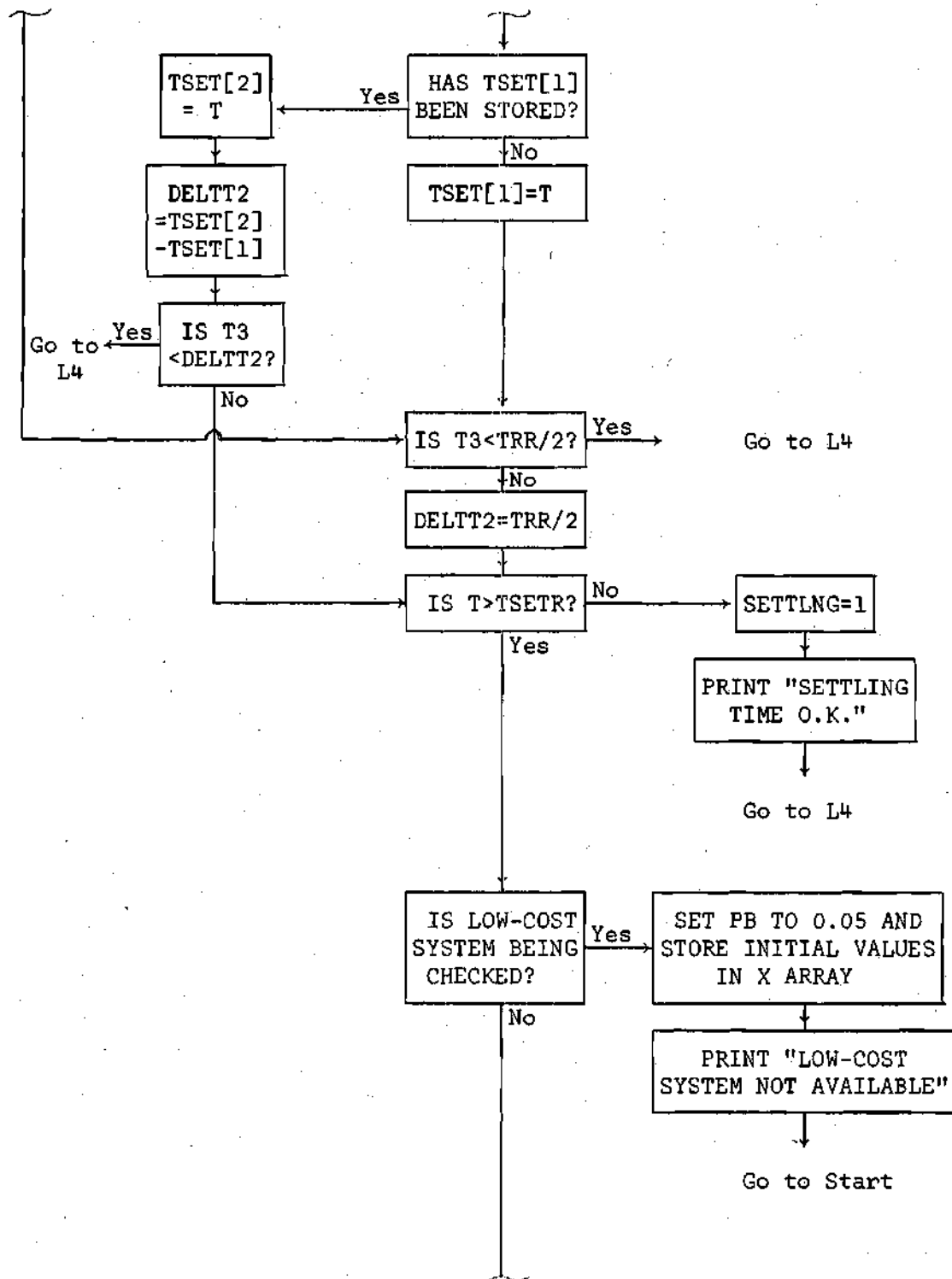


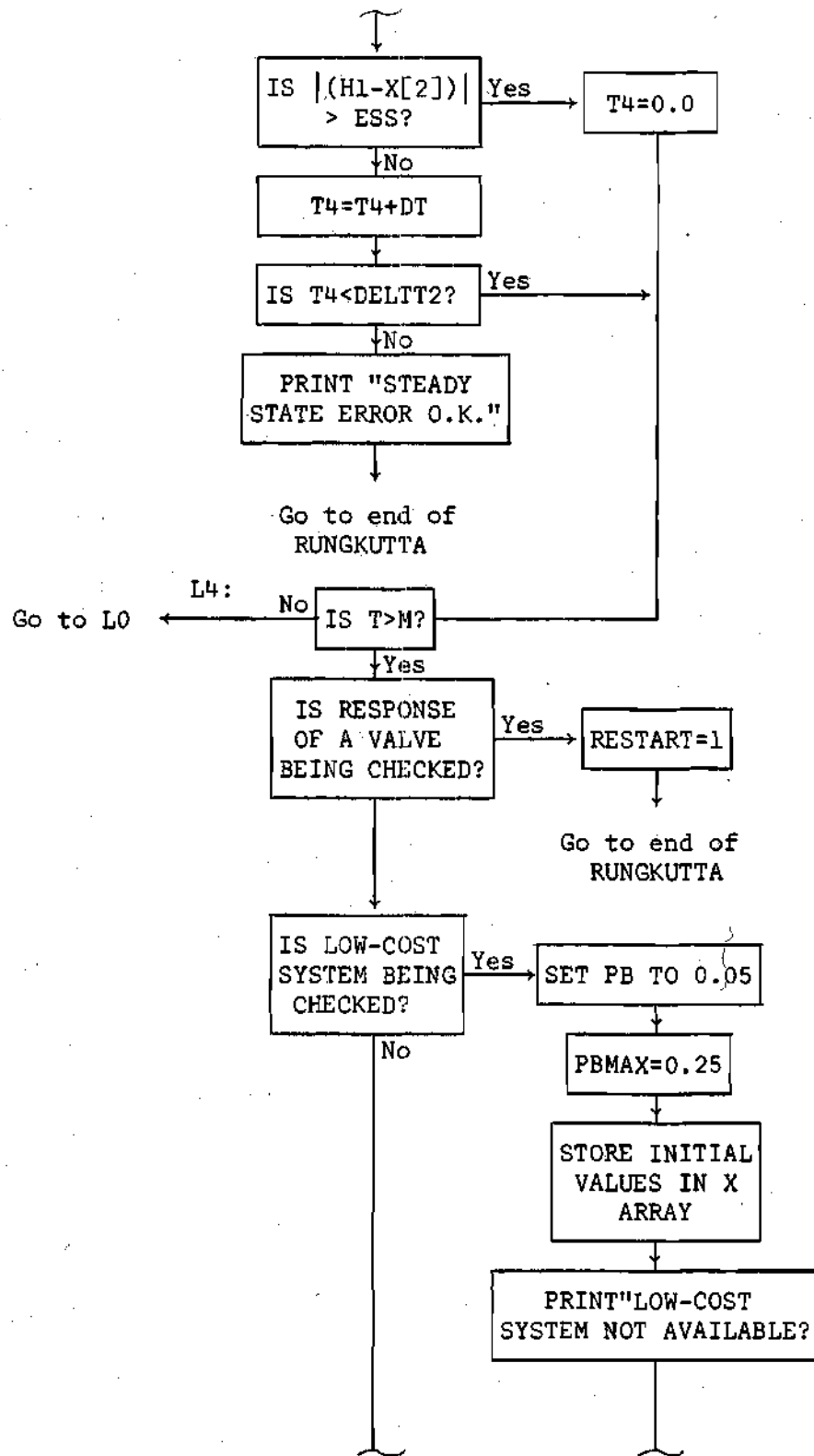


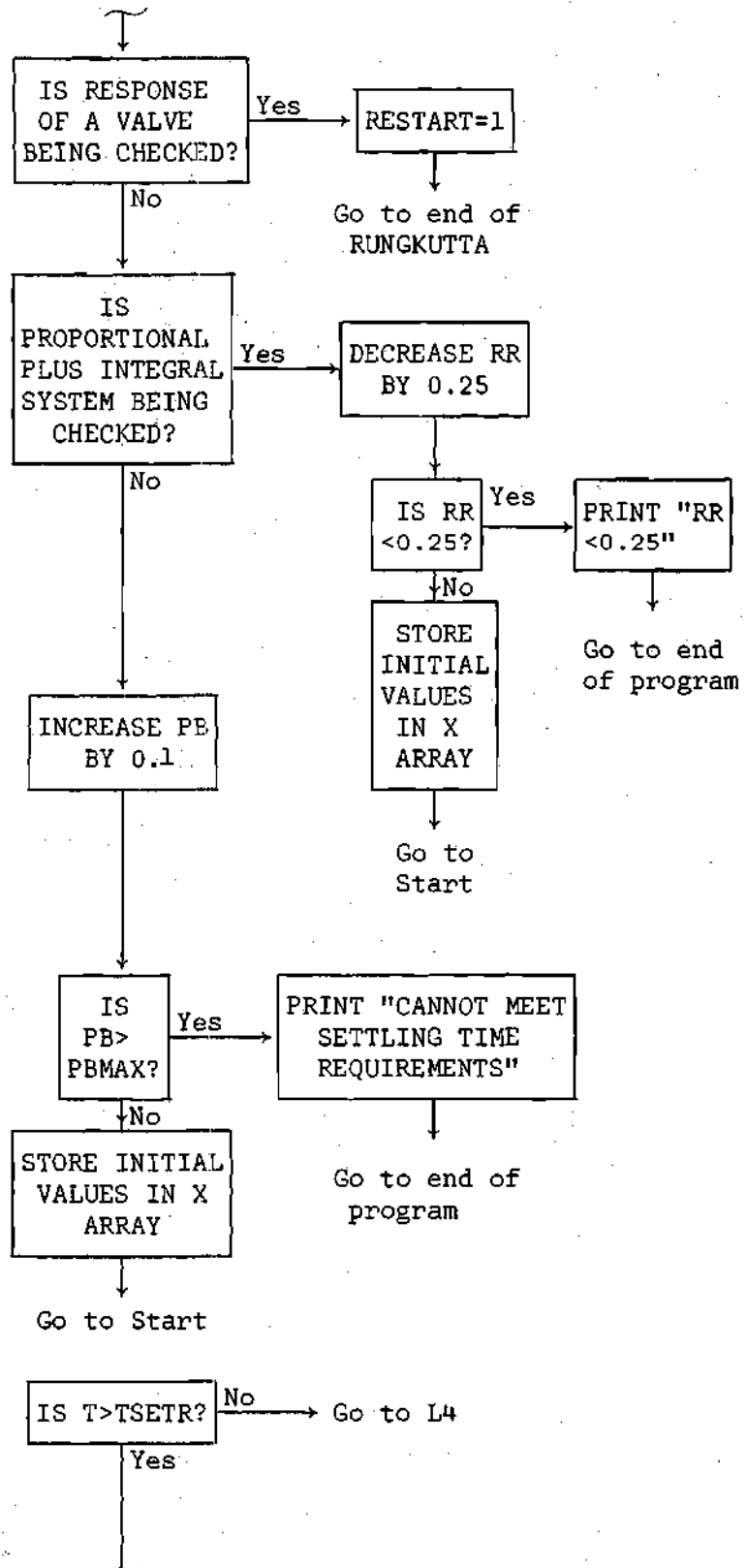


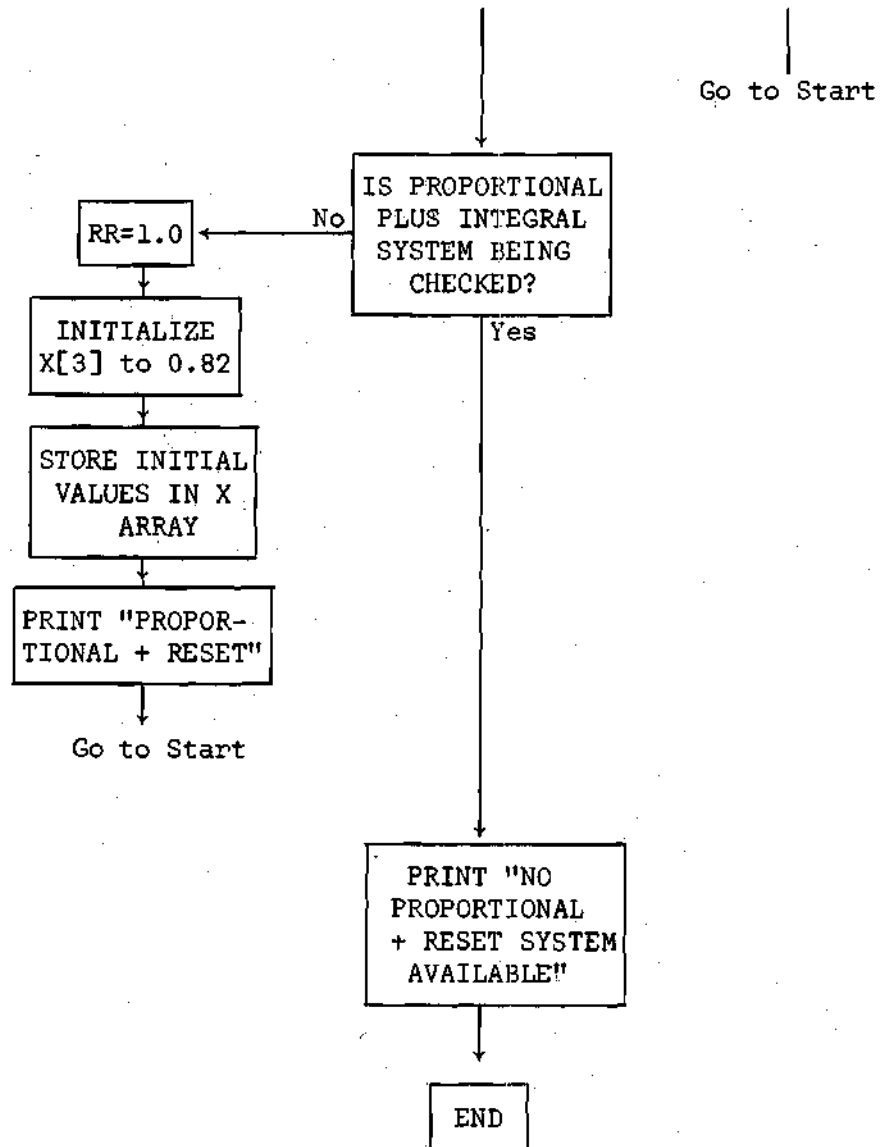












APPENDIX F

COMPLETE COMPUTER PROGRAM

The complete computer program is given on the following pages.

```

      BEGIN
      0000
      START OF SEGMENT ***** 0002
      COMMENT TEST CASES
      0000
      FILE IN CARD (1,10), TAPE1, "VALVES" (2,1023)
      0000
      FILE OUT PRINT (2,15)
      0010
      INTEGER A, I, J, K, L, M, N, VALVS, DX1, DUN, SEAL1, MONEL, EXTCOL, RADFIN,
      0015
      SLURRY, IND, CAPCTY, COMPAT, DP, TOP, MEON, J1, K1, STEEL, PLAIN,
      0015
      TANK, ONOFF, CONFIG, NEXTV, CR, SAUNDERS, PNEU, SPECIAL, STOPX,
      0015
      INITL, IFINAL, STATIC
      0015
      REAL DELTPI, P1MAX, P1MIN, T, P15, P30, P40, P60, CV1, CVMIN, CVMAX, FMIN, FMAX, Z,
      0015
      SUML, NO, O1, O3, H1, H3, QDM, P1, P0, H2, G, U, E, P13, L1, K20, P7, P8, X1M, X2M,
      0015
      CVCN1, CVCN3, CV3, RP1, F2, F4, P8, PU, FLOUT, AB, R, RR, OS, DH, TRR, TRD, DV3,
      0015
      TNKRATNG, NMRTR, ESS, P00, PBMAX, TSETR, DVT, COST, N1, N2, O6, TOTAL, COMPONENT,
      0015
      INTEGER ARRAY NB1(0:14), NTX(0:10), BC(0:10), RATNG1(0:14, 0:2), NTI(0:10),
      0015
      MODEL, VPORT, NO, VPF, VPRC, BODY, EXT, RAD, SEAL, LINVFC, LINVRC, NUMCVSE(0:100),
      0025
      ACTCOST(0:100, 0:14, 0:2), NB(0:100, 0:3), RATNB, BCOST(0:100, 0:3, 0:14),
      0030
      NBON, BNCOST(0:100, 0:10), NT, TNCOST(0:100, 0:16), AVAIL(0:1001, 0:25),
      0037
      MCOST(0:100, 0:14), NBX, RATNBX, COSTX(0:14, 0:2), WFIT(0:22), DISPL(0:3),
      0046
      DISCOST(0:19, 0:4), POSCOST(0:100, 0:2), COL(0:100), CHEAP, INDX1, INDX2,
      0055
      TMCOL, MAN, AVAILX(0:25)
      0063
      REAL ARRAY CV(0:100, 0:14), DELTP, SPRMIN, SPRMAX, AIR, ACTNO(0:100, 0:14, 0:2),
      0065
      SIZE, TYPE, VPLINK, QOLINK, TRAVL, VPTIME, QOTIME, TMIN, THAX(0:100), LT(0:5),
      0071
      LD(0:22), LE(0:5), OM(6X(0:14), X(0:14), POS(0:100, 0:2), NAME(0:2, 1:10),
      0077
      TUTCOST(0:25)
      0089
      FORMAT FL1000("INVESTIGATE POSSIBILITY OF LINED SAUNDERS VALVES"),
      0091
      START OF SEGMENT ***** 0003
      FL1001("NO BUNNET AVAILABLE"),
      0091
      FL1003("DELTP BEYOND RANGE"),
      0091
      FL1004("NO CV AVAILABLE"),
      0091
      FL1005("NO BODY MATERIAL AVAILABLE"),
      0091
      FL1006("NO FLANGE SIZE AVAILABLE"),
      0091
      FL1007("NO VALVE W/ALLOWABLE DELTP AVAILABLE"),
      0091

```

FL1008("NO BUNNET AVAILABLE"),	0091
FL1009("NO TRIM AVAILABLE"),	0091
FL1010("NO MANUAL OPERATOR AVAILABLE"),	0091
FL2000("PROGRAM TIME = ",F10.2," SECONDS"),	0091
FL2010("FLUID NOT COMPATIBLE W/MONEL OR 316 STEEL"),	0091
FL2011("FLUID NOT COMPATIBLE W/316 STEEL"),	0091
FL2012("FIRST RPI = ",E15.5),	0091
FL2013("RUNGKUTTA RPI = ",E15.5),	0091
FL2014("RUNGKUTTA RP3 = ",E15.5),	0091
FL3000("NO ELECTRIC VALVES AVAILABLE"),	0091
FL20("CVCN1 = ",F8.2),	0091
FL2((4,F7.2,A5,I5,I4,X2,A6,F8.2,X2,A6,F8.2,2F7.2,F8.4/1),	0091
FL3(F6.2,2F6.1,F6.1,X6,F5.1,X11,A6,I9/),	0091
FL4(I5,X4,2I7/),	0091
FL5(I3,X1,2I7,I8/),	0091
FL6(I6,X6,I6/),	0091
FL7(I5,X5,I5/),	0091
FL8("TOTAL COST = ",I6/),	0091
FL9(3F6.2,6F7.2,F8.2,F7.2),	0091
FL10(20I4),	0091
FL11(4F7.2,F6.2,6I2,2E9.2,F5.3,F5.2),	0091
FL12(2I13),	0091
FL13(3F6.2,F6.1,I4,F6.1,F7.1,F6.3,F6.2,I2),	0091
FL1(2I2,I4,I2,F6.1,I2,7F7.2),	0091
	0003 IS 0247 LONG, NEXT SEG 0002
PROCEDURE TAKOUT,	0091
BEGIN	0091
INTEGER I,K,TEMP,	0091
	START OF SEGMENT ***** 0004
LABEL L1,L2,	0000
I=1,	0000
TEMP=COMPAT,	0000

L1: IF OMIT(I)=1 THEN BEGIN	0001
TEMP=TEMP+1	0003
IF I>COMPAT THEN GO TO L2	0004
FOR K=1 STEP 1 UNTIL COMPAT=1 DO BEGIN	0006
AVAIL(K)+AVAIL(K+1); OMIT(K)+OMIT(K+1);	0013
COL(K)+COL(K+1); MAN(K)+MAN(K+1);	0017
END	0021
AVAIL(TEMP+1)+0; OMIT(TEMP+1)+0;	0021
COL(TEMP+1)+0; MAN(TEMP+1)+0;	0025
GO TO L1;	0028
END;	0029
I=I+1;	0029
IF I=COMPAT THEN GO TO L1;	0030
L2: COMPAT=TEMP;	0031
END YAKUJI;	0032
	0004 IS 0035 LONG. NEXT SEG 0002
PROCEDURE NTRIM;	0091
BEGIN	0091
INTEGER J;	0091
	START OF SEGMENT ***** 0005
LABEL L1,L2;	0000
FOR J=1 STEP 1 UNTIL 10 DO BEGIN	0000
IF M*TX(J) THEN GO TO L1;	0001
IND=J;	0002
M=TX(J);	0003
GO TO L2;	0004
L1: IND=0;	0005
END;	0005
L2: END NTRIM;	0008
	0005 IS 0010 LONG. NEXT SEG 0002
PROCEDURE CVSECLIMIT;	0091
VALUE LIMIT; INTEGER LIMIT;	0091

41414	0091
INTEGER K1	0091
	START OF SEGMENT ***** 0006
FOR K=J+1 STEP 1 UNTIL J=LIMIT DD	0000
READ(CAPE1,*,*,CV(I,K),*);FOR DX(=1 STEP 1 UNTIL 2 DD(DELIP1(I,K,DX1),*	0007
SPMIN(I,K,DX1),*,SPHMAX(T,K,DX1),*);AIR(I,K,DX1),*);ACTNU(I,K,DX1),*	0018
ACTCUS(I,K,DX1));	0030
END CVST	0039
	0006 IS 0091 LONG; NEXT SEG 0002
PROCEDURE PRINTZ;	0091
BEGIN	0091
FORMAT FL8(X2),*BODY MATERIAL *,2A6);	0091
	START OF SEGMENT ***** 0007
	START OF SEGMENT ***** 0008
FL9(X2),*(SCREWED END)*);	0000
FL10(X2),*(FLANGED END)*);	0000
FL11(X2),*(PROCEED WITH CAUTION)*);	0000
FL12(X2),*BODY RATING: 16," LBS,"X7,"S",F6.2);	0000
FL13(X2),*PLAIN BONNET*);	0000
FL14(X2),*BELLOWS SEAL " *,2A6," S",F6.2);	0000
FL15(X2),*EXTENSION COLUMN",X12,"S",F6.2);	0000
FL16(X2),*HAU[A]FING F(IN",X15,"S",F6.2);	0000
FL17(X2),*THIN MATERIAL ",2A6," S",F6.2);	0000
FL22(X2),*MAN. OPERATOR (SIDE)",X8,"S",F6.2);	0000
FL23(X2),*MAN. OPERATOR (TOP)",X9,"S",F6.2);	0000
FL24(X2),*POSITIONER (SIDE) ",A6,X3,"S",F6.2);	0000
FL25(X2),*POSITIONER (TOP) ",A6,X4,"S",F6.2);	0000
FL20(X3),*TOTAL COST S",F6.2);	0000
	0008 IS 0139 LONG; NEXT SEG 0007
LABEL L3,L4,L5,L6,L7,L8,L9,L10;	0000
FOR I=1 STEP 1 UNTIL COMPAT DD IF J=AVAIL(1) THEN BEGIN	0000
J(I)=I(1); K1=(N(X21)); END;	0002

COST+BCUST(J,J1,K11) TOTAL+TOTAL+COST1	0006
I+NB(J,J1)	0010
WRITE(PRINT,FL4,NAME(1,1),NAME(2,1))	0012
FOR I+1 STEP 1 UNTIL 10 DO IF NB(J,J1)=ABS(NB1(I)) THEN GO TO L31	0025
L31 IF NB1(I)<0 THEN WRITE(PRINT,FL11)	0031
WRITE(PRINT,FL12,ABS(RATNG(J,J1,K11),COST))	0036
IF RATNG(J,J1,K11)<0 THEN WRITE(PRINT,FL9) ELSE WRITE(PRINT,FL10)	0048
IF PLAIN=1 THEN BEGIN WRITE(PRINT,FL13) GO TO L41 END	0058
IF SEAL=0 THEN GO TO L51	0064
IF STEEL=0 THEN BEGIN FOR I+4,5,6 DO IF NB(J,J1)=NBONE(I) THEN M+1	0065
END ELSE BEGIN FOR I+7,8,9 DO IF NB(J,J1)=NBON(I) THEN M+1 END	0077
COST+BUNCUST(J,M) TOTAL+TOTAL+COST	0089
WRITE(PRINT,FL14,IF STEEL=0 THEN NAME(1,5) ELSE NAME(1,4),	0092
IF STEEL=0 THEN NAME(2,5) ELSE NAME(2,4),COST)	0103
GO TO L41	0114
L51 FOR I+1 STEP 1 UNTIL 3 DO IF NB(J,J1)=NBONE(I) THEN	0114
COST+BUNCUST(J,I) TOTAL+TOTAL+COST	0118
IF EXTCOL=1 THEN WRITE(PRINT,FL15,COST)	0124
IF RADFIN=1 THEN WRITE(PRINT,FL16,COST)	0134
L41 FOR I+1 STEP 1 UNTIL COMPAT DO IF J=AVAIL(I) THEN M+MCOL(I)	0143
I+NT(J,M)	0149
COST+MFCOST(J,M) TOTAL+TOTAL+COST	0151
WRITE(PRINT,FL17,NAME(1,I),NAME(2,I),COST)	0154
FOR I+1 STEP 1 UNTIL 10 DO IF NT(J,M)=ABS(NT1(I)) THEN GO TO L61	0168
L61 IF NT1(I)<0 THEN WRITE(PRINT,FL11)	0174
IF OPP=1 THEN GO TO L71	0179
FOR I+1 STEP 1 UNTIL COMPAT DO IF AVAIL(I)=J THEN M+MAN(I)	0180
COST+MCOST(J,M) TOTAL+TOTAL+COST	0185
IF TOP=1 THEN WRITE(PRINT,FL22,COST) ELSE WRITE(PRINT,FL23,COST)	0188
L71 IF UNOFF=1 THEN GO TO L101	0207
IF POSCUST(J,2)=0 THEN GO TO L91	0209
COST+POSCUST(J,2)	0211

```

DX1=21 0213
GO TO L91 0214
L81 IF PUSCOST(J,1)50 THEN GO TO L101 0214
COST=PUSCOST(J,1) 0217
DX1=1 0219
L91 TOTAL=TOTAL+COST 0219
IF DX1=2 THEN WRITE(PRINT,FL24,PUSC(J,DX1),COST) ELSE 0221
WRITE(PRINT,FL25,PUSC(J,DX1),COST) 0233
L101 WRITE(PRINT,FL20,TOTAL) 0245
END PRINT 0254

0007 IS 0257 LONG, NEXT SEG 0002

PROCEDURE HUNGKUTTA(DIAP1,DIAP3,DIAV1,DIAV3,ROTIN,RESTART) 0091
VALUE DIAP1,DIAP3,DIAV1,DIAV3,ROTIN 0091
REAL DIAP1,DIAP3,DIAV1,DIAV3,ROTIN 0091
INTEGER RESTART 0091
BEGIN 0091
INTEGER I,K,V1,VLAST,M,CHANGE,USOK,SETTLNG 0091

START OF SEGMENT ***** 0009

REAL T,DT,T1,FAC0,FAC1,FAC10,FAC2,FAC20,FAC3,FAC30,FAC300,FAC4,FAC40, 0000
FAC5,FAC50,FAC6,FAC33,TLAST,H1X2,RP3,RV3,P,RP1,RV1,XT,T3,T4,TSETD, 0000
DEL1,T,RE1,M,DIFF 0000
ARRAY T(10:21),CX(0:2),XLAST(0:4),DT(0:4,0:4),TSET(0:2) 0000
FORMAT FMT(6E20,5) 0010

START OF SEGMENT ***** 0010
0010 IS 0004 LONG, NEXT SEG 0009

FORMAT FL1("RISE TIME 0,K.") 0010

START OF SEGMENT ***** 0011
FL2("NEW CV = "E15.5) 0010
FL3("CANNOT MEET RISE TIME REQUIREMENTS") 0010
FL4("OVERSHOOT 0,K.") 0010
FL5("CANNOT MEET OVERSHOOT REQUIREMENTS") 0010
FL6("CANNOT MEET SETTLING TIME REQUIREMENTS") 0010

```

```

FL7("SETTLING TIME U.K."), 0010
FL8("STEADY STATE ERROR U.K."), 0010
FL9("CANNOT MEET OVERTHOOT REQUIREMENTS(RR<0.25)"), 0010
FL10("CANNOT MEET SETTLING TIME REQUIREMENTS(RR<0.25)"), 0010
FL12("PROPORTIONAL+RESET"), 0010
FL13("LOW-COST SYSTEM NOT AVAILABLE"), 0010
FL14("/X14,"T",X17,"X(1)",X14,"X(2)",X16,"X(3)",X17,"X(T)", 0010
FL15("/X14,"T",X17,"X(1)",X14,"X(2)",X16,"X(3)",X17,"X(R)", 0010
FL11("NO PROPORTIONAL+RESET SYSTEM AVAILABLE") 0010

                                0011 IS 0133 LONG. NEXT SEG 0009

LABEL L0,L10,L31,L4,L5,L6,L7,808,START,L2,HAX,L11,L12,L13,L14, 0010
L8,L9,L15,L16,L17,L18,L19,L20,L21,L22) 0010
DIFF=H1-ORIGX(2) 0010
N=TRR*4) 0011
IF RESTART#0 THEN PH=0.30) 0013
CX(1)+0.1) CX(2)+0.90) 0015
IF RR#0.0 THEN WRITE(PRINT,FL14) ELSE WRITE(PRINT,FL15)) 0017
START: V1=1) 0029
VLAST=0) 0030
T3=0.0) 0031
OSUR=0) 0032
SETTLNG=0) 0033
T=0.0) 0033
DT=0.1) 0034
FOR I=1,2 DO BEGIN TR(I)+0.0) TSET(I)+0.0) END) 0035
IF RR#0.0 THEN BEGIN 0044
ORIGX(3)+X(3)+X(4) 0045
IF CONF(0)=1 THEN N=2 ELSE N=3) 0047
GO TO L2) 0051
END) 0051
IF CONF(0)=1 THEN N=3 ELSE N=4) 0051
L2: FACD=PH*CN) 0054

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FAC1+303*DIAP1*DIAP1/(LT(2)+LT(3))	0056
FAC2+(LE(2)+LE(3))/DIAP1*5	0059
FAC3+3.30*3*G/(DIAP1*U)	0063
IF DIAV1#0 THEN FAC33+3.30*3*G/(DIAV1*U)	0065
FAC4+1.9404*DIAP1*U/G	0069
FAC5+2*0*E/DIAP1	0071
FAC6+1/A0	0073
IF CONFIG=2 THEN BEGIN	0074
FAC10+303*DIAP3*DIAP3/(LI(4)+LI(5))	0075
FAC20+(LE(4)+LE(5))/DIAP3*5	0078
FAC30+3.30*3*G/(DIAP3*U)	0082
FAC300+3.30*3*G/(DIAP3*U)	0084
FAC40+1.9404*DIAP3*U/G	0087
FAC50+2*0*E/DIAP3	0089
END	0091
LO: FOR I=1 STEP 1 UNTIL 4 DO XLAST((I)+X(I))	0091
FLAST=I	0098
CHANGE+SIGN(DI2*4)	0099
FOR K=1 STEP 1 UNTIL 4 DO BEGIN	0103
RP1+FAC3*X(I)	0104
RV1+FAC33*X(I)	0105
IF RP1<200 THEN BEGIN WRITE(PRINT,FL2013,RP1) STOPX+1 GO TO L6 END	0107
IF RP1<2000 THEN F2+FAC4/X(I) ELSE F2+5.50*3*(1+(FAC5+1*6/RP1)+0.3333)	0117
IF X(2)>H3 THEN H2*X(2)=H3 ELSE H2+0.01	0126
IF TANK=3 THEN PD+PD*X(1)/(L1-X(2))	0134
IF DIAV1#0.0 THEN RE1+RV1 ELSE RE1+RP1	0138
IF RE1<1000 THEN CV1+CVCN1*(0.707+0.11*(LN(RE1)-5.3))	0141
ELSE IF RE1<5000 THEN CV1+CVCN1*(0.800+0.0398*(LN(RE1)-6.91))	0145
ELSE IF RE1<100000 THEN CV1+CVCN1*(0.948+0.01739*(LN(RE1)-8.52))	0154
ELSE CV1+CVCN1	0165
HIX2+H1*X(2)	0173
IF (HUFF=1) THEN BEGIN	0175

IF H1X2>=0H AND H1X2<0H THEN BEGIN	0176
IF VLAST=0 THEN BEGIN XT=0.03 GO TO B0B3 END3	0179
IF K=2 OR K=3 THEN T1=T1+DT/23	0181
IF T1<=0T1M THEN XT=T1/0DT1M ELSE XT=1.03	0185
GO TO B0B3	0189
END3	0191
IF H1X2<0H THEN BEGIN	0191
IF VLAST=0 THEN T1=0.0 ELSE BEGIN IF K=2 OR K=3 THEN T1=T1+DT/23 END3	0192
VLAST=V13	0198
IF T1<=0T1M THEN XT=T1/0DT1M ELSE XT=1.03	0199
GO TO B0B3	0203
END3	0205
VLAST=03	0205
XT=0.03	0205
GO TO B0B3	0206
END3	0207
IF RR=0.0 THEN BEGIN	0207
XT=H1X2/FAC0+0.023	0208
IF XT<0.0 THEN XT=0.03	0210
IF XT>1.0 THEN XT=1.03	0212
GO TO B0B3	0214
END3	0217
IF X(3)<0.0 THEN X(3)=0.03	0217
IF X(3)>1.0 THEN X(3)=1.03	0219
$0(1,K) = (FAC1 \times ((P1 - P0) / G - H2) = (27.0 / (R + (2 \times (X(3) - 1)) \times CV1 + 2) + 2.1 \times 3 \times F2$	0222
$\times FAC2) \times X(1) \times X(1)) \times DT3$	0231
IF N=3 THEN D(2,K) = FAC0K(X(1) - FLBUT) \times DT ELSE D(2,K) = FAC0K(X(1) - X(4)) \times DT3	0235
$DT3 = K1 + (RR \times H1X2 - D(2,K)) / FAC0 \times DT3$	0247
GO TO MAX2	0253
B0B3: IF ONOFF=1 THEN IF XT=0.0 THEN X(1) = URIGX(1)3	0253
$0(1,K) = (FAC1 \times ((P1 - P0) / G - H2) = (27.0 / (R + (2 \times (X(1) - 1)) \times CV1 + CV1) + 2.1 \times 3 \times F2$	0258
$\times FAC2) \times X(1) \times X(1)) \times DT3$	0266

IF N=2 THEN D(2,K)+FAC6*(X(1)-F(007))*DT	0270
ELSE D(2,K)+FAC6*(X(1)-X(3))*DT	0274
BACK: IF CUNFIG=2 THEN BEGIN	0282
RP3+FAC30*X(N)	0283
RV3+FAC300*X(N)	0284
IF RP3<200 THEN BEGIN WRITE(PRINT,FL2014,RP3) GO TO L63 END	0286
IF RP3>2000 THEN F4+FAC40/X(N)	0295
ELSE F4+5.50-3*(1+(FAC50+1#6/KP3)+0.33333)	0297
IF RV3<1000 THEN CV3+CVCN3*(0.707+0.11*(LN(RV3)-5.3))	0304
ELSE IF RV3<5000 THEN CV3+CVCN3*(0.884+0.0398*(LN(RV3)-6.91))	0308
ELSE IF RV3<100000 THEN CV3+CVCN3*(0.948+0.01734*(LN(RV3)-8.52))	0320
ELSE CV3+CVCN3	0331
DIN,K)+(FAC10*((P0-P7)/G+X(2))-(27.6/(K20*CV3)+2+2.18-3*F4+FAC20)*X(N)	0339
KX(N))*DT	0346
END	0349
IF K=0 THEN GO TO L31 ELSE IF K=2 THEN GO TO L101	0349
T+0.5*DT	0352
L101 F+0.5	0353
IF K=3 THEN P+1.01	0354
FOR I=1 STEP 1 UNTIL N DO X(I)+X(I)+D(1,K)*P	0356
IF X(I)<0.0 THEN GO TO L111	0367
IF UNOFF=1 THEN IF X(I)<150*ORIG(I) THEN GO TO L71	0369
IF X(I)<3.0*ORIG(I) THEN GO TO L71	0372
L111 DT+DT/101	0375
T+0.01	0376
FOR I=1 STEP 1 UNTIL 4 DO X(I)+ORIG(I)	0377
GO TO L01	0382
L71 IF K>1 THEN FOR I=1 STEP 1 UNTIL N DO X(I)+X(I)+0.5*D(1,K-1)	0383
IF X(I)<0.0 THEN GO TO L121	0392
IF UNOFF=1 THEN IF X(I)<150*ORIG(I) THEN GO TO L311	0393
IF X(I)<3.0*ORIG(I) THEN GO TO L311	0397
L121 DT+DT/101	0399

T=0.01	0401
FOR I=1 STEP 1 UNTIL 4 DO X(I)=ORIGX(I)	0402
GO TO L01	0408
L31: END	0409
FOR I=1 STEP 1 UNTIL N DO	0412
X(I)=X(I)+(D(I,1)*2+D(I,2)+2*D(I,3)+D(I,4))/6.0*(I,3)	0414
IF X(I)<0.0 THEN GO TO L131	0426
IF INDIFF=1 THEN IF X(I)>150*ORIGX(I) THEN GO TO L141	0428
IF X(I)<3.0*ORIGX(I) THEN GO TO L141	0431
L131: DT=DT/10	0434
T=0.01	0435
FOR I=1 STEP 1 UNTIL 4 DO X(I)=ORIGX(I)	0436
GO TO L01	0442
L141: IF TR(2)≠0 THEN GO TO L01	0443
IF TR(1)≠0 THEN I=2 ELSE I=1	0445
IF X(2)=(CX(I)*DIFF+ORIGX(2)) THEN BEGIN	0449
TR(I)=T	0452
GO TO L41	0453
END	0453
IF X(2)>(CX(I)*DIFF+ORIGX(2)) THEN IF XLAST(2)<(CX(I)*DIFF+ORIGX(2))	0453
THEN GO TO L51	0458
GO TO L41	0459
L51: TR(I)+TLAST+DT/2	0460
IF TR(2)≠0 THEN GO TO L41	0463
TRD+TR(2)-TR(1)	0464
IF TRD>0 THEN BEGIN WRITE(PRINT,FL1) GO TO L61 END	0466
IF RESTART=0 THEN BEGIN RESTART=1 GO TO L61 END	0471
IF P=0.30 THEN GO TO L22	0474
CVCN1=CVCN1*(1.25)	0475
IF CVCN1>500.0 THEN BEGIN WRITE(PRINT,FL3) STOPX=1 GO TO L61 END	0477
WRITE(PN(1),FL7,CVCN1)	0486
FOR I=1 STEP 1 UNTIL 4 DO X(I)=ORIGX(I)	0494

L16: IF T3<DEL12 THEN GO TO L43	0581
L17: IF T1SETR THEN BEGIN	0583
IF PB=0,30 THEN GO TO L22:	0585
IF RESTART=0 THEN BEGIN RESTART+1: GO TO L6: END:	0586
IF RR>0,0 THEN BEGIN	0591
RR+RR*0,25:	0592
IF RR<0,25 THEN BEGIN WRITE(PRINT,FL10): STOPX+1: GO TO L6: END:	0593
SETTLNG+0:	0601
FOR I=1 STEP 1 UNTIL 4 DO X(I)+UNTGX(I):	0601
GO TO START:	0606
END:	0607
PB+PB*0,1:	0607
IF PB>PBMAX THEN BEGIN WRITE(PRINT,FL6): STOPX+1: GO TO L6: END:	0608
SETTLNG+0:	0616
FOR I=1 STEP 1 UNTIL 4 DO X(I)+URTGX(I):	0616
GO TO START:	0621
END:	0622
SETTLNG+1:	0622
WRITE(PRINT,FL7):	0623
L20: IF T1SETR THEN GO TO L43	0626
IF ABS(N1-X(2))>ESS THEN BEGIN T4+0,0: GO TO L4: END:	0628
T4+T4*0,1:	0631
IF T4<DEL12 THEN GO TO L43	0633
WRITE(PRINT,FL8):	0634
GO TO L6:	0637
L4: IF RR#0,0 THEN WRITE(PRINT,FMT,T,X(1),X(2),X(3),X(4)):	0638
IF TSM THEN GO TO L0:	0654
IF RESTART=0 THEN BEGIN RESTART+1: GO TO L6: END:	0655
L22: IF Pd=0,30 THEN BEGIN	0658
Pd+0,05: PBMAX+0,25:	0660
FOR I=1 STEP 1 UNTIL 4 DO X(I)+ONTGX(I):	0661
WRITE(PRINT,FL13): GO TO START: END:	0669

GO TO START1	0498
L81 IF USUK=1 THEN GO TO L91	0499
IF SIGN(DI2+4)CHANGE THEN GO TO L151	0501
IF DI2+4>.0001 THEN GO TO L81	0505
L151 IF ABS(M1-X(2))<.05 THEN BEGIN USUK=17	0507
WRITE(PRINT,FL4) GO TO L41 END1	0511
IF RESTART=0 THEN BEGIN RESTART=11 GO TO L61 END1	0516
IF PR=0.30 THEN GO TO L221	0518
IF RR>0.0 THEN BEGIN	0520
RR=0.251	0521
IF RR<0.25 THEN BEGIN WRITE(PRINT,FL5) STOPX=11 GO TO L61 END1	0522
FOR I=1 STEP 1 UNTIL 4 DO X(I)=UNIGX(I)1	0531
GO TO START1	0535
END1	0536
-----	0536
PB+PB+0.11	0537
IF PB>PBMAX THEN BEGIN	0537
WRITE(PRINT,FL5) STOPX=11 GO TO L61 END1	0538
-----	0545
FOR I=1 STEP 1 UNTIL 4 DO X(I)=ORIGX(I)1	0545
GO TO START1	0549
L91 IF SETTLNG=1 THEN GO TO L201	0550
IF ABS(M1-X(2))<0.05*DIFF THEN T3=T3+DT ELSE BEGIN	0552
T3=0.01 GO TO L81 END1	0558
IF SIGN(M1-XLAST(2))*SIGN(M1-X(2)) THEN GO TO L161	0559
IF (M1-X(2))>0.0 THEN GO TO L161	0565
L191 IF T3<THR/2 THEN GO TO L41	0567
DELTT2=TRM/21	0569
GO TO L171	0571
L161 IF TSET(2)≠0 THEN GO TO L181	0571
IF TSET(1)≠0 THEN I=2 ELSE I=11	0573
TSET(I)=T3	0577
IF I=1 THEN GO TO L191	0578
DELTT2=(SET(I)-TSET(I)1)	0579

IF RR#0,0 THEN GO TO L21)	0673
RR#0,50)	0675
ORIGX(3)#0,02)	0675
FOR I#1 STEP 1 UNTIL 4 DO X(I)#ORIGX(I))	0677
WRITE(PRINT,FL(2))	0683
WRITE(PRINT,FL(5))	0687
GO TO START)	0690
L21: RR#RR#0,25)	0691
IF RR#0,25 THEN BEGIN WRITE(PRINT,FL(1)) STOPK#1) GO TO L6) END)	0693
FOR I#1 STEP 1 UNTIL 4 DO X(I)#ORIGX(I))	0701
GO TO START)	0705
L6: END RUNGKUTTA)	0706
	0009 IS 0724 LONG, NEXT SEG 0002
BEGIN	0091
LABEL L2,L50,L51,L100,LX)	0091
	START OF SEGMENT ***** 0012
Z#TIME(1))	0060
READ(CARD,FL1,CONFIG,TANK,MEDM,SLURRY,TKRRTNG,PNEU,PDO,P1,PIHAX,PWIN,	0001
P7,P13,T7))	0019
WRITE(PRINT,FL1,CONFIG,TANK,MEDM,SLURRY,TKRRTNG,PNEU,PDO,P1,PIHAX,	0025
PWIN,P7,P13,T7))	0041
READ(CARD,FL9,D1,D3,DV3,L1,H1,H3,M1,M2,DB,QDM,CVCM3))	0069
WRITE(PRINT,FL9,D1,D3,DV3,L1,H1,H3,M1,M2,DB,QDM,CVCM3))	0071
READ(CARD,FL11,FOR DX(2 STEP 1 UNTIL 5 DO LT(DX),RD,NB(1),NT(1),	0093
SEAL,CAPCTY,OP,TOP,E,U,G,K20))	0106
WRITE(PRINT,FL11,FOR DX(2 STEP 1 UNTIL 5 DO LT(DX),RD,NB(1),NT(1),	0121
SEAL,CAPCTY,OP,TOP,E,U,G,K20))	0136
READ(CARD,FL13,FON DX(1,2,4 DO ORIGX(DX),TRR,CR,OS,TSETR,ESS,DH,	0149
STATIC))	0171
WRITE(PRINT,FL13,FON DX(1,2,4 DO ORIGX(DX),TRR,CR,OS,TSETR,ESS,DH,	0175
STATIC))	0197
READ(CARD,FL10,FOR DX(1 STEP 1 UNTIL 20 DO LD(DX(1))	0201

FOR I=1 STEP 1 UNTIL 4 DO XE1:=ORIGX(I)F	0214
FLDUT=X(A)F	0219
FMAX=1.1F	0220
FMIN=0.05F	0221
R=50.0F	0222
HR=0.0F	0223
IF DB>0.0 THEN AB=3.14159*DB*DB/4 ELSE AB=PI*W*W	0223
IF DB>0 THEN DNUFF=1 ELSE DNUFF=0F	0233
FOR I=2 STEP 1 UNTIL 5 DO BEGIN	0236
IF LTI(I)=0 THEN FOR J=1 STEP 1 UNTIL 21 DO NFITEJ(I)=0	0238
ELSE READ(CARD,FL12,FOR J=1 STEP 1 UNTIL 21 DO NFITEJ(I))	0241
WRITE(PRINT,FL12,FOR J=1 STEP 1 UNTIL 21 DO NFITEJ(I))	0258
SUML=0F	0271
FOR J=1 STEP 1 UNTIL 20 DO SUML=SUML+NFITEJ(I)*LD(I)	0272
IF NFITE(21)=0 THEN GO TO L100F	0277
SUML=SUML+NFITE(21)*	0279
(5.9996500E-01-5.9953110E+01*RD+2.5896379E+01*RD+2	0279
-7.2291399*RD+3+1.0614061*RD+4-9.353222E-02*RD+5	0281
+4.8506803E-03*RD+6-1.3621214E-04*RD+7+1.5958646E-06*RD+8)F	0287
L100: IF I>3 THEN LECT(I)+LTI(I)+D3*SUML ELSE LECT(I)+LTI(I)+D1*SUML	0298
ENDF	0314
CLOSE(CARD,RELEASE)F	0316
IF DNUFF=1 THEN XIM=1.1*QUM ELSE XIM=XE1(I)F	0318
HP1=3.3E-3*AI*KG/(D1*U)F	0324
IF HP1<200 THEN BEGIN WRITE(PRINT,FL2012,HP1) GO TO LX: ENDF	0326
IF HP1<2000 THEN F2=1.9494*U/(G*XIM)	0337
ELSE F2=5.5E-3*(1+(2*PA*E/01+1*P6/HP1)*0.33333)F	0340
IF H1>H3 THEN H2=H1*H3 ELSE H2=0.0F	0350
P0=IF IANK=1 THEN 15.0 ELSE IF IANK=2 THEN P13 ELSE P00*LI/(L1-H1)F	0358
IF DNUFF=1 THEN NMRIR=5.275441H ELSE NMRIR=5.275*XIM*HR+0.19F	0365
CVI=NMRIR/SQRT((P1-P0)/G-H2*(2.1E-3*F2*(LE1E)+LE(3))*XIM*XIM/D1*5)F	0373
IF HP1<1000 THEN CYCNT+CVI/(0.707+0.11*(LN(HP1)-5.3))	0382

ELSE IF HP155000 THEN CVCN1+CV1/(0.804+0.0398*(LN(RP)))=6.91))	0346
ELSE IF HP15100000 THEN CVCN1+CV1/(0.948+0.01734*(LN(RP))=8.52))	0399
ELSE CVCN1+CV1	0410
WRITE(PRINT,FL20,CVCN1))	0414
NEXTV=1	0427
DV1=0.0	0428
RUMKUTIA(D1,D3,DV1,DV3,IF ONOFF=1 THEN 60.0 ELSE 0.0,NEXTV))	0428
IF STOPX=1 THEN GO TO L4	0432
NVALVS=0	0434
DELTP1=IF TANK=1 THEN P1MAX=15.0 ELSE IF TANK=2 THEN P1MAX=P13 ELSE	0434
P1MAX=P00	0439
CVMIN=FMIN*CVCN1*0.26	0441
CVMAX=FMAX*CVCN1*0.26	0442
P8=P1MAX	0444
DO READ(TAPE1,*,A,FOR DX1=1 STEP 1 UNTIL 10 DO BLOC1)) UNTIL A=NEON)	0445
DO READ(TAPE1,*,A) UNTIL A=99999	0464
READ(TAPE1,*,FOR I=1 STEP 1 UNTIL 9 DO DISPL1))FOR DX1=1 STEP 1 UNTIL	0474
4 DO DISCUST1(=DX1))	0482
IF NB1(1)≠0 THEN BEGIN	0493
FOR I=2 STEP 1 UNTIL 10 DO NIX1(=0)	0495
FOR I=2,3,4 DO NB1(=0)	0499
GO TO L50) END)	0508
J=0	0509
FOR I=1 STEP 1 UNTIL 4 DO BEGIN	0509
DUM=ASC(1))	0511
RATNG1(=1)+0)	0512
RATNG1(=2)+0)	0514
IF DUM=1 OR DUM=2 OR DUM=3 OR DUM=4 THEN BEGIN	0516
J=J+1	0520
NB1(J)=0(1))	0521
END END)	0523
FOR I=1 STEP 1 UNTIL 10 DO NIX1(=NIX1)	0525

LS0: I+1	0530
L51: READ(TAPE1,*,*MODEL(I),SIZE(I),TYPE(I),VPORT(I),QOZ(I),VPFC(I),	0531
VPNC(I),VPLNK(I),VPTIME(I),QOLNK(I),MOTIME(I),TMNE(I),TMAX(I),	0545
TRAVL(I),HODY(I))(L2)	0556
J=0	0563
CVS(VPFC(I))	0564
J=J+VPFC(I)	0565
L1MVFC(I)+J	0567
CVS(VPNC(I))	0568
J=J+VPNC(I)	0569
L1MVRC(I)+J	0570
CVS(QOZ(I))	0572
J=J+QOZ(I)	0573
NUMCVS(I)+J	0574
READ(TAPE1,*,*FOR J=1,2,3 DOINB(I,J),FOR DX=1 STEP 1 UNTIL 4	0575
DO(RATNG(I,J,DX),BNCOST(I,J,DX))	0590
FOR J=1,2,3 DO RATNG(I,J,DX)=RATNG(I,J,DX)	0602
READ(TAPE1,*,*EXT(I),RAD(I),FOR J=1,2,3 DOINBON(I,J),BONCOST(I,J),	0615
SEAL(I),FOR J=4 STEP 1 UNTIL 9 DOINBON(I,J),BONCOST(I,J))	0635
READ(TAPE1,*,*FOR J=1 STEP 1 UNTIL 6 DOINT(I,J),TMCOST(I,J),	0647
FOR J=1 STEP 1 UNTIL 4 DO MCOS(I,J),FOR J=1,2 DO(POS(I,J),	0660
POSCOST(I,J))	0673
I=I+1	0679
GO TO L51	0681
L2: NVALVS=I-1	0681
BEGIN	0683
LABEL L4,L5,L10,L11,L7,L6,L8,L13,L9,L12,L14,L15,L16,L17,L18,L20	0683
	START OF SEGMENT ***** 0013
L21,L24,L25,L23,L22,L26,L27,L30,L35,L36,L34,L37,L32,L31,L33	0000
L46,L45,L47,L41,L40,L43,L44,L61,L42,L3,L60,L63,L64,L65,L66,L67,L68	0000
L42,L28	0000
SWITCH SWG01=L10,L11,L12,L12	0000

FOR I=1 STEP 1 UNTIL 4 DO BEGIN	0005
IF NBI[I]=0 THEN GO TO L2I	0007
DUM=ABS(NBI[I])	0008
GO TO SNGH(DUM)	0009
L1I: IF P8>125 THEN GO TO L3I	0011
IF T7>353 THEN GO TO L4I	0013
RATNGI[I+1]=125	0014
RATNGI[I+2]=125	0016
GO TO L2I	0018
L3I: IF P8>175 THEN GO TO L5I	0019
IF T7>150 THEN GO TO L4I	0021
RATNGI[I+1]=125	0022
RATNGI[I+2]=125	0024
GO TO L2I	0026
L4I: IF T7<406 THEN BEGIN	0027
RATNGI[I+1]= 250	0029
RATNGI[I+2]=250 END	0031
GO TO L2I	0033
L5I: IF P8<250 THEN GO TO L4I	0034
IF P8>400 OR T7>150 THEN GO TO L2I	0035
RATNGI[I+1]= 250	0037
RATNGI[I+2]=250	0039
GO TO L2I	0042
L1I: IF P8>125 THEN GO TO L6I	0042
IF T7>400 THEN GO TO L7I	0044
RATNGI[I+2]=125	0045
GO TO L2I	0047
L7I: IF T7>422 THEN GO TO L2I	0048
RATNGI[I+1]=300	0050
GO TO L2I	0052
L6I: IF P8>150 OR T7>400 THEN GO TO L8I	0052
RATNGI[I+1]=150	0055

GO TO L20;	0057
L9: IF P8>175 THEN GO TO L9;	0058
IF T7>150 THEN GO TO L13;	0059
RATNGI(2,2)+125;	0060
GO TO L20;	0062
L13: IF T7>405 THEN GO TO L7;	0063
RATNGI(2,2)+250;	0065
GO TO L20;	0067
L9: IF P8<250 THEN GO TO L13; IF P8<300 THEN GO TO L7;	0068
IF P8>400 THEN GO TO L10;	0070
L15: IF T7<150 THEN RATNGI(2,2)+250;	0071
GO TO L20;	0075
L14: IF P8>500 THEN GO TO L20;	0076
IF T7<150 THEN RATNGI(2,1)+300;	0077
GO TO L20;	0080
L12: IF DUM=3 THEN P15+3.0105262#2=1.6768665#-1#T7+7.0941098#-4#T7#T7	0081
+1.2550600#-4#T7+3-6.5264316#-10#T7+4 ELSE P15+2.5691054#2	0084
+4.6506157#-1#T7+4.4136753#-3#T7#T7+1.1069110#-5#T7+3	0095
-1.3206283#-8#T7+4+7.053455#-12#T7+5+1.5912500#-15#T7+6;	0098
IF P8>P15 THEN GO TO L16;	0107
RATNGI(DUM,1)+ 150;	0108
RATNGI(DUM,2)+150;	0110
GO TO L20;	0112
L16: IF DUM=3 THEN P30+7.6256457#2=7.1526411#-1#T7+2.4541191#-3#T7+2	0121
=4.0495042#-6#T7+3+1.6352804#-9#T7+4 ELSE P30+8.3103557#2=2.402578#T7	0124
+2.0435072#-2#T7#T7+9.8796968#-5#T7+3+2.8161753#-7#T7+4	0135
=4.9388141#-10#T7+5+5.1928453#-13#T7+6+3.0120952#-16#T7+7	0140
+7.2269829#-20#T7+8+6.8980817#-24#T7+9+4.6982276#-27#T7+10;	0147
IF P8>P30 THEN GO TO L17;	0158
RATNGI(DUM,1)+ 300;	0160
RATNGI(DUM,2)+300;	0162
GO TO L20;	0164

L17: IF DUM=3 THEN P40+1.0174975P3=8.4497081P-1xT7+3.0019446P-3xT7xT7	0176
=5.1700392P=6xT7+3+2.1133420P=9xT7+4 ELSE P40+1.1861529P3=4.0173019xT7	0179
+2.3386695P=2xT7xT7+6.4850059P=5xT7+3+8.8982563P=6xT7x4	0190
=6.3425416P=11xT7+5+2.2109879P=14xT7+6+2.8998542P=18xT7+7	0195
IF P8>P40 THEN GO TO L18	0205
RATNGI(DUM,1)+ 400	0206
RATNGI(DUM,2)+=400	0208
GO TO L20	0210
L18: IF DUM=3 THEN P60+1.2966267P3+4.2001297xT7=4.2924664P=2xT7+2	0220
+1.9283055P=4xT7+3+4.5558383P=7xT7+4+5.7819949P=10xT7+5	0223
=3.7739105P=13xT7+6+9.9562479P=17xT7+7 ELSE P60+1.6750744P3	0228
=4.4849198xT7+3.0768629P=2xT7+2+1.1768237P=4xT7+3+2.7933649P=7xT7+4	0245
+4.4917889P=10xT7+5+4.6604068P=13xT7+6+2.7668428P=16xT7+7	0250
+6.9711866P=20xT7+8+5.647992P=24xT7+9+4.4213702P=27xT7+10	0257
IF P8>P60 THEN GO TO L28	0268
RATNGI(DUM,1)+ 600	0270
RATNGI(DUM,2)+=600	0272
L20: END	0274
L28: FOR I=1 STEP 1 UNTIL 4 DO FOR K=1,2 DO IF RATNGI(I,K)≠0 THEN	0277
GO TO L21	0296
WRITE(PRINT,FL1000)	0300
GO TO LX	0303
L21: IF SEALI=0 THEN GO TO L22	0305
IF T75750 THEN GO TO L25	0307
L24: WRITE(PRINT,FL1001)	0308
GO TO LX	0312
L25: IF P8>150 THEN GO TO L23	0314
FOR I=1 STEP 1 UNTIL 10 DO IF ABS(CR(I))=5 THEN MONEL=1	0316
FOR I=1 STEP 1 UNTIL 10 DO IF ABS(CR(I))=4 THEN STEEL=1	0321
IF MONEL=0 AND STEEL=0 THEN BEGIN	0327
WRITE(PRINT,FL2010) GO TO LX	0330
END	0335

GO TO L30F	0335
L23: IF PH>400 THEN GO TO L24J	0336
MONEL=0J	0337
FOR I+1 STEP 1 UNTIL 10 DO IF ABS(REII)≠0 THEN STEEL+1J	0338
IF STEEL=0 THEN BEGIN	0343
WRITE(PRINT,FL2011)J GO TO LXJ	0345
ENDJ	0350
GO TO L30J	0350
L22: IF T7<450 OR T7>32 THEN GO TO L26J	0351
EXTCOL+1J	0353
GO TO L30J	0354
L26: IF T7<32 OR T7>450 THEN GO TO L27J	0355
GO TO L30J	0357
L27: IF T7<450 OR T7>1200 THEN GO TO L24J	0358
RAOFIN=1J	0360
L30: FOR I+1 STEP 1 UNTIL 10 DO NTI(I)+0J	0361
IF SLURRY=1 THEN GO TO L31J	0367
IF DELTPI>50 THEN BEGIN	0368
IF DELTPI>300 THEN GO TO L32 ELSE GO TO L35J	0370
ENDJ	0371
M+2J	0371
NTRIMJ	0372
IF IN0=1 THEN NTI(1)+M ELSE NTI(1)+0J	0373
L35: M+4J	0377
NTRIMJ	0378
IF IN0=1 THEN NTI(2)+M ELSE NTI(2)+0J	0379
IF NTI(1)≠0 OR NTI(2)≠0 THEN GO TO L33J	0383
L36: IF ONOFF≠1 THEN GO TO L34J	0386
NTI(1)+10J	0388
GO TO L33J	0389
L34: M+5J	0390
NTRIMJ	0390

IF IND=1 THEN NTI(1)+M ELSE NTI(1)+0	0391
M+7	0395
NTRIM	0396
IF IND=1 THEN NTI(2)+M ELSE NTI(2)+0	0396
M+8	0401
NTRIM	0401
IF IND=1 THEN NTI(3)+M ELSE NTI(3)+0	0402
M+9	0406
NTRIM	0407
IF IND=1 THEN NTI(4)+M ELSE NTI(4)+0	0407
IF NTI(1)≠0 OR NTI(2)≠0 OR NTI(3)≠0 OR NTI(4)≠0 THEN GO TO L33	0412
L37: SAUNDERS+1	0417
GO TO L33	0418
L32: IF DELTPI=000 THEN GO TO L36	0419
WRITE(PRINT,FL1003)	0421
GO TO LX	0424
L31: IF DELTPI>50 THEN GO TO L37	0426
M+8	0428
NTRIM	0429
IF IND=1 THEN NTI(1)+M ELSE NTI(1)+0	0429
IF NTI(1)≠0 THEN GO TO L37	0433
L33: J+0	0435
END REQUIREMENT BLK	0436
	0013 IS 0441 LONG. NEXT SEG 0012
BEGIN	0684
COMMENT ELIMINATION BLK	0684
LABEL L45,L46,L47,L41,L40,L42,L43,L48,L61,L60,L62,L63,L30,	0684
	START OF SEGMENT ***** 0014
L66,L67,L65,L64,L68,L1,L3,L75,L76,L80,L105,L101,L102,L103,L104,L74,	0000
L77,L79,L78,L81,L82,L83,L84,L85,L86,L20,L21,L22,L90,L91,L92,L69	0000
IF UNOFF#1 THEN GO TO L85	0000
FOR J=1 STEP 1 UNTIL NVALS DO BEGIN	0001

IF NVALS=0 THEN GO TO L46J	0002
K=NUMCVS(IJ)	0003
IF CV(I,K)<CVMIN OR CV(I,K)>CVMAX THEN GO TO L46J	0004
J=J+1J	0009
AVAIL(IJ)=IJ	0010
L46J ENDJ	0011
GO TO L47J	0014
L45J FOR I=1 STEP 1 UNTIL NVALS DO BEGIN	0014
IF CAPCY=1 THEN BEGIN [INITL+1J IFINAL+LIMVFC(IJ) END	0016
ELSE BEGIN [INITL+LIMVFC(IJ)+1J IFINAL+LIMVRC(IJ) ENDJ	0019
FOR K=[INITL STEP 1 UNTIL IFINAL DO BEGIN	0022
IF CV(I,K)<CVMIN OR CV(I,K)>CVMAX THEN GO TO L1J	0023
J=J+1J	0027
AVAIL(IJ)=IJ	0028
COLL(IJ)=KJ	0030
GO TO L3J	0031
L1J ENDJ	0031
L3J ENDJ	0034
L47J IF J=0 THEN BEGIN	0037
WRITE(PRINT,FL1004J)	0039
GO TO L4J	0042
ENDJ	0044
COMPAT=J	0044
FOR I=1 STEP 1 UNTIL COMPAT DO BEGIN	0045
M=AVAIL(IJ)	0047
DMITE(IJ)=0J	0048
FOR J=1 STEP 1 UNTIL 4 DO BEGIN	0049
IF NM(IJ)=0 THEN GO TO L41J	0050
L=0J	0051
FOR K=1 STEP 1 UNTIL 3 DO IF ABS(NM(IJ))=NM(I,K) THEN L=L+1J	0052
IF L>0 THEN GO TO L40J	0059
L41J ENDJ	0060

OMIT(I)+1	0063
L40: END	0064
TAKOUT	0067
IF COMPAT=0 THEN BEGIN	0067
WRITE(PRINT,FL1005) GO TO LX	0069
END	0074
FOR (I) STEP 1 UNTIL COMPAT DO BEGIN	0074
M=AVAIL(I)	0076
OMIT(I)+0	0077
L+0	0078
FOR J=1 STEP 1 UNTIL 4 DO BEGIN	0079
DX1=0	0080
IF NB(I,J)=0 THEN GO TO L43	0080
FOR K=1 STEP 1 UNTIL 3 DO IF ABS(NB(I,J))+NB(M,K) THEN DX1+K	0082
IF DX1=0 THEN GO TO L43	0088
IF RATNG(I,J,1)=0 THEN GO TO L42	0090
FOR K=1 STEP 1 UNTIL 3 DO IF MATNG(I,J,1)+RATNG(M,DX1,K) THEN	0092
L+L+1	0096
IF L>0 THEN GO TO L44	0100
L42: IF RATNG(I,J,2)=0 THEN GO TO L43	0101
IF RATNG(I,J,2)+RATNG(M,DX1,K) THEN L+L+1	0104
IF L>0 THEN GO TO L44	0109
L43: END	0110
OMIT(I)+1	0113
L44: END	0114
TAKOUT	0117
IF COMPAT=0 THEN BEGIN	0117
WRITE(PRINT,FL1006) GO TO LX	0119
END	0124
IF DNUPFA) THEN GO TO L60	0124
FOR I=1 STEP 1 UNTIL COMPAT DO BEGIN	0125
M=AVAIL(I)	0127

DMIT(I)+0)	012A
IF DELTPIN,NUMCVSEM,K)=0 THEN GO TO L61)	0129
L+0)	0132
FOR K=1,2 DO	0133
IF DELTPI)DELTPIN,NUMCVSEM,K) THEN L=L+1)	0138
IF L>0 THEN GO TO L61)	0143
DMIT(I)+1)	0144
L61) END)	0145
GO TO L62)	0148
L60) FOR I=1 STEP 1 UNTIL COMPAT DO BEGIN	0148
M+AVAIL(I)	0150
DMIT(I)+0)	0151
K+COL(I)	0152
IF DELTPIN,K,1)=0 THEN GO TO L30)	0153
L+0)	0156
FOR DX1=1,2 DO IF DELTPI) DELTPIN,K,DX1) THEN L=L+1)	0157
IF L>0 THEN GO TO L30)	0166
DMIT(I)+1)	0168
L30) END)	0169
L62) TAKOUT)	0172
IF COMPAT=0 THEN BEGIN	0173
WRITE(PRINT,FL1007)) GO TO LX)	0174
END)	0180
FOR I=1 STEP 1 UNTIL COMPAT DO BEGIN	0180
M+AVAIL(I)	0181
DMIT(I)+0)	0182
IF TYPE(M)="SAN" THEN BEGIN	0183
IF EXTCOL#1 AND RADFIN#1 AND SEAL#0 THEN GO TO L64) ELSE BEGIN	0184
DMIT(I)+1)	0188
GO TO L64)	0189
END)	0191
END)	0191

IF SEAL1>0 THEN GO TO L631	0191
IF EXTCOL#1 THEN IF EXT[M]#1 THEN GO TO L64 ELSE GO TO L651	0192
IF RADFIN#1 THEN IF RAD[M]#1 THEN GO TO L64 ELSE GO TO L651	0195
GO TO L641	0198
L631 IF SEAL[M]=0 THEN GO TO L651	0199
L#01	0201
IF MUNEL#1 THEN GO TO L661	0202
FOR N#4 STEP 1 UNTIL 9 DO FOR J#1 STEP 1 UNTIL 4 DO	0203
IF ABS(NBI[J])=NBUN[M,K] THEN L#L+11	0206
GO TO L671	0214
L661 FOR K#7,8,9 DO FOR J#1 STEP 1 UNTIL 4 DO	0215
IF ABS(NBI[J])=NBUN[M,K] THEN L#L+11	0223
L671 IF L>0 THEN GO TO L681	0229
L651 OMIT[I]+1	0231
L641 END1	0233
TAKOUT1	0236
IF COMPAT#0 THEN BEGIN	0236
WRITE(PRINT,FL10081) GO TO LK1	0238
END1	0243
FOR I#1 STEP 1 UNTIL COMPAT DO BEGIN	0243
M#AVAIL[I]1	0245
OMIT[I]+01	0246
IF TYPE[M]="SAN" THEN GO TO L681	0247
L#01	0248
FOR K#1 STEP 1 UNTIL 10 DO BEGIN	0249
IF NTI[K]=0 THEN GO TO L691	0252
FOR J#1 STEP 1 UNTIL 5 DO IF ABS(NTI[K])=NT[M,J] THEN L#L+11	0253
IF L>0 THEN GO TO L681	0261
L691 END1	0262
OMIT[I]+11	0265
L681 END1	0266
TAKOUT1	0269

IF COMPAT=0 THEN BEGIN	0269
WRITE(PRINT,FL1009) GO TO LX3	0271
END3	0276
IF UP#1 THEN GO TO L203	0276
FOR I=1 STEP 1 UNTIL COMPAT DD BEGIN	0277
M+AVAIL(I)	0279
OMIT(I)+0	0280
IF UNOFF#1 THEN K+NUMCVS(M) ELSE K+COL(I)	0281
IF DELTP(M,K)+1=0.0 THEN BEGIN DX1+1 GO TO L223 END3	0285
FOR DX1+1,2 DO IF DELTP(SDELTP(M,K+DX1)) THEN GO TO L223	0289
L223 IF ACTCOST(M,K+DX1)>0 THEN BEGIN	0297
IF TOP#1 THEN J#2 ELSE J#43 END ELSE	0301
BEGIN IF TOP#1 THEN J#1 ELSE J#33 END3	0308
MAN(I)+J	0308
IF NCCOST(M,J)>0 THEN GO TO L213	0309
OMIT(I)+1	0311
L213 END3	0312
TAKOUT3	0315
IF COMPAT=0 THEN BEGIN WRITE(PRINT,FL1010) GO TO LX3 END3	0315
L203 IF UNOFF#1 THEN GO TO L1053	0322
IF RR#0 THEN GO TO L1053	0324
IF PNEU#1 THEN GO TO L1013	0325
FOR I=1 STEP 1 UNTIL COMPAT DD BEGIN	0326
M+AVAIL(I)	0328
IF (MODEL(M) MOD 100)=6 THEN GO TO L1023	0329
OMIT(I)+1	0331
L1023 END3	0332
TAKOUT3	0335
IF COMPAT=0 THEN BEGIN WRITE(PRINT,FL3000) GO TO LX3 END3	0335
GO TO L1053	0342
L1013 FOR I=1 STEP 1 UNTIL COMPAT DD BEGIN	0343
M+AVAIL(I)	0344

IF (MODEL(4) MOD 100)*6 THEN GO TO L103	0345
DMIT(I)*1	0347
L103: END	0348
TAKOUT	0351
L105: FOR I=1 STEP 1 UNTIL CUMPA: DO BEGIN	0351
TOTCUST(I)+0.0	0353
M+AVAIL(I)	0354
IF UNOFF=1 THEN K+NUMCYS(4) ELSE K+COLC(I)	0355
IF DELTP(M,K+1)=0.0 THEN BEGIN DX1=1 GO TO L104: END	0359
FOR DX1=1,2 DO IF DELTP(SDELTP(M,K,DX1)) THEN GO TO L104:	0363
L104: TOTCUST(I)+ACTCOST(M,K,DX1)	0371
FOR J=1 STEP 1 UNTIL 4 DO FOR K=1,2 DO BEGIN	0375
NBX(J,K)+0: RATNGX(J,K)+0: CUSTX(J,K)+0:	0381
END	0387
FOR J=1 STEP 1 UNTIL 4 DO BEGIN	0389
IF NB1(J)=0 THEN GO TO L76:	0391
FOR K=1 STEP 1 UNTIL 3 DO	0392
IF ABS(NB1(J))-NB(M,K) THEN BEGIN	0394
IF RATNG(J,1)=0 THEN GO TO L75:	0396
FOR DX1=1 STEP 1 UNTIL 3 DO	0399
IF RATNG(J,1) < RATNG(M,K,DX1) THEN BEGIN	0400
NBX(J,1)+NB(M,K):	0404
RATNGX(J,1)+RATNG(M,K,DX1):	0407
CUSTX(J,1)+BCUST(M,K,DX1):	0410
GO TO L75:	0414
END	0415
L75: IF RATNG(J,2)=0 THEN GO TO L76:	0417
IF RATNG(J,2) < RATNG(M,K,4) THEN BEGIN	0420
NBX(J,2)+NB(M,K):	0424
RATNGX(J,2)+RATNG(M,K,4): CUSTX(J,2)+BCUST(M,K,4):	0427
END	0434
L76: END: END	0439

FOR J=1 STEP 1 UNTIL 4 DO FOR K=1,2 DO	0439
IF NBX(J,K)≠0 THEN BEGIN	0444
J1=J K1=K GO TO L303	0448
END	0450
L001 FOR J=1 STEP 1 UNTIL 4 DO FOR K=1,2 DO	0453
IF COSTX(J,K)≠0 THEN	0459
IF COSTA(J,K) < COSTX(J,K) THEN BEGIN	0460
J1=J K1=K	0464
END	0465
TOTCOST(1)+TOTCOST(1)+COSTX(J1,K1)	0468
FOR J=1 STEP 1 UNTIL 3 DO IF NBX(J,K1)≠NBH(M,J) THEN BEGIN	0471
INDX1(1)=J FOR K=1 STEP 1 UNTIL 4 DO IF RATNGX(J,K1)≠RATNG(M,J,K)	0476
THEN INDX2(1)=K END	0482
IF SEAL≠0 THEN GO TO L743	0488
IF EXTCOL=1 OR RADFIN=1 THEN BEGIN	0490
FOR J=1 STEP 1 UNTIL 3 DO	0492
IF NBX(J,K1)≠NBUN(M,J) THEN K=J2	0493
END ELSE PLAIN=1	0499
IF PLAIN≠1 THEN TOTCOST(1)+TOTCOST(1)+BONCOST(M,K)	0500
GO TO L773	0504
L743 IF STEEL=0 THEN BEGIN FOR J=4,5,6 DO	0505
IF NBI(J,K1)≠NBUN(M,J) THEN K=J2 END	0514
ELSE BEGIN FOR J=7,8,9 DO	0519
IF NBX(J,K1)≠NBUN(M,J) THEN K=J2 END	0526
TOTCOST(1)+TOTCOST(1)+BONCOST(M,K)	0531
L773 FOR J=1 STEP 1 UNTIL 10 DO FOR K=1 STEP 1 UNTIL 6 DO	0530
IF NT(J)≠0 THEN	0537
IF ABS(NT(J))=NT(M,K) THEN BEGIN	0538
K1=K GO TO L793	0541
END	0542
L793 FOR J=1 STEP 1 UNTIL 10 DO BEGIN	0547
IF NT(J)=0 THEN GO TO L783	0548

FOR K+1 STEP 1 UNTIL 6 DO	0549
IF ABS(NTI(JJ))=NI(M,K) THEN IF TMCUST(M,K)<TMCUST(M,K1) THEN K1=K	0551
L78: END	0560
TOTCUST(I)+TOTCOST(I)+TMCUST(M,K1)	0562
TMCOST(I)*K1	0565
IF DP#1 THEN GO TO L90	0566
TOTCOST(I)+TOTCOST(I)+TMCOST(M,MANC(I))	0567
L90: IF POSCOST(M,2)50 THEN GO TO L91	0571
TOTCOST(I)+TOTCOST(I)+POSCOST(M,2)	0573
GO TO L92	0576
L91: IF POSCOST(M,1)50 THEN GO TO L92	0576
TOTCOST(I)+TOTCOST(I)+POSCOST(M,1)	0579
L92: END	0582
FOR I+1 STEP 1 UNTIL COMPAT DO BEGIN	0585
AVAILX(I)+AVAIL(I) CHEAP(I)+0: END	0586
J+0	0591
IFINAL=COMPAT	0591
L81: J+J+1	0592
K+1	0594
FOR I+1 STEP 1 UNTIL IFINAL DO	0595
IF TOTCOST(I)<TOTCOST(K) THEN K=I	0596
CHEAP(J)+AVAILX(K)	0600
FOR M+K STEP 1 UNTIL IFINAL=1 DO BEGIN	0602
TOTCUST(M)+TOTCOST(M+1) AVAILX(M)+AVAILX(M+1)	0609
END	0613
IFINAL=IFINAL-1	0613
IF IFINAL=0 THEN GO TO L82	0614
GO TO L81	0616
L82: J+1	0616
M+0	0617
L83: IF PNEU#1 OR MK>0.0 THEN GO TO L84	0618
IF TAM#1 THEN GO TO L84	0621

IF H157.2/G AND PB>0.25 AND PB50.35 THEN GO TO L851	0622
L841 IF DNOFF=1 THEN BEGIN K=NUNCVS(CHEAP(J)); GO TO L842 END	0626
FOR I=1 STEP 1 UNTIL COMPAT DO IF AVATL(I)=CHEAP(J) THEN K=COL(I)	0633
L861 CVCN1=CVC(CHEAPE(J),K)/0.26	0639
NEXTV=0	0641
FOR I=1 STEP 1 UNTIL 4 DO X(I)=ORIG(X(I))	0642
RUNGKUTTA(D1,D3,SIZE(CHEAP(J)),DV3,IF DNOFF=1 THEN DOT(CHEAP(J))	0647
ELSE 0,NEXTV)	0651
IF NEXTV=0 THEN BEGIN M=M+1; CHEAP(M)=CHEAP(J); END	0652
IF M>1 THEN GO TO L851	0656
J=J+1	0658
IF JSCOMPAT THEN GO TO L831	0659
L851 END ELIMINATION BLK	0660
	0014 IS 0662 LONG NEXT SEG 0012
BEGIN	0685
COMMENT PRINTING BLOCK	0685
FORMAT FL4000("WARNING - CHECK DISPLACER COMPATIBILITY"),	0685
	START OF SEGMENT ***** 0015
	START OF SEGMENT ***** 0016
FL4001(" (1)",X16,"ORDER SPECIAL DISPLACER"),	0000
FL4002("NO TRANSMITTER AVAILABLE"),	0000
FL4003("SYSTEM",X18,"COMPONENT",X18,"COST"),	0000
FL4004(" (1)",X16,"TYPE 72-25,SERIES 1",X10,"S",F6.2),	0000
FL4005(" (2)",X16,"TYPE 72-25,SERIES 1",X10,"S",F6.2),	0000
FL4006(" (2)",X16,"ORDER SPECIAL DISPLACER"),	0000
FL4007(" (1)",X20,A1,X20,"S",F6.2),	0000
FL4008(" (2)",X24,A1,X20,"S",F6.2),	0000
FL4009(" (1)",X16,"MODEL NO. 29501",X14,"S",F6.2),	0000
FL4010(X20,"VERTICAL SCALE CONTROL",X7,"S",F6.2),	0000
FL4011(" (2)",X16,"MODEL NO. 29501",X14,"S",F6.2),	0000
FL4012(" (1)",X16,"ELECTRIC TEL-O-SET DELT/P/L",X7,"S",F6.2),	0000
FL4013(X20,"K7165A",X23,"S",F6.2),	0000

FL0014(X20,"727070"001",X19,"S",F6.2),	0000
FL0015(" (2)",X16,"ELECTRICK TEL=0-SET DELTP/1 S",F6.2),	0000
FL0("RN = ",F5.2),	0000
FL1("/PB = ",F5.2),	0000
FL0016(" (1)",X16,"R7089 B",X23,"S",F5.2),	0000
FL0017(X20,"PROBES",X24,"S",F5.2),	0000
FL0018(" (2)",X16,"R7089 B",X23,"S",F5.2),	0000
FL2(X20,"VALVE MODEL NO. ",X14/X21,"SIZE",F5.2," INCHES"),	0000
FL3(X21,"TYPE ",A6,"SEATED"),	0000
FL4(X21,"TYPE ANGLE"),	0000
FL5(X21,"TYPE SAUNDERS"),	0000
FL6(X21,"CV",F7.2),	0000
FL7(X21,"ACTUATOR NO. ",A3,X12,"S",F6.2),	0000
FL30("/DELTA H = ",F5.2),	0000
	0016 IS 0256 LONG, NEXT SEG 0015
	START OF SEGMENT ***** 0017
FL21(X21,"LINKAGE NO.",A6/X21,"MOTOR NO.",A6,X13,"S",F6.2),	0000
	0017 IS 0015 LONG, NEXT SEG 0015
LABEL L12,L3,L4,L5,L6,L7,L8,L9,L10,L11,L20,L21,L22,L27,L28,	0000
L30,L31,L32,L33,L34,L40,L41,L42,L49,L41,L52,L57,L60,	0000
L70,L71,L72,L73)	0000
FILL NAME(1,*) WITH "CAST J", "BRONZE", "CARBON", "316 S.", "MONEL "	0000
	START OF SEGMENT ***** 0018
"DURIME", "HASTEL", "HASTEL", "440C S.", "STELLI")	0002
	0018 IS 0010 LONG, NEXT SEG 0015
FILL NAME(2,*) WITH "MON " " " "STEEL", " S. " " "	0002
	START OF SEGMENT ***** 0019
"T 20 " "LOY B " "LOY C " " S. " "TE "J	0004
	0019 IS 0010 LONG, NEXT SEG 0015
IF ONOFF=1 THEN GO TO L12)	0004
IF NNF0 THEN GO TO L3)	0005
IF PNEU#1 THEN GO TO L4)	0004

L31 IF YANK#1 THEN GO TO L51	0007
L111 FOR I=1 STEP 1 UNTIL 10 DO BEGIN	0009
IF ABS(RC(I))=1 THEN GO TO L61	0011
END	0012
L101 FOR I=1 STEP 1 UNTIL 10 DO BEGIN	0015
IF ABS(RC(I))=3 THEN GO TO L71	0016
END	0017
WRITE(PRINT,FL4000)	0020
L71 IF INKRATNG\$300.0 THEN BEGIN K=31 GO TO L81 END	0023
IF INKRATNG\$600.0 THEN BEGIN K=41 GO TO L81 END	0028
WRITE(PRINT,FL4001)	0032
SPECIAL=1	0035
GO TO L91	0036
L81 FOR I=1 STEP 1 UNTIL 9 DO IF CR<DISPL(I) THEN GO TO L91	0036
I=91	0041
L91 WRITE(PRINT,FL1+PB)	0042
IF RR#0.0 THEN WRITE(PRINT,FL6+RR)	0051
WRITE(PRINT,FL4003)	0060
IF SPECIAL=1 THEN BEGIN TOTAL=0.01 GO TO L271 END	0063
COST+DISCOST(I,K)	0066
IF RR#0.0 THEN COST=COST+25.01 TOTAL=COST	0068
WRITE(PRINT,FL4004+COST)	0071
L271 J=CHLAP(I)	0080
L301 FOR J=1 STEP 1 UNTIL COMPAT OR IF AVAILC(J)=J THEN M=COLE(J)	0082
WRITE(PRINT,FL2+MODEL(J),SIZE(J))	0087
IF TYPE(J)#"05" AND TYPE(J)#"55" THEN GO TO L201	0096
IF TYPE(J)#"05" THEN TYPE(J)+"DOUBLE" ELSE TYPE(J)+"SINGLE"	0101
WRITE(PRINT,FL3+TYPE(J))	0109
GO TO L211	0119
L201 IF TYPE(J)#"ANG" THEN WRITE(PRINT,FL4) ELSE WRITE(PRINT,FL5)	0119
L211 WRITE(PRINT,FL6+CV(J)+M)	0130
IF DELT(C(J),M)=0.0 THEN BEGIN DX1=11 GO TO L221 END	0140

FOR I=1,2 DO IF DELTPISELTPI(J,M,DX1) THEN GO TO L22	0144
L22: CUST*ACTCOST(J,M,DX1) TOTAL+TOTAL+CUST	0153
WRITE(PRINT,FL7,ACTNO(J,M,DX1),CUST)	0156
PRINT	0168
IF CHEAP(2)=0 OR J=CHEAP(2) THEN GO TO LX	0168
J=CHEAP(2)	0173
WRITE(PRINT,CBL)	0174
IF SPECIAL=1 THEN BEGIN WRITE(PRINT,FL4006) TOTAL+0.0 GO TO L30 END	0177
FOR I=1 STEP 1 UNTIL 9 DO IF CR<DISPL(I) THEN GO TO L28	0183
I=9	0188
L28: CUST+DISCOST(I,K)	0189
IF RR#0.0 THEN COST+COST+25.0	0191
TOTAL+COST	0194
WRITE(PRINT,FL4005,COST)	0195
GO TO L30	0204
L6: IF INKRATNGS125.0 THEN BEGIN K=1 GO TO L6 END	0204
IF INKRATNGS250.0 THEN BEGIN K=2 GO TO L6 END	0209
GO TO L10	0213
L5: IF M1>7.2/6 THEN GO TO L11	0213
IF RR#0.0 THEN GO TO L33	0215
IF PB<0.25 OR PB>0.35 THEN GO TO L11	0217
PB=0.30	0219
J=0	0220
FOR I=1 STEP 1 UNTIL COMPAT DO BEGIN	0221
M=CHEAP(I)	0226
IF MODEL(M)#4705 AND MODEL(M)#1405 AND MODEL(M)#9105	0227
AND MODEL(M)#9205 THEN GO TO L31	0229
NEXTV=0	0232
FOR K=1 STEP 1 UNTIL 4 DO K(K)=OHIG(K)	0233
RUNGUITAC(I)=33+S(ZETN)/DV3+0/NEXTV	0241
IF NEXTV#0 THEN BEGIN J=J+1 CHEAP(J)=CHEAP(I) END	0244
IF J#1 THEN GO TO L32	0244

L314 END;	0249
IF J=0 THEN GO TO L327	0252
CHEAP(2)=0;	0253
L321 IF J=0 THEN COMPONENT="B" ELSE COMPONENT="A";	0254
WRITE(PRINT,FL1,PB);	0258
WRITE(PRINT,FL4003);	0266
IF COMPONENT="A" THEN COST=118.0 ELSE COST=147.0;	0269
TOTAL=COST;	0274
WRITE(PRINT,FL4007,COMPONENT,COST);	0275
J=CHEAP(1);	0286
L341 FOR I=1 STEP 1 UNTIL COMPAT DO IF AVAIL(I)=J THEN M=COL(I);	0287
WRITE(PRINT,FL2,MODEL(I),SIZE(I));	0293
IF TYPE(I)≠"DS" AND TYPE(I)≠"SS" THEN GO TO L401	0304
IF TYPE(I)≠"DS" THEN TYPE(I)="DOUBLE" ELSE TYPE(I)="SINGLE";	0307
WRITE(PRINT,FL3,TYPE(I));	0315
GO TO L417	0325
L401 IF TYPE(I)≠"RANG" THEN WRITE(PRINT,FL4) ELSE WRITE(PRINT,FL5);	0325
L411 WRITE(PRINT,FL6,CV(I,M));	0338
IF DELTP(I,M,1)=0.0 THEN BEGIN DX1=1; GO TO L421; END;	0346
FOR DX1=1,2 DO IF DELTP(1,DELTP(I,M,DX1)) THEN GO TO L421;	0350
L421 COST=ACTCOST(I,M,DX1); TOTAL=TOTAL+COST;	0359
WRITE(PRINT,FL7,ACTNO(I,M,DX1),COST);	0362
PRINTZ;	0374
IF CHEAP(2)=0 OR J=CHEAP(2) THEN GO TO LX;	0378
J=CHEAP(2);	0379
WRITE(PRINT,LOBL);	0380
IF RR=0 THEN BEGIN	0383
COST=85.0; TOTAL=COST;	0385
WRITE(PRINT,FL4011,COST);	0386
COST=310.0; TOTAL=TOTAL+COST;	0395
WRITE(PRINT,FL4010,COST);	0397
GO TO L341;	0406

END)	0406
COST=IF COMPONENT="A" THEN 118.0 ELSE 147.0)	0406
TOTAL=COST)	0409
WRITE(PRINT,FL4008,COMPONENT,COST))	0410
GO TO L34)	0422
L33: IF STATIC=0 THEN GO TO L11)	0422
WRITE(PRINT,FL1,PB))	0424
WRITE(PRINT,FL0,RR))	0432
WRITE(PRINT,FL4003))	0440
COST=65.0) TOTAL=COST)	0443
WRITE(PRINT,FL4009,COST))	0445
COST=310.0) TOTAL=TOTAL+COST)	0454
WRITE(PRINT,FL4010,COST))	0456
J=CHEAP(1))	0465
GO TO L34)	0466
L4: IF M1\$14.0/8 THEN GO TO L80)	0466
L52: WRITE(PRINT,FL4002))	0468
GO TO L4)	0472
L80: FOR J=1 STEP 1 UNTIL 10 DO	0474
IF ABS(B(I))=3 OR ABS(C(I))=4 THEN GO TO L81)	0477
GO TO L52)	0482
L81: WRITE(PRINT,FL1,PB))	0483
WRITE(PRINT,FL4003))	0492
COST=345.0) TOTAL=COST)	0495
WRITE(PRINT,FL4012,COST))	0497
COST=125.0) TOTAL=TOTAL+COST)	0506
WRITE(PRINT,FL4013,COST))	0508
COST=95.0) TOTAL=TOTAL+COST)	0517
WRITE(PRINT,FL4014,COST))	0519
J=CHEAP(1))	0528
L60: FOR I=1 STEP 1 UNTIL COMPAT OR IF AVAIL(I)=J THEN M=COLE(I))	0529
WRITE(PRINT,FL7,MODL(I),SI/EC(I))	0535

IF TYPE(J) = "DS" THEN TYPE(J) = "DOUBLE" ELSE TYPE(J) = "SINGLE"	0546
WRITE(PRINT, FL3, TYPE(J))	0553
WRITE(PRINT, FL6, CV(J, M))	0561
IF DELTPE(J, M, L) = 0.0 THEN BEGIN DX1 = 1; GO TO L57; END	0572
FOR DX1 = 1 TO 2 DO IF DELTPE(DELTPE(J, M, DX1)) THEN GO TO L57	0576
L57: COST = ACTCOST(J, M, DX1); TOTAL = TOTAL + COST	0585
WRITE(PRINT, FL21, VPLINK(J), ACTNO(J, M, DX1), COST)	0588
PRINT	0602
IF J = CHEAP(2) OR CHEAP(2) = 0 THEN GO TO LX	0602
J = CHEAP(2)	0607
WRITE(PRINT, DBL)	0608
COST = 345.0; TOTAL = TOTAL + COST	0611
WRITE(PRINT, FL4015, COST)	0613
COST = 125.0; TOTAL = TOTAL + COST	0622
WRITE(PRINT, FL4013, COST)	0624
COST = 95.0; TOTAL = TOTAL + COST	0633
WRITE(PRINT, FL3014, COST)	0635
GO TO L60	0644
L12: WRITE(PRINT, FL30, DM)	0648
WRITE(PRINT, FL4003)	0653
COST = 30.05; TOTAL = TOTAL + COST	0656
WRITE(PRINT, FL4016, COST)	0658
COST = 10.0; TOTAL = TOTAL + COST	0667
WRITE(PRINT, FL4017, COST)	0669
J = CHEAP(1)	0678
L70: FOR I = 1 STEP 1 UNTIL COMPAT DO IF AVAIL(I) = J THEN M = COL(I)	0679
WRITE(PRINT, FL2, MODEL(J), SIZE(I))	0685
IF TYPE(J) = "DS" AND TYPE(J) = "SS" THEN GO TO L71	0696
IF TYPE(J) = "DS" THEN TYPE(J) = "DOUBLE" ELSE TYPE(J) = "SINGLE"	0699
WRITE(PRINT, FL3, TYPE(J))	0707
GO TO L72	0717
L71: IF TYPE(J) = "DS" THEN WRITE(PRINT, FL4) ELSE WRITE(PRINT, FL5)	0717

L72: WRITE(PRINT,FL6,CV(J,M))	0728
IF DELTP(J,M,1)≠0.0 THEN BEGIN DX1=1; GO TO L73; END	0734
FOR DX1=1,2 DO IF DELTP(1,DELTP(J,M,DX1)) THEN GO TO L73	0742
L73: COST+ACTCOST(J,M,DX1); TOTAL+TOTAL+COST;	0751
IF (MODEL(J) MOD 100)≠6 THEN WRITE(PRINT,FL21,QLINK(J),ACTN(J,M,DX1),	0754
COST) ELSE WRITE(PRINT,FL7,ACTN(J,M,DX1),COST);	0766
PRINTZ;	0762
IF CHEAP(2)≠0 OR CHEAP(2)=J THEN GO TO LX;	0782
J+CHEAP(2);	0767
WRITE(PRINT,DBL);	0788
COST+30.05; TOTAL+COST;	0791
WRITE(PRINT,FL4010,COST);	0793
COST+30.07; TOTAL+TOTAL+COST;	0802
WRITE(PRINT,FL4017,COST);	0804
GO TO L70;	0813
END PRINTING BLK;	0813
	0015 IS 0815 LONG, NEXT SEG 0012
LX: WRITE(PRINT,PAGE3);	0688
WRITE(PRINT,FL2000,(TIME(I)-Z)/60);	0689
END END.	0700
	0012 IS 0704 LONG, NEXT SEG 0002
	0002 IS 0096 LONG, NEXT SEG 0001
LXP IS SEGMENT NUMBER 0020,PRT ADDRESS IS 0331	
LN IS SEGMENT NUMBER 0021,PRT ADDRESS IS 0330	
SORT IS SEGMENT NUMBER 0022,PRT ADDRESS IS 0355	
OUTPUT(M) IS SEGMENT NUMBER 0023,PRT ADDRESS IS 0311	
OUTPUT(C) IS SEGMENT NUMBER 0024,PRT ADDRESS IS 0307	
INPUT(W) IS SEGMENT NUMBER 0025,PRT ADDRESS IS 0304	
INPUT(C) IS SEGMENT NUMBER 0026,PRT ADDRESS IS 0301	
X TO THE 1 IS SEGMENT NUMBER 0027,PRT ADDRESS IS 0332	
GO TO SOLVER IS SEGMENT NUMBER 0028,PRT ADDRESS IS 0365	
FILE CNTL(W) IS SEGMENT NUMBER 0029,PRT ADDRESS IS 0014	

FILE CNTMLCC) IS SEGMENT NUMBER 0030.PRT ADDRESS IS 0015

HEAD/WHILE IS SEGMENT NUMBER 0031.PRT ADDRESS IS 0016

NUMBER OF ERRORS DETECTED = 000. COMPILATION TIME = 0131 SECONDS.

PRT SIZE=0319)TOTAL SEGMENT SIZE=04602 WORDS)DRUM STORAGE REQ.=05109 WORDS)NO. SEGS.=0031.

ESTIMATED CORE STORAGE REQUIREMENT = 10574 WORDS.

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